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RESEARCH ON STRUCTURAL DYNAMIC TESTING  
BY IMPEDANCE METHODS. VOLUME I.  
STRUCTURAL SYSTEM IDENTIFICATION FROM  
MULTIPOINT EXCITATION

William G. Flannelly, et al

Kaman Aerospace Corporation

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**USAAMRDL TECHNICAL REPORT 72-63A  
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TESTING BY IMPEDANCE METHODS  
VOLUME I  
STRUCTURAL SYSTEM IDENTIFICATION FROM  
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By

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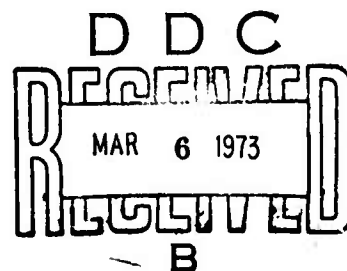
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**U. S. ARMY AIR MOBILITY RESEARCH AND DEVELOPMENT LABORATORY  
FORT EUSTIS, VIRGINIA**

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KAMAN AEROSPACE CORPORATION  
BLOOMFIELD, CONNECTICUT**

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This program was conducted under Contract DAAJ02-70-C-0012 with Kaman Aerospace Corporation.

This report contains the theoretical derivation and the presentation of a methodology for system identification of structures. Computer experiments were run to verify this methodology.

The report has been reviewed by this Directorate and is considered to be technically sound. It is published for the exchange of information and the stimulation of future research.

This program was conducted under the technical management of Mr. Arthur J. Gustafson, Technology Applications Division.

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RESEARCH ON STRUCTURAL DYNAMIC  
TESTING BY IMPEDANCE METHODS

Volume I  
Structural System Identification From  
Multipoint Excitation

Final Report

Kaman Report R-1001-1

By

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for

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## FOREWORD

The work presented in this report was performed by Kaman Aerospace Corporation under Contract DAAJ02-70-C-0012 (Task 1F162204AA4301) for the Eustis Directorate, U. S. Army Air Mobility Research and Development Laboratory, Fort Eustis, Virginia. The program was implemented under the technical direction of Mr. Joseph H. McGarvey of the Reliability and Maintainability Division\* and Mr. Arthur J. Gustafson of the Structures Division.\*\* The report is presented in four volumes, each describing a separate phase of the basic theory of structural dynamic testing using impedance techniques.

Volume I presents the results of an analytical and numerical investigation of the practicality of system identification using fewer measurement points than there are degrees of freedom. The parameters in Lagrange's equations of motion, mass, stiffness, and damping for a mathematical model having fewer degrees of freedom than the linear elastic structure it represents may be determined directly from measured mobility data. Volume II describes the method of system identification wherein the necessary impedance data are experimentally determined by applying a force excitation at a single point on the structure. Volume III presents a method of determining the free-body dynamic responses from data obtained on a constrained structure. Volume IV describes a method of obtaining the equations for the combination of measured mobility matrices of a helicopter and its subsystems. The response of the combination of a helicopter and its subsystems is determined from data based on the experimental results of the main system and subsystems separately.

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\*Division name changed to Military Operations Technology Division.

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## LIST OF SYMBOLS

- [c]      the damping matrix
- [d]      a damping matrix;  $[d] = \omega[c]$ ; for damping forces which are proportional to displacement
- {f}      vector of external forces acting along the generalized coordinates
- $\tilde{\{f\}}$       force phasor,  $\{f\} = \tilde{\{f\}}e^{i\omega t}$
- $g_i$       the structural damping coefficient of the i-th mode
- i or j    indices; imaginary operator ( $i = \sqrt{-1}$ )
- $\chi_i$       the generalized stiffness of the i-th mode
- [k]      the stiffness matrix
- $m_i$       the generalized mass of the i-th mode
- [m]      the mass matrix
- N or n    the number of degrees of freedom in the structure
- $\{\dot{y}\}$       vector of velocities of the generalized coordinates
- $\tilde{\{\dot{y}\}}$       velocity phasor,  $\{\dot{y}\} = \tilde{\{\dot{y}\}}e^{i\omega t}$
- $[Y_{(\omega)}]$     matrix of mobilities at forcing frequency  $\omega$ ;  
 $[Y_{(\omega)}] = [\partial \tilde{y}_i / \partial f_j]_{(\omega)}$

# LIST OF SYMBOLS (Continued)

$Y_i^*(\omega)$	generalized mobility of the i-th mode at forcing frequency $\omega$
$[Y]$	matrix of acceleration mobilities
$[z(\omega)]$	matrix of impedances at forcing frequency $\omega$ ; $[z(\omega)] = [\partial \tilde{f}_i / \partial \dot{y}_j](\omega)$
$\dot{z}_i^*(\omega)$	generalized impedance of the i-th mode at forcing frequency $\omega$
$\bar{z}_i^*(\omega)$	complex conjugate of the i-th mode generalized impedance at forcing frequency $\omega$
$ \dot{z}_i^*(\omega) $	absolute value of the i-th mode generalized impedance at forcing frequency $\omega$
$\{\gamma\}_i$	the i-th column of $[\Gamma]$ ; the gamma vector of the i-th mode; a left-hand eigenvector of $[k]^{-1}[m]$
$[\Gamma]$	the left-hand eigenvectors of $[k]^{-1}[m]$ ; $[\Phi]^{-T}$
$\delta_i^j$	Kronecker's delta
$\{\phi\}_i$	the modal vector of the i-th mode
$[\Phi]$	matrix of modal vectors
$\omega$	forcing frequency
$\Omega_i$	the natural frequency of the i-th mode

## LIST OF SYMBOLS (Continued)

### SUPERSCRIPTS

- R        the real part of a complex quantity
- I        the imaginary part of a complex quantity
- \*        a generalized parameter associated with a particular mode
- T        the transpose
- T       the inverse transpose

### SUBSCRIPTS

- ( $\omega$ )     the forcing frequency at which the quantity was measured or calculated
- k        forcing frequency

A dot over a quantity indicates differentiation with respect to time

### BRACKETS

- [ ], ( )   matrix
- [ J ]     diagonal matrix
- { }       column or row vector

## INTRODUCTION

The success of a helicopter structural design is highly dependent on the ability to predict and control the dynamic response of the fuselage and mechanical components. Conventionally, this involves the formulation of intuitively based equations of motion. Ideally, this process would reduce the physical structure to an analytical mathematical model which would predict accurately the dynamic response characteristics of the actual structure. Obviously, the creation of such an intuitive abstraction of a complicated real structure requires considerable expertise and inherently includes a high degree of uncertainty. Structural dynamic testing is required to substantiate the analytical results. The analysis is modified until successful correlation is obtained between the analytical predictions and the test results. Finally, the mathematical model can be used to incorporate changes to improve the structural integrity of the helicopter.

This report describes the theory of structural dynamic testing using impedance techniques as applied to a mathematical model having fewer degrees of freedom than the structure. Reference 1 describes the method of obtaining a model directly from test measurements for a hypothetical structure which has the same number of degrees of freedom as the mathematical model. In reality, the number of degrees of freedom of a physical structure is infinite; therefore, the usefulness of model identification, necessarily with a finite number of degrees of freedom, using impedance testing techniques depends on the ability to simulate the real structure with a small mathematical model.

The process of deriving the equations of motion from test data is referred to as system identification. The only input information required in this theory is measured mobility data and the approximate natural frequency of each mode. This information can be obtained from impedance testing of the actual structure over the frequency range of interest yielding the second order, structurally damped linear equations of motion.

System identification theories to be of any practical engineering significance must be functional with a reasonable degree of experimental error. In this report, a series of computer experiments incorporating experimental errors was documented. This report presents a modification and extension of the analysis derived in Reference 1 such that an identified model with a finite number of degrees of freedom simulates the actual structure wherein the number of degrees of freedom is infinite.



## THEORY

### DERIVATION

The equations of motion in matrix form of a linear system are, as shown in Reference 1,

$$[m]\{\ddot{y}\} + [c]\{\dot{y}\} + [k]\{y\} = \{f\} \quad (1)$$

Assume a steady-state solution of the form

$$\{\dot{y}\} = \{\tilde{y}\}e^{i\omega t} \text{ and } \{f\} = \{\tilde{f}\}e^{i\omega t}$$

Substitute these equations into Equation (1) to give

$$\left[ ([m]\omega - \frac{1}{\omega}[k])i + [c] \right] \{\tilde{y}\} = \{\tilde{f}\} \quad (2)$$

or

$$(i[\dot{z}_{\omega}^I] + [\dot{z}_{\omega}^R])\{\tilde{y}\} \equiv [\dot{z}_{\omega}]\{\tilde{y}\} = \{\tilde{f}\}$$

where  $\dot{z}_{ij}(\omega)$  is defined herein as the element velocity impedance measured at  $\omega$ .

The element impedance can also be expressed as

$$\dot{z}_{ij}(\omega) = \partial \tilde{f}_i / \partial \tilde{y}_j$$

If Equation (2) is premultiplied by  $[\phi]^{-T}[\phi]^T$  and post-multiplied by  $[\phi][\phi]^{-1}$  where  $[\phi]$  is the matrix of modal vectors, the result is

$$[\phi]^{-T} \left[ i([\phi]^T[m][\phi]\omega - \frac{1}{\omega}[\phi]^T[k][\phi]) + [\phi]^T[c][\phi] \right] [\phi]^{-1} = [\dot{z}_{(\omega)}] \quad (3)$$

The diagonal generalized mass is expressed by

$$[M] = [\phi]^T[m][\phi] \quad (4)$$

The diagonal generalized stiffness is given by

$$[K] = [\phi]^T[k][\phi] \quad (5)$$

Assume that

$$\frac{1}{\omega} [gk] = [\phi]^T [c] [\phi] \quad (6)$$

such as would be expected from structural damping in a lightly damped structure. Substituting Equations (4), (5) and (6) into Equation (3) yields

$$[\dot{z}(\omega)] = [\phi]^{-T} \left[ j(\eta\omega - \frac{k}{\omega}) + \frac{gk}{\omega} \right] [\phi]^{-1} \quad (7)$$

Define the  $i$ -th modal impedance as

$$\dot{z}_i^*(\omega) = j(\eta_i\omega - \frac{k_i}{\omega}) + \frac{g_i k_i}{\omega}$$

and substitute into Equation (7) to give

$$[\dot{z}(\omega)] = [\phi]^{-T} [\dot{z}_i^*(\omega)] [\phi]^{-1} \quad (8)$$

The elemental mobility at forcing frequency  $\omega$  is defined as

$\dot{y}_{ij}(\omega) \equiv \partial \tilde{y}_i / \partial f_j$  and is equal to the ratio of the velocity

phaser along the coordinate  $i$  to the external force phaser along the coordinate  $j$  when no other forces are externally applied. The full mobility matrix is given by

$$[\dot{Y}(\omega)] = [\partial \tilde{y} / \partial f]_{(\omega)} = [\partial \tilde{f} / \partial \tilde{y}]_{(\omega)}^{-1} \equiv [z(\omega)]^{-1} \quad (9)$$

Therefore, using Equation (8) it is seen that

$$[\dot{Y}(\omega)] = [\phi] \left[ \frac{1}{\dot{z}_i^*(\omega)} \right] [\phi]^T \equiv [\phi] [\dot{Y}_i^*(\omega)] [\phi]^T \quad (10)$$

The modal mobility of the  $i$ -th mode measured at  $\omega$  is

$$\begin{aligned} \dot{Y}_i^*(\omega) &= \dot{Y}_i^{*R}(\omega) + i\dot{Y}_i^{*I}(\omega) = \frac{1}{\dot{z}_i^*(\omega)} = \frac{\overline{\dot{z}_i^*(\omega)}}{(\dot{z}_i^*(\omega))^2} \\ &= \frac{\dot{z}_i^{*R}(\omega) - i\dot{z}_i^{*I}(\omega)}{(\dot{z}_i^{*R}(\omega))^2 + (\dot{z}_i^{*I}(\omega))^2} = \frac{\frac{g_i k_i}{\omega} - i(\eta_i\omega - \frac{k_i}{\omega})}{(\frac{g_i k_i}{\omega})^2 + (\eta_i\omega - \frac{k_i}{\omega})^2} \end{aligned}$$

Dividing numerator and denominator of the previous equation by the generalized mass  $m_i$

$$\dot{Y}_{i(\omega)}^* = \frac{\frac{g_i \chi_i}{m_i \omega} - i(\omega - \frac{\chi_i}{m_i \omega})}{m_i \left( \frac{g_i \chi_i}{\omega m_i} \right)^2 + m_i \left( \omega - \frac{\chi_i}{m_i \omega} \right)^2}$$

Substituting the natural frequency of the i-th mode

$$\Omega_i = \sqrt{\frac{\chi_i}{m_i}}$$

$$\dot{Y}_{i(\omega)}^* = \frac{\frac{g_i \Omega_i}{\omega} - i(\omega - \frac{\Omega_i}{\omega})}{m_i \left( \frac{g_i \Omega_i}{\omega} \right)^2 + m_i \left( \omega - \frac{\Omega_i}{\omega} \right)^2}$$

Separating this equation into the real and imaginary components yields

$$\dot{Y}_{i(\omega)}^* = \frac{1}{\omega m_i} \left( \frac{\omega}{\Omega_i} \right)^2 \left[ \frac{\frac{g_i}{2} - i \frac{\left( \frac{\omega^2}{\Omega_i^2} - 1 \right)}{2}}{g_i + \left( \frac{\omega^2}{\Omega_i^2} - 1 \right)} \right] \quad (11)$$

Finally, from Equation (10), the real mobility may be written as

$$[\dot{Y}_{(\omega)}^R] = [\phi] \left[ \dot{Y}_{(\omega)}^{*R} \right] [\phi]^T \quad (12)$$

Reference 1 indicated that because the real modal mobilities of modes far removed from the forcing frequency become negligible compared to adjacent modes, the real mobility matrix at any frequency is ordinarily affected only by modes in close proximity to the forcing frequency. The measured real mobility matrix at a particular frequency reflects the influence of only the most dominant modes in that frequency of measurement region. Therefore, it is unrealistic to use the real mobility matrix measured at any specific frequency to determine parameters other than those associated with neighboring modes.

Reference 1 also shows that the imaginary modal mobilities of modes associated with frequencies less than the forcing frequency asymptotically approach a constant. An imaginary mobility matrix contains the effect of all lower modes in proportion to, or greater than, the magnitudes of their generalized masses. Therefore, it is impractical to use imaginary mobility matrices to evaluate properties associated with natural frequencies far above the forcing frequency.

These characteristics of the modal mobility make it impossible to determine the system parameters from the  $n$  equations in  $n$  unknowns obtained from mobility matrices measured at any two forcing frequencies.

Even if the modal mobility were amenable to determination of the system parameters, the precision of measurement which would be required to do this for most systems is impossible to achieve. The modal approach derived below avoids this problem.

#### DERIVATION OF THE DOMINANT MODE EIGENVALUE PROBLEM

Equation (10) may be written

$$[\dot{Y}_{(\omega)}] = [\Phi] \left[ \dot{Y}_{(\omega)}^* \right] [\Phi]^T = \sum_{i=1}^N \dot{Y}_i^* (\omega) \{\phi\}_i \{\phi_i\}^T \quad (13)$$

where  $\{\phi\}$  is a column in  $[\Phi]$  and  $N$  is the order of the matrices. Define  $[\Gamma] = [\Phi]^{-T}$ , and Equation (8) may be written as

$$[\dot{Y}_{(\omega)}]^{-1} = [Z_{(\omega)}] = [\Gamma] \left[ \dot{Z}_{(\omega)}^* \right] [\Gamma]^T = \sum_{i=1}^N \frac{1}{Y_i^* (\omega)} \{\gamma\}_i \{\gamma_i\}^T \quad (14)$$

where  $\{\gamma\}$  is a column in  $[\Gamma]$ .

Each matrix  $\left[ \dot{Y}_i^* (\omega) \{\phi_i\} \{\phi_i\}^T \right]$  and  $\left[ \frac{1}{\dot{Y}_i^* (\omega)} \{\gamma\} \{\gamma_i\}^T \right]$  in Equations (13)

and (14) is of rank one, but the summation of as many of these successive modal matrices as the order  $N$  of the matrix is a nonsingular matrix.

Similarly,

$$\begin{aligned}
 [\dot{Y}_{(\omega)}^R] &= \sum_{i=1}^N \dot{Y}_{i(\omega)}^{*R} \{\phi\}_i \{\phi\}_i^T \\
 [\dot{Y}_{(\omega)}^I] &= \sum_{i=1}^N \dot{Y}_{i(\omega)}^{*I} \{\phi\}_i \{\phi\}_i^T \\
 [\dot{Y}_{(\omega)}^R]^{-1} &= \sum_{i=1}^N \frac{1}{\dot{Y}_{i(\omega)}^{*R}} \{\gamma\}_i \{\gamma\}_i^T \\
 [\dot{Y}_{(\omega)}^I]^{-1} &= \sum_{i=1}^N \frac{1}{\dot{Y}_{i(\omega)}^{*I}} \{\gamma\}_i \{\gamma\}_i^T
 \end{aligned} \tag{15}$$

The iteration procedure used to solve the eigenvalue problem in Reference 1 employed the imaginary part of a mobility matrix measured at a frequency just above the N-th natural frequency. The method used to solve the eigenvalue problem in the present report, which was found to give more accurate results, utilizes the sum of the real parts of the mobility matrices measured near each of the natural frequencies associated with the actual model. It has been indicated previously that a measured real mobility matrix reflects the influence of only the most dominant modes in the vicinity of the forcing frequency. Therefore, summation of a discrete set of the real mobility matrices measured at forcing frequencies near the corresponding natural frequencies should contain precisely the information relevant to the model normal modes.

The eigenvalue problem may be formulated as follows. Consider the summation of the real mobility matrices measured at a discrete set of frequencies near the first NR natural frequencies. Take the inverse of this matrix and pre-multiply by a real mobility matrix measured at any frequency  $\omega_k$ .

$$\begin{aligned}
& [\dot{Y}^R(\omega_k)] \left[ \sum_{\omega_j = \Omega_1}^{\Omega_{NR}} \dot{Y}_i^R(\omega_j) \right]^{-1} \\
&= \sum_{i=1}^{NR} \dot{Y}_i^{*R}(\omega_k) \{\phi_i\} \{\phi_i\}^T \left( \sum_{i=1}^{NR} \sum_{j=1}^{NR} \dot{Y}_i^{*R}(\omega_j) \{\phi_i\} \{\phi_i\}^T \right)^{-1} \\
&= [\Phi] \left[ \dot{Y}_i^{*R}(\omega_k) \right] [\Phi]^T \left( [\Phi] \left[ \sum_{j=1}^{NR} \dot{Y}_i^{*R}(\omega_j) \right] [\Phi]^T \right)^{-1} \\
&= [\Phi] \left[ \dot{Y}_i^{*R}(\omega_k) \right] [\Phi]^T [\Phi]^{-T} \left[ \sum_{j=1}^{NR} \frac{1}{\dot{Y}_i^{*R}(\omega_j)} \right] [\Phi]^{-1} \\
&= [\Phi] \left[ \frac{\dot{Y}_i^{*R}(\omega_k)}{\sum_{j=1}^{NR} \dot{Y}_i^{*R}(\omega_j)} \right] [\Phi]^{-1} \tag{16}
\end{aligned}$$

If Equation (16) is postmultiplied by  $\{\phi\}_i$ , there results

$$\left[ \dot{Y}^R(\omega_k) \right] \left[ \sum_{\omega_j = \Omega_1}^{\Omega_{NR}} \dot{Y}_i^R(\omega_j) \right]^{-1} \{\phi\}_i = [\Phi] \left[ \frac{\dot{Y}_i^{*R}(\omega_k)}{\sum_{j=1}^{NR} \dot{Y}_i^{*R}(\omega_j)} \right] [\Phi]^{-1} \{\phi\}_i$$

but  $[\Phi]^{-1} \{\phi\}_i$  yields a column matrix comprised of zeroes except for a 1 in the  $i$ -th position. Finally,

$$[\Phi] \left[ \begin{array}{c} \dot{Y}_i^{*R}(\omega_k) \\ \text{NR} \\ \sum_{j=1} \dot{Y}_i^{*R}(\omega_j) \end{array} \right] [\Phi]^{-1} \{\phi\}_i = \frac{\dot{Y}_i^{*R}(\omega_k)}{\sum_{j=1} \dot{Y}_i^{*R}(\omega_j)} \{\phi\}_i$$

The eigenvalue problem is finally formulated as

$$[\dot{Y}_{(\omega_k)}^R] \left[ \sum_{j=\Omega_1}^{\Omega_{NR}} \dot{Y}_{(\omega_j)}^R \right]^{-1} \{\phi\}_i = \frac{\dot{Y}_i^{*R}(\omega_k)}{\sum_{j=\Omega_1}^{\Omega_{NR}} \dot{Y}_i^{*R}(\omega_j)} \{\phi\}_i \quad (17)$$

If the order of multiplication is reversed in Equation (16), an eigenvalue problem is developed in which the eigenvector is the gamma vector of the  $i$ -th mode. Consider the same parameters as in Equation (16); only the order of multiplication of the matrices is changed.

$$\begin{aligned} & \left[ \sum_{j=\Omega_1}^{\Omega_{NR}} \dot{Y}_i^R(\omega_j) \right]^{-1} [\dot{Y}_{(\omega_k)}^R] \\ &= \left( \sum_{i=1}^{\text{NR}} \sum_{j=1}^{\text{NR}} \dot{Y}_i^{*R}(\omega_j) \{\phi_i\} \{\phi_i\}^T \right)^{-1} \left( \sum_{i=1}^{\text{NR}} \dot{Y}_i^{*R}(\omega_k) \{\phi_i\} \{\phi_i\}^T \right) \\ &= ([\Phi] \left[ \sum_{j=1}^{\text{NR}} \dot{Y}_i^{*R}(\omega_j) \right] [\Phi]^T)^{-1} [\Phi] \left[ \dot{Y}_i^{*R}(\omega_k) \right] [\Phi]^T \\ &= [\Phi]^{-T} \left[ \begin{array}{c} 1 \\ \text{NR} \\ \sum_{j=1} \dot{Y}_i^{*R}(\omega_j) \end{array} \right] [\Phi]^{-1} [\Phi] \left[ \dot{Y}_i^{*R}(\omega_k) \right] [\Phi]^T \\ &= [\Phi]^{-T} \left[ \begin{array}{c} \dot{Y}_i^{*R}(\omega_k) \\ \text{NR} \\ \sum_{j=1} \dot{Y}_i^{*R}(\omega_j) \end{array} \right] [\Phi]^T \end{aligned} \quad (18)$$

By definition,  $[\Gamma] = [\phi]^{-T}$  and  $[\Gamma]^{-1} = [\phi]^T$ ; substituting into Equation (18) yields

$$\begin{bmatrix} \Omega_{NR} \\ \Sigma \\ \omega_j = \Omega_1 \end{bmatrix} \begin{bmatrix} \dot{Y}_i^R(\omega_j) \end{bmatrix} [\dot{Y}_i^R(\omega_k)] = [\Gamma] \begin{bmatrix} \dot{Y}_i^{*R}(\omega_k) \\ NR \\ \Sigma \\ j=1 \end{bmatrix} [\Gamma]^{-1} \quad (19)$$

If Equation (19) is postmultiplied by  $\{\gamma\}_i$ , a column of  $[\Gamma]$ , and the same procedure is followed as was used in obtaining Equation (17), Equation (19) becomes

$$\begin{bmatrix} \Omega_{NR} \\ \Sigma \\ \omega_j = \Omega_1 \end{bmatrix} \begin{bmatrix} \dot{Y}_i^R(\omega_j) \end{bmatrix}^{-1} [\dot{Y}_i^R(\omega_k)] \{\gamma\}_i = \frac{\dot{Y}_i^{*R}(\omega_k)}{\begin{bmatrix} NR \\ \Sigma \\ \omega_j = \Omega_1 \end{bmatrix} \dot{Y}_i^{*R}(\omega_j)} \{\gamma\}_i \quad (20)$$

which is an eigenvalue problem with the eigenvector equal to the gamma vector of the  $i$ -th mode.

#### IDENTIFICATION OF STRUCTURAL PARAMETERS

A successful identification procedure, using normal mode techniques, should separate the effect of each mode in a mathematical sense, regardless of the number of stations where mobility measurements are taken on the structure. If satisfactory normal mode separation required a certain minimum number of measurement stations greater than the number of degrees of freedom chosen for the model, the most that can be expected is an approximate model, possibly including optimization procedures designed to satisfy all system constraints. This situation is considered in detail in Reference 2 in which a mathematical model is derived from test data such that identification of the structure is obtained closest to any specified analytical approximation.

Satisfactory normal mode separation requires that the values of  $\dot{Z}^{*R}(\omega_j)$  and  $\dot{Z}^{*I}(\omega_j)$  be independent of the number of degrees of

freedom of the model. The values of the generalized mass ( $m_i$ ), the corresponding identified natural frequency ( $\Omega_i$ ), and the generalized stiffness as defined below are then also independent of the number of measurement stations.



$$\eta_i = \frac{\omega_k \dot{z}_{i(\omega_k)}^{*I} - \omega_j \dot{z}_{i(\omega_j)}^{*I}}{(\omega_k^2 - \omega_j^2)} \quad (21)$$

and

$$\Omega_i^2 = \omega_j \omega_k \frac{\omega_j \dot{z}_{i(\omega_k)}^{*I} - \omega_k \dot{z}_{i(\omega_j)}^{*I}}{\omega_k \dot{z}_{i(\omega_k)}^{*I} - \omega_j \dot{z}_{i(\omega_j)}^{*I}} \quad (22)$$

$$\kappa_i = \Omega_i^2 \eta_i \quad (23)$$

The two forcing frequencies ( $\omega_k$ ) and ( $\omega_j$ ) are chosen in the vicinity of the corresponding natural frequency which is available from test data. The generalized impedance of the i-th mode at forcing frequency ( $\omega$ ) is obtained from the generalized mobility of the i-th mode at forcing frequency ( $\omega$ ). It follows from Equation (13) that the modal mobilities are given by

$$\begin{aligned} \left[ \dot{y}_{(\omega)}^* \right] &= [\Phi]^{-1} [\dot{Y}_{(\omega)}] [\Phi]^{-T} \\ &= [\Gamma]^T [\dot{Y}_{(\omega)}] [\Gamma] \end{aligned} \quad (24)$$

and, therefore, the orthogonality condition for gamma vectors is

$$\{\gamma\}_i^T [\dot{Y}_{(\omega)}] \{\gamma\}_i = \dot{y}_{i(\omega)}^* \delta_i^j$$

The modal impedance of the i-th mode at  $\omega_j$  is

$$\dot{z}_{i(\omega_j)}^* = \frac{\dot{y}_{i(\omega_j)}^*}{|\dot{y}_{i(\omega_j)}^*|^2} = \frac{\dot{y}_{i(\omega_j)}^{*R} - i \dot{y}_{i(\omega_j)}^*}{|\dot{y}_{i(\omega_j)}^*|^2}$$

It follows that

$$z_i^{*I}(\omega_j) = \frac{-Y_i^{*I}(\omega_j)}{|Y_i^{*}(\omega_j)|^2}$$

and

$$z_i^{*R}(\omega_j) = \frac{Y_i^{*R}(\omega_j)}{|Y_i^{*}(\omega_j)|^2}$$

The damping coefficient for the  $i$ -th mode is most readily given by

$$g_i = \frac{\omega_j z_i^{*R}(\omega_j)}{\kappa_i} \quad (25)$$

which follows directly from Equation (7). The damping coefficient for the  $i$ -th mode may also be obtained by

$$g_i = \left( \frac{\omega_j^2}{\Omega_i^2} - 1 \right) \frac{z_i^{*R}(\omega_i)}{z_i^{*I}(\omega_j)} \quad (26)$$

Using a measurement of real mobility taken precisely at resonance, the damping coefficient may be calculated using Equation (11) as

$$g_i = \frac{1}{Y_i^{*R}(\Omega_i) \Omega_i \eta_i}$$

#### PARAMETERS OF THE MATHEMATICAL MODEL

The elements of the influence coefficient matrix, being a measure of displacement per unit force, are independent of the number of measurement stations defining the order of the matrix. Conversely, the elements of the stiffness and mass matrices assume different values as the number of degrees of freedom of the model is changed. The identification procedure used in Reference 1 calculates both stiffness and mass matrices by summing the effects of each consecutive mode and defining the incomplete matrices as the sum up to and including a particular mode. If the order of the model

degrees of freedom is changed from ND for the structure to NR for the model, the corresponding incomplete mass and stiffness matrices will not be directly comparable, on a modal basis, to the structure mass and stiffness matrices. It is more expedient to identify the influence coefficient matrix [c] and the inverse of the mass matrix [M]. Premultiplying Equation (4) by  $[\phi]^{-T}$  and postmultiplying  $[\phi]^{-1}$  and taking the inverse of the resulting equation yields

$$[M] = [m]^{-1} = \sum_{i=1}^{NR} \frac{1}{m_i} \{\phi_i\} \{\phi_i\}^T \quad (28)$$

If the same operations are performed on Equation (5), the result is

$$[c] = [K]^{-1} = \sum_{i=1}^{NR} \frac{1}{\Omega_i^2 m_i} \{\phi_i\} \{\phi_i\}^T \quad (29)$$

Set  $[c] = \frac{1}{\omega} [d]$  and using Equation (6) there results

$$\frac{1}{\omega} [gk] = \frac{1}{\omega} [\phi]^T [d] [\phi]$$

Solving for the damping matrix yields

$$[d] = \phi^{-T} [gk] \phi^{-1}$$

Substituting  $[\Gamma] = [\phi]^{-T}$ ,  $[\Gamma]^T = [\phi]^{-1}$  and  $[k] = [\Omega^2 m]$  into the previous equation gives

$$[d] = [\Gamma] [g \Omega^2 m] [\Gamma]^T$$

The damping matrix can be expressed as

$$[d] = \sum_{i=1}^{NR} g_i \Omega_i^2 m_i \{\gamma_i\} \{\gamma_i\}^T \quad (30)$$

### ITERATION PROCEDURE

The calculation of the modal parameters such as generalized mass, stiffness and the corresponding natural frequency requires the generalized impedance at a particular frequency for each mode under consideration. The modal impedance is a function of the generalized mobility for the same mode and forcing frequency. As indicated in Equation (24), the modal mobilities are dependent upon the matrix of gamma vectors and its transpose. The iteration process as originally formulated

in the present work involved iteration on the normal mode vectors with a subsequent inversion operation to determine the gamma vectors. This sequence introduced errors into the system, with the result that the gamma vectors did not resemble the associated gamma vectors obtained from the specimen representing the actual structure. The iterated normal mode vectors obtained from the mathematical model were extremely close, particularly at the lower modes, to the specimen, or exact, normal mode vectors. Nevertheless, any discrepancy between the model iterated modal vectors and the exact values, however small, was magnified in the inversion process, causing the gamma vectors to bear little resemblance to the specimen gamma vectors. Therefore, it was deemed advisable to iterate on the gamma vectors directly and dispense with the intermediate inversion operation.

To equalize the effect of each modal mobility in the matrix iteration Equation (20), several normalization procedures were incorporated into the method. First, each real mobility matrix was normalized on the largest element of the respective matrix. This procedure proved satisfactory except in some situations where the elements of the mobility matrices were approximately of the same magnitude but the largest elements differed in algebraic sign. This resulted in a cancellation effect among the real mobility matrices and an incorrect summation, thereby causing erroneous calculated gamma vectors. A modification to the normalization technique was applied whereby the real mobility matrices at each forcing frequency were divided by the absolute value of the largest element in the respective matrices. As a further refinement on the normalization procedure, the real mobility matrix at each forcing frequency was normalized on the root mean square associated with each respective matrix. Occasionally, these operations also caused problems in the final modal generalized mass and natural frequency calculations. For example, if a mobility matrix calculated at a particular frequency contained one element that dominated the matrix, normalization of the mobility matrix on this element would effectively submerge the influence of the matrix in the summation of the real modal mobilities. Again, the calculated modal parameters would obviously be incorrect. Similarly, if several elements of the mobility matrix measured at a specific forcing frequency were of greater magnitude than the remaining elements, the root mean square value would be affected and normalization by this value would yield a matrix wherein the elements were substantially reduced. Therefore, any such matrix would not be realistically represented in the summation of the real mobility matrices; consequently, the modal generalized mass and natural frequency would be incorrect.

Finally, each mobility matrix was multiplied by the respective forcing frequency yielding an acceleration mobility. These acceleration mobility matrices were substituted for the velocity mobilities appearing in Equation (17) and Equation (20) when iterating for the mode shapes and gamma vectors respectively. This technique was also plagued with similar difficulties that the aforementioned normalization procedures incurred. Fortunately, when the computer experiments were executed incorporating any of the previously discussed normalization methods, the conditions which yielded erroneous results were readily discernible. In these instances, the calculations for the modal generalized mass and natural frequency produced results which were obviously incorrect. For the conditions which were recognized to be in error, the computer experiments were reevaluated substituting a different normalization option. Generally, the results obtained by altering a normalization procedure yielded modal parameters which were correct.

#### INTERPRETATION OF ELEMENTS IN THE REDUCED MASS MATRIX

In general, it may be expected that the algebraic sum of all the elements of a reduced mass matrix from system identification will approximate the gross weight of the aircraft. Due partly to restraints, the sum of the elements should not exactly equal the gross weight, because masses at elastic restraints do not act as if they were ungrounded. Masses at pinned joints to ground do not even figure in the mass matrix because they do not move.

Individual mass elements cannot be interpreted as reflecting lumped physical weights at their assigned locations. The elements of any reduced mass matrix represent the inertial, as opposed to elastic and damping, dynamic effects of the two (for off-diagonal) degrees of freedom with which they are associated in an actual system having many more degrees of freedom than the model. The off-diagonal terms in a reduced mass matrix will usually be large and sometimes negative. The matrix will usually be fully populated.

The identified mass and stiffness matrices can be used to draw a dynamic circuit of the helicopter and, if any one were interested, it would be possible to construct an actual spring-mass system (utilizing both positive and negative springs and moments of inertia) which would be an exact physical duplication of the identified model, element by element, and would have the same natural frequencies and modal eigenvectors as the helicopter; but it would not "look" like a helicopter. Neither negative spring rates nor negative off-diagonal masses are physically unrealizable; the former are used by

Lockheed in its control system and by all light-switch manufacturers, the latter are the essential part of the dynamic antiresonant vibration isolator.

The objective is not to identify a system which "looks" like a helicopter but one which "performs" like a helicopter under various dynamic loadings. The physical interpretation of the  $ij$ -th element of the identified mass matrix, for example, is that the helicopter will generally exhibit a partial derivative of a force at  $i$  with respect to a response at  $j$  which has an effective\* mass component that is the  $ij$ -th element of the identified mass matrix (similarly for the stiffness and damping matrices).

It is immaterial in the identification whether there are as many points on the structure as there are degrees of freedom in the model, or if up to three degrees of freedom (in orthogonal directions) occur at any one point. It is important only that elements in the motion vector have the properties of generalized coordinates for the holonomic model considered. An identified reduced model in which some of the displacement elements represent the orthogonal Cartesian or polar coordinates of a given structural point would look much like an identified model of a similar system with parallel coordinates of separate points.

The impedance matrix, of which the mass and stiffness matrices are terms, of a mathematical model of a larger system is a function of the size of the model, and the terms must reflect this. It was found that frequency-independent mass, stiffness and damping matrices as described can accurately reflect the responses of a continuous structure over a finite spectrum by approximating a lambda matrix the inverse of which very closely approximates the mobility. The spectral mobility matrix, even of an order that equals the number of degrees of freedom in the structure, cannot be expressed as a lambda matrix.

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\*Not to be confused with the formal definition of "Effective Mass" as

$$ME_{jki} \equiv \frac{\{\phi\}_i^T [m] \{\phi\}_i}{\phi_{ji} \phi_{ki}}$$

## THE REDUCED MASS MATRIX

Consider the actual structure to consist of an infinite number of degrees of freedom of which R degrees of freedom are retained in the model. The mobility

$$\begin{bmatrix} [Y_{RR}] & [Y_{RE}] \\ [Y_{ER}] & [Y_{EE}] \end{bmatrix} = \begin{bmatrix} [Z_{RR}] & [Z_{RE}] \\ [Z_{ER}] & [Z_{EE}] \end{bmatrix}^{-1} = \left( \begin{bmatrix} [K_{RR}] & [K_{RE}] \\ [K_{ER}] & [K_{EE}] \end{bmatrix} - \omega^2 \begin{bmatrix} [M_{RR}] & 0 \\ 0 & [M_{EE}] \end{bmatrix} \right)^{-1} \quad (31)$$

The model impedance is defined as the inverse of the mobility matrix in the R degrees of freedom:

$$[Z_m] \equiv [Y_{RR}]^{-1} = [Z_{RR}] - [Z_{RE}][Z_{EE}]^{-1}[Z_{ER}] = [K_m] - \omega^2 [M_m] \quad (32)$$

The stiffness of the model,  $[K_m]$ , is the inverse of the RxR influence coefficients:

$$[K_m] \equiv [C_{RR}]^{-1} = [K_{RR}] - [K_{RE}][K_{EE}]^{-1}[K_{ER}] \quad (33)$$

From Equation (31),

$$[Z_{RR}] = [K_{RR}] - \omega^2 [M_{RR}]$$

$$[Z_{EE}] = [K_{EE}] - \omega^2 [M_{EE}]$$

$$[Z_{RE}] = [K_{RE}]$$

$$[Z_{ER}] = [K_{ER}]$$

Substitute into Equation (32)

$$\begin{aligned}
[Z_m] &= [K_m] - \omega^2 [M_m] = [K_{RR}] - \omega^2 [M_{RR}] - [K_{RE}] \left( [K_{EE}] \right. \\
&\quad \left. - \omega^2 [M_{EE}] \right)^{-1} [K_{ER}] = [K_{RR}] - [K_{RE}] [K_{EE}]^{-1} [K_{ER}] \\
&\quad - \omega^2 [M_{RR}] - [K_{RE}] \left( [I] - \omega^2 [K_{EE}]^{-1} [M_{EE}] \right)^{-1} [K_{EE}]^{-1} [K_{ER}] \\
&\quad + [K_{RE}] [K_{EE}]^{-1} [K_{ER}]
\end{aligned}$$

Substitute Equation (33). Then

$$\begin{aligned}
[M_m] &= [M_{RR}] + \frac{1}{\omega^2} [K_{RE}] \left[ \left( [I] - \omega^2 [K_{EE}]^{-1} [M_{EE}] \right)^{-1} \right. \\
&\quad \left. - [I] \right] [K_{EE}]^{-1} [K_{ER}] = [M_{RR}] + \frac{1}{\omega^2} [K_{RE}] \left[ \left( [I] \right. \right. \\
&\quad \left. \left. - \omega^2 [K_{EE}]^{-1} [M_{EE}] \right)^{-1} - \left( [I] - \omega^2 [K_{EE}]^{-1} [M_{EE}] \right) \left( [I] \right. \right. \\
&\quad \left. \left. - \omega^2 [K_{EE}]^{-1} [M_{EE}] \right) \right]^{-1} [K_{EE}]^{-1} [K_{ER}] \\
[M_m] &= [M_{RR}] + [K_{RE}] [K_{EE}]^{-1} [M_{EE}] \left( [I] \right. \\
&\quad \left. - \omega^2 [K_{EE}]^{-1} [M_{EE}] \right)^{-1} [K_{EE}]^{-1} [K_{ER}]
\end{aligned} \tag{34}$$

Equation (34) is the "exact" RxR reduced mass matrix of a system with an infinite number of degrees of freedom. Note that  $[M_m]$  is not diagonal and is a function of forcing frequency.

The frequency dependency of the "exact" reduced mass matrix simply reflects the fact that R linear differential equations with constant coefficients cannot contain enough information to exactly reflect the action of an infinite number of degrees of freedom over a spectrum containing R modes. The frequency dependency makes it impractical to use this in a linear engineering mathematical model.

The "Consistent Mass Matrix" (Reference 3), often used in finite-element dynamics work, is also based on a model stiffness matrix  $[K_m]$  being the inverse of the RxR influence coefficient matrix:

$$[K_m] = [C_{RR}]^{-1} = [K_{RR}] - [K_{RE}] [K_{EE}]^{-1} [K_{ER}].$$

The kinetic energy of the structure is set equal to the



kinetic energy of the model:

$$\begin{Bmatrix} \dot{Y}_R \\ \dot{Y}_E \end{Bmatrix}^T \begin{bmatrix} [M_{RR}] & 0 \\ 0 & [M_{EE}] \end{bmatrix} \begin{Bmatrix} \dot{Y}_R \\ \dot{Y}_E \end{Bmatrix} = \{\dot{Y}_R\}^T [M_{RR}] \{\dot{Y}_R\} + \{\dot{Y}_E\}^T [M_{EE}] \{\dot{Y}_E\} = \{\dot{Y}_R\}^T [M_m] \{\dot{Y}_R\} \quad (35)$$

It is implicitly assumed, however, that the inertial forces occur only along the R generalized coordinates, giving

$$\begin{bmatrix} [K_{RR}] & [K_{RE}] \\ [K_{ER}] & [K_{EE}] \end{bmatrix} \begin{Bmatrix} Y_R \\ Y_E \end{Bmatrix} = \begin{Bmatrix} [M_m] \ddot{Y}_R \\ 0 \end{Bmatrix}$$

which is clearly not the case but from which it follows that

$\{\dot{Y}_E\} = -[K_{EE}]^{-1}[K_{ER}]\{\dot{Y}_R\}$  in sinusoidal vibration. Sub-

stituting the above in Equation (35) gives

$$\{\dot{Y}_R\}^T [M_{RR}] + \left( [K_{ER}]^T [K_{EE}]^{-T} [M_{EE}] [K_{EE}]^{-1} [K_{ER}] \right) \{\dot{Y}_R\} = \{\dot{Y}_R\}^T [M_m] \{\dot{Y}_R\} \quad (36)$$

and the "Consistent Mass Matrix" is given by

$$[M_m] = [M_{RR}] + [K_{ER}]^T [K_{EE}]^{-T} [M_{EE}] [K_{EE}]^{-1} [K_{ER}] \quad (37)$$

This matrix is nondiagonal, like the "exact" reduced mass matrix, and has the advantage of being independent of frequency. However, comparison of Equation (37) with Equation (34) shows that the "Consistent Mass Matrix" reduces to the "exact" reduced mass matrix only at zero frequency; that is, in the static condition. As the frequency increases, the "Consistent Mass Matrix" yields increasingly erroneous results.

The reduced mass matrix in system identification is, like the others, nondiagonal and related to a model stiffness matrix which is the inverse of the RxR influence coefficients (as represented by the first R modes, which is accurate beyond direct measurement capability by many orders of magnitude); but the system identification reduced mass matrix also is independent of frequency and is exact at all the natural frequencies of the model, which are the first R natural frequencies of the helicopter. The system identification mass matrix is given by

$$[M_m] = [M_{RR}] + [\phi_{RR}]^{-T} [\phi_{ER}]^T [M_{EE}] [\phi_{ER}] [\phi_{RR}]^{-1} \quad (38)$$

$$\text{or } [M_m] \cong [C_{RR}]^{-1} [\phi_{RR}] \left[ \frac{1}{\Omega_R^2} \right] [\phi_{RR}]^{-1} \text{ very nearly.} \quad (39)$$

At the r-th natural frequency,

$$\begin{aligned} [C_{RR}]^{-1} \left[ [M_{RR}] + [K_{RE}] [K_{EE}]^{-1} \left( \frac{1}{\Omega_r^2} [I] \right. \right. \\ \left. \left. - [C_{EE}] [M_{EE}] \right) [K_{EE}]^{-1} [K_{ER}] [C_{RR}] [M_{RR}] \right] \{ \phi_{Rr} \} = \{ \phi_{Rr} \} \frac{1}{\Omega_r^2} \end{aligned} \quad (40)$$

exactly. Note in Equation (38) that the reduced system identification mass matrix is expressed in terms of the modal eigenvectors of the first R modes only but includes all the masses of the actual helicopter.

Alterations in masses on the R generalized coordinates which do not affect the modal eigenvectors are, as seen from Equation (38), exactly represented. Such alterations can substantially change natural frequencies and responses. Other types of changes which do alter the modal eigenvectors may or may not be accurately reflected in the model response depending on the degree of eigenvector effects - a limit which has not been algebraically defined for any mathematical model, whether from intuitive analysis or system identification.

That such a limit should somewhere exist is a practical engineering fact. One cannot expect to obtain the equations of a sweet pea on a rubber band, then attach it to the Golden Gate bridge and expect to find the dynamic response of the bridge (the reverse, incidentally, is equally impractical). Prudence marks the boundary between utility and uselessness.

#### INFORMATION LOSS IN MATRIX INVERSION

It is inevitable that there will be a loss in information in numerically obtaining the response matrix from any mathematical model, or in obtaining the mathematical model from responses, even if no deliberate error is introduced.

The following is a slight modification of a derivation by Rosanoff and Ginsburg (Reference 4). Consider the equation

$$[A]\{x\} = \{b\} \quad (41)$$

in which  $[A]$  is a real symmetric nonsingular matrix. Because we calculate with numbers which have a finite number of digits, we actually solve the equation

$$([A] - [E])\{x + \delta x\} = \{b\} \quad (42)$$

where  $[E]$  is an "error" matrix. Premultiplying both sides of Equation (42) by  $[A]^{-1}$  and substituting  $[A]^{-1}\{b\} = \{x\}$  gives

$$([I] - [A]^{-1}[E])\{x + \delta x\} = \{x\} \quad (43)$$

or

$$\{\delta x\} = \left[ ([I] - [A]^{-1}[E])^{-1} - [I] \right] \{x\} \quad (44)$$

Take the norm (see References 5 and 6, for example) of both sides:

$$||\{\delta x\}|| = || \left[ ([I] - [A]^{-1}[E])^{-1} - [I] \right] \{x\} ||$$

But the norm of the product of a matrix and a vector is less than the product of the matrix norm and the consistent vector norm:

$$||\{\delta x\}|| \leq || \left[ ([I] - [A]^{-1}[E])^{-1} - [I] \right] || \cdot ||\{x\}|| \quad (45)$$

or

$$\frac{||\{\delta x\}||}{||\{x\}||} \leq || \left[ ([I] - [A]^{-1}[E])^{-1} - [I] \right] ||$$

Assume that  $||[A]^{-1}[E]|| < 1$ . From Faddeeva (Reference 5), it is well known that

$$|| ([I] - [A]^{-1}[E])^{-1} - [I] + ([A]^{-1}[E]) + ([A]^{-1}[E])^2 \dots + ([A]^{-1}[E])^k || \leq \frac{||[A]^{-1}[E]||^{k+1}}{1 - ||[A]^{-1}[E]||} \quad (46)$$

if  $||[A]^{-1}[E]|| < 1$ . Setting  $k = 0$  gives

$$|| ([I] - [A]^{-1}[E])^{-1} - [I] || \leq \frac{||[A]^{-1}[E]||}{1 - ||[A]^{-1}[E]||} \quad (47)$$

or

$$\frac{||\{\delta x\}||}{||\{x\}||} \leq \frac{||[A]^{-1}[E]||}{1 - ||[A]^{-1}[E]||} \leq \frac{||[A]^{-1}|| \cdot ||[E]||}{1 - ||[A]^{-1}|| \cdot ||[E]||}$$

which is identical to the result obtained by Rosanoff and Ginsburg.

$$\frac{||\{\delta x\}||}{||\{x\}||} \leq \frac{||[A]|| \cdot ||[A]^{-1}|| \cdot ||[E]|| / ||[A]||}{1 - ||[A]|| \cdot ||[A]^{-1}|| \cdot ||[E]|| / ||[A]||} = \frac{k_n^\ell}{1 - k_n^\ell} \quad (48)$$

where, following Rosanoff et al,  $k_n$  is defined as a conditioning number and  $\ell$  as a relative error:

$$k_n \equiv ||[A]|| \cdot ||[A]^{-1}||$$

$$\ell \equiv ||[E]|| / ||[A]||$$

Taking the number of digits in the arithmetic as  $\log_{10} \frac{1}{\ell}$ , the reciprocal of Equation (48) gives an estimate of the number of significant digits  $p$ .

$$p = \log_{10} ||x|| - \log_{10} ||\delta x|| \geq \log_{10} (1 - k_n^\ell) + \log_{10} \frac{1}{\ell} - \log_{10} k_n$$

but, assuming  $1 \gg k_n^\ell$ , this estimate may be written

$$p = \log_{10} \frac{1}{k} - \log_{10} k_n \quad (49)$$

Thus, as shown in Reference 4, the number of information digits  $q$  lost in inverting  $[A]$  is approximately

$$q = \log_{10} k_n = \log_{10} ||[A]|| \cdot ||[A]^{-1}|| \quad (50)$$

This is true for any norm. However, the norm of a symmetrical positive definite matrix, subordinate to the Euclidian vector, is the maximum eigenvalue; and the maximum eigenvalue of the inverse is the reciprocal of the minimum eigenvalue of the matrix. Substituting this norm of  $[A]$  into Equation (50) gives the lost digits estimate.

$$q = \log_{10} \frac{\lambda(A)_{\max}}{\lambda(A)_{\min}} \quad (51)$$

To illustrate the immense practical importance of this, consider as an example a matrix having

$$\frac{\lambda(A)_{\max}}{\lambda(A)_{\min}} = 1.72 \times 10^3$$

This is the ratio of natural frequencies in the 20x20 specimen of the helicopter used in this contract. The IBM 360 uses six hexadecimal places resulting in  $16^6-1$  or 16777215 as the largest decimal mantissa in single precision. The inversion of the  $k^{-1}_m$  matrix with single precision on the computer results in an inverse having (estimated)  $\log_{10} 16777215 - \log_{10} 1.72 \times 10^3 = 3.99$  significant digits. In other words, even starting with eight decimal places in floating point, we end up with approximately four decimal places of information in the inverse.

It is absolutely essential when dealing with test data matrices which will be inverted that the ratio of the extreme eigenvalues be minimized. Otherwise, all the physical information in the matrix is likely to be destroyed in the inversion, leaving meaningless numbers. Test data has few enough significant figures of information to begin with.

## HOW TO MINIMIZE INFORMATION LOSS

A major step in this system identification process is the determination of the  $\{\gamma\}$  and  $\{\phi\}$  vectors by iteration. The matrix involved is the product of a mobility matrix and the inverse of another mobility matrix. This inverse presents a serious danger of information loss.

To minimize the extraneous information content of modes higher than the order of the matrix, which amounts to noise, and to narrow the spread of modal mobilities, the matrix to be inverted was made the sum of the dissipative (e.g.,  $[\ddot{Y}^*I]$ ) matrices measured near each natural frequency.

Each dissipative mobility matrix has a high information content about the dominant mode and very little information about other modes. This minimizes pollution by unwanted modes but results in a very poorly conditioned matrix. For example, the 10x10 imaginary acceleration mobility of a typical helicopter measured at 3 Hz has an extreme modal mobility ratio of  $10^6$ . However, the sum of mobility matrices over the frequency range is a matrix having the same modal vectors as a mobility matrix at any one frequency.

$$[\ddot{Y}_{\omega_p}^*I] = \sum_{i=1}^N \ddot{Y}_{\omega_p i}^*I \{\phi_i\} \{\phi_i\}^T = [\Phi] \left[ \ddot{Y}_{\omega_p i}^*I \right] [\Phi]^T \quad (52)$$

$$\sum_{\omega} [\ddot{Y}_{\omega}^*I] = \sum_{i=1}^n \sum_{\omega} \ddot{Y}_{\omega i}^*I \{\phi_i\} \{\phi_i\}^T = [\Phi] \left[ \sum_{\omega} \ddot{Y}_{\omega i}^*I \right] [\Phi] \quad (53)$$

Therefore, Equation (53) can be used as one of the matrices in the modal eigenvector equations

$$[\sum_{\omega} \ddot{Y}_{\omega}^*I]^{-1} [\ddot{Y}_{\omega}^*I] \{\gamma\}_i = \lambda \{\gamma\}_i \quad (54)$$

$$[\ddot{Y}_{\omega}^*I] [\sum_{\omega} \ddot{Y}_{\omega}^*I]^{-1} \{\phi\}_i = \alpha \{\phi\}_i \quad (55)$$

The range of values from the maximum to the minimum in  $\sum_{\omega} \ddot{Y}_{\omega i}^*I$  is very small compared to the range of  $\ddot{Y}_{\omega i}^*I$ .

If  $\sum_{\omega} [\ddot{Y}_{\omega}^I]$  or  $\sum_{\omega} [\dot{Y}_{\omega}^R]$  is used in place of  $\sum_{\omega} [\ddot{Y}_{\omega}^I]$  it is necessary to normalize each of the matrices in the sum because the displacement and velocity mobilities decrease in magnitude with increased frequency. Normalization on the root mean square of the matrix elements and on the largest element absolute value were both investigated experimentally. Normalization on the RMS gave results about as satisfactory as those from acceleration mobility and is preferred over normalization on the largest element, as the latter is sensitive to errors in one term which could throw off the entire matrix. However, the differences in results, while evident, were not dramatic.

The  $\log_{10}$  of the ratio of the maximum  $\sum_{\omega} [\ddot{Y}_{\omega i}^{*I}]$  to the minimum was generally about .75 for the 5x5 models in these experiments and generally around 1.8 for the 15 x 15 models. The 5 x 5 models performed excellently but the 15 x 15 models performed capriciously.

If the engineer could normalize so that the matrix  $[\sum_{\omega} [\ddot{Y}_{\omega i}^{*I}]]$  is unity, information would still be lost in the inversion but certainly less information than if  $\sum_{\omega} [\ddot{Y}_{\omega i}^{*I}]$  terms the ratio of extreme values is very high. The  $\sum_{\omega} [\ddot{Y}_{\omega i}^{*I}]$  terms

are not the eigenvalues of  $\sum_{\omega} [\ddot{Y}_{\omega}^I]$ . The only matrix which has a unit eigenvalue matrix is the unit matrix itself; it follows therefore that some information is always lost in the numerical inversion of any matrix other than unity.

The matrix we wish to invert is

$$\sum_{\omega} [\ddot{Y}_{\omega}^I] = [\Phi] \left[ \sum_{\omega} [\ddot{Y}_{\omega i}^{*I}] \right] [\Phi]^T \quad (56)$$

Express the modal vector matrix in terms of its own eigenvectors  $[J]$  and its own eigenvalues  $\lambda_{\phi}$  (that is,  $[J]$  is the eigenvector matrix of the eigenvector matrix of  $[[k]^{-1}[m]]$ ).

$$[\Phi] = [J] \lambda_{\phi} [J]^{-1} \quad (57)$$

Substitute Equation (57) into Equation (56).

$$\Sigma_{\omega} [\ddot{Y}_{\omega}^I] = [J] [\lambda_{\phi}] [J]^{-1} [\Sigma_{\omega i}^I] [J]^{-T} [\lambda_{\phi}] [J]^T \quad (58)$$

invert,

$$\Sigma_{\omega} [\ddot{Y}_{\omega}^I]^{-1} = [J]^{-T} \left[ \frac{1}{\lambda_{\phi}} \right] [J]^T \left[ \frac{1}{\Sigma_{\omega i}^I} \right] [J] \left[ \frac{1}{\lambda_{\phi}} \right] [J]^{-1} \quad (59)$$

The only operation on the eigenvectors  $[J]$  between Equation (58) and Equation (59) was to change relative positions; all the inversions were of diagonal matrices. As a diagonal matrix is a matrix of its own eigenvalues, having the unit matrix for eigenvectors, the central term may be treated rigorously as an eigenvalue matrix. The matrix of Equation (58) may be substituted for  $[A]$  in Equation (41), and in Equation (50), we can consider  $[A]$  as the product of the three matrices of Equation (56).

$$\Sigma_{\omega} [\ddot{Y}_{\omega}^I] = [A] = [\Phi] [\Sigma_{\omega i}^{*I}] [\Phi]^T \quad (60)$$

It is well known that

$$||[A]|| \leq ||[\Phi]|| \cdot ||[\Sigma_{\omega i}^{*I}] [\Phi]^T|| \quad (61)$$

and that

$$||[\Sigma_{\omega i}^{*I}] [\Phi]^T|| \leq ||[\Sigma_{\omega i}^{*I}]|| \cdot ||[\Phi]^T|| \quad (62)$$

Therefore

$$||[A]|| \leq ||[\Phi]|| \cdot ||[\Sigma_{\omega i}^{*I}]|| \cdot ||[\Phi]^T|| \quad (63)$$

At this point we wish to substitute eigenvalues, but  $[\Phi]$  is not symmetric so  $||[\Phi]|| \neq |\max \lambda_{\phi}|$ . Rather,  $||[\Phi]|| \geq |\max \lambda_{\phi}|$ . Consider, therefore, the eigenvalues  $\lambda_b$  of  $[\Phi]^T [\Phi]$  which is symmetrical.

$$[\Phi]^T [\Phi] = [L] [\lambda_b] [L]^{-1} = [L] [\lambda_b] [L]^T \quad (64)$$

where  $[L]$  is the orthogonal matrix of eigenvectors of  $[\Phi]^T [\Phi]$ .

$$||[\Phi]|| = |\max \sqrt{\lambda_b}| \quad (65)$$



Substitute Equation (65) into Equation (63).

$$||[A]|| \leq |\max \lambda_b| \cdot |\max \ddot{Y}_{\omega i}^{*I}| \quad (66)$$

Using Equation (51), the number of digits lost in inverting  $\Sigma[Y_{\omega}^*]$  is approximated by

$$\begin{aligned} q &\approx \log_{10} \frac{|\max \lambda_b| \cdot |\max \ddot{Y}_{\omega i}^{*I}|}{|\min \lambda_b| \cdot |\min \ddot{Y}_{\omega i}^{*I}|} \\ &= \log_{10} \frac{|\max \lambda_b|}{|\min \lambda_b|} + \log_{10} \frac{|\max \ddot{Y}_{\omega i}^{*I}|}{|\min \ddot{Y}_{\omega i}^{*I}|} \quad (67) \end{aligned}$$

$[\lambda_b]$  would equal a scalar times the unit matrix only if the modal vectors  $\{\phi\}$  were orthogonal (i.e.,  $\{\phi_i^T\} \{\phi_j\} = 0$ ), a condition which could occur only in the academic cases of uniform mass:  $[m] = [m][I]$ . In this case, the loss of information digits would be indicated by

$$q \approx \log_{10} \frac{|\max \ddot{Y}_{\omega i}^{*I}|}{|\min \ddot{Y}_{\omega i}^{*I}|} \quad (68)$$

and only in this case could zero information loss be achieved by normalizing the matrices such that  $|\max \ddot{Y}_{\omega i}^{*I}|/|\min \ddot{Y}_{\omega i}^{*I}| = 1$ .

But the case is trivial, for if it were true, an inversion would be unnecessary as  $\phi$  would be the eigenvector matrix of  $\Sigma[Y_{\omega}^*]$ .

If the mass distribution is not uniform diagonal but the engineer could so normalize the matrices in the summation so that  $|\max \ddot{Y}_{\omega i}^{*I}|/|\min \ddot{Y}_{\omega i}^{*I}| = 1$ , it is seen from Equation (67) that there would still be a loss of information digits approximated by

$$q \approx \log_{10} \frac{|\max \lambda_b|}{|\min \lambda_b|} \quad (69)$$

The ratio  $|\max \lambda_b|/|\min \lambda_b|$  increases with the order of the mobility matrix; that is, with the number of degrees of freedom of the model. It follows, therefore, that there is an upper limit to the size of a physically meaningful reduced complete model regardless of normalization of the matrices in the summation.

As a crude "rule of thumb", Figure 1 shows the trend in the reliability of the inversion of  $\sum_{\omega} [\ddot{Y}_{\omega}^I]$ .

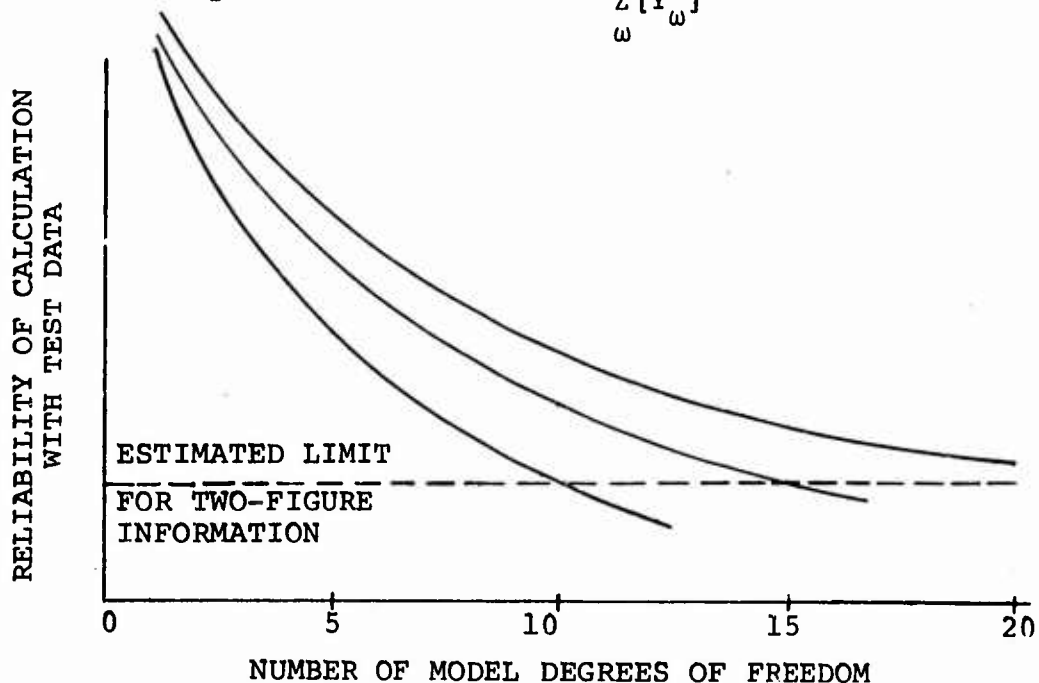


Figure 1. Reliability of the Inversion of  $\sum_{\omega} [\ddot{Y}_{\omega}^I]$ .

It is seen that the reliability of the calculation becomes questionable above 10 or 15 degrees of freedom. This does not mean that accurate identifications cannot be made using the iterative step for modes of, say, 20 degrees of freedom but, rather, that any one calculation has a higher probability of failure.

In passing, it should be noted that the treatment of bounds using matrix norms, as above, opens up some highly promising avenues of research on the reliability of many helicopter

theoretical calculations as well as on the reliability of the processing of test data in general. Whether, for example, some of the conventional methods of processing strain gage data yield physically meaningful results is open to question in the light of the above method.

#### WHEN A CALCULATION FAILS

The most common mode of failure of iteration on  $[\ddot{Y}_\omega^I](\Sigma[\ddot{Y}_\omega^I])^{-1}$  is catastrophic, producing such absurd values for one or more generalized masses as negative numbers or unusually large numbers. This is signified also by a very large number of iterations required for convergence on one or more modes. Failures almost never occur with small number of degrees of freedom (e.g., five), and an identification which is quite accurate with one seed may, in the larger models, diverge with another seed.

This phenomenon results from the fact that the significant effect of error is not insidious accumulation of inaccuracies in the generalized masses but, rather, information destruction in the inversion. Fortunately, it is usually very obvious to the engineer when an identification fails on the computer, and corrective measures may be taken without rerunning the test on the helicopter.

A most obvious and effective corrective measure is to eliminate one or more of the degrees of freedom. This can be done on the computer, as the program is written so that the system may be instructed to select any of the available data which is in digital form on tape. The size of the model and the number of modes covered are consequently reduced. It is possible also to eliminate any mode, not just the highest, if it appears that a certain mode contributes little information - a local resonance, for example, in which only a small portion of the helicopter is significantly responding.

The computer experiments included a local resonance in the form of a mode in which only the most forward station showed substantial motion. When this station was not included in the identification, but the local resonance associated with its movement was included, then, as expected, there were evidences of failure in generalized mass calculation. The computer was attempting to identify a natural mode for which the input mobility data showed a largely nonresponding helicopter. This situation would be detected from the mobility plots before committing the data, as it is very apparent in the dissipative mobility spectra. The

ability to handle local resonances, or dispose of them when required, is important to a practical identification because all real structures have them. In fact, as the number of degrees of freedom of a simulated structure are realistically increased, the modal density usually increases more rapidly than the simple mathematics of uniform chains would lead one to believe. When that degree of freedom which is the pre-dominant motion of a local resonance is eliminated, the mode it causes should be eliminated also; the mobility spectra plots for the included degrees of freedom would indicate this by an insignificant peak.

#### THE $\Gamma$ MATRIX AND MODAL PARAMETERS

The dominant modal vector at frequency  $\omega_i$ , near the  $i$ -th natural frequency  $\Omega_i$ , is given by Equation (55) and the  $i$ -th gamma vector by iteration on the transpose, Equation (54). The modal mobility is obtained from

$$\{\gamma_i\}_{(ITR)}^T [\bar{Y}_{\omega_i}^I] \{\gamma_i\}_{(ITR)} = \bar{Y}_{i\omega_i}^{*I} \quad (70)$$

where  $\{\gamma_i\}_{(ITR)}$  is the vector from iteration (Equation 54).

It is impractical to attempt the calculation using  $\{\gamma_i\}$  from  $[\Phi]^{-T}$  because of information loss in the inversion, as shown in Equation (59). The dominant mode is the only one used, of course, as there is negligible information content in  $[\bar{Y}_{\omega_i}^I]$  about modes other than the  $i$ -th. Therefore,

$$[\bar{Y}_{\omega_i}^I] \approx \bar{Y}_{i\omega_i}^{*I} \{\phi_i\} \{\phi_i\}^T \approx \bar{Y}_{i\omega_i}^{*I} \{\phi_i\}_{(ITR)} \{\phi_i\}_{(ITR)}^T \quad (71)$$

and  $\{\gamma_i\}_{(ITR)}^T \{\phi_i\}_{(ITR)} = 1$  is forced.

A peculiar situation often occurred when a calculation diverged: it was noticed that the natural frequency of the "bad" mode was usually identified with great accuracy although the calculated generalized mass was absurd, often negative, and negative calculated values of  $\bar{Y}_{i\omega_i}^{*R}$  often occurred.

The key here is the occurrence of negative values of  $\dot{Y}_{i\omega_i}^{*R}$ . Ideally,  $[\dot{Y}^R]$  is a positive definite matrix and cannot, theoretically, be negative definite on grounds that it represents the dissipation, not a source, of energy. For any positive definite matrix  $B$ ,  $\{x\}^T[B]\{x\}$  is a positive number regardless of the choice of the vector  $\{x\}$ . The fault for negative values of  $\dot{Y}_{i\omega_i}^{*R}$ , which are physically impossible,

cannot therefore be laid solely to  $\{\gamma\}$ , and therefore to the loss of information in the inverse of  $\sum_{\omega} [\dot{Y}_{\omega}^I]$ , because

$\{\gamma_i\}^T [\dot{Y}^R] \{\gamma_i\}$  must be positive even for arbitrary  $\{\gamma_i\}$  if  $[\dot{Y}^R]$  is, as it is supposed to be, positive definite. We are forced to conclude that numerical errors can act in such a way as to make  $[\dot{Y}^R]$  not positive definite.

The mobility  $[\dot{Y}_{\omega_i}^R]$  is very nearly equal to the positive semi-definite matrix  $\left[ \dot{Y}_{i\omega_i}^{*R} \{\phi_i\} \{\phi_i\}^T \right]$  in which  $\dot{Y}_{i\omega_i}^{*R}$  is necessarily positive. Then

$$\{\gamma_i\}^T [\dot{Y}_{i\omega_i}^{*R} \{\phi_i\} \{\phi_i\}^T] \{\gamma_i\} = \dot{Y}_{i\omega_i}^{*R} (\{\gamma_i\}^T \{\phi_i\})^2 \quad (72)$$

But  $\{\gamma_i\}$  and  $\{\phi_i\}$  are composed of real numbers, as opposed to imaginary or complex numbers, which makes  $\{\gamma_i\}^T \{\phi_i\}$  real and  $(\{\gamma_i\}^T \{\phi_i\})^2$  real and positive even for arbitrary elements in  $\{\gamma\}$ . The dominance of  $[\dot{Y}_{\omega_i}^R]$  by one mode is therefore not a cause of calculating negative values of  $\dot{Y}_{i\omega_i}^{*R}$ .

The calculation of absurd values of  $\dot{Y}_{i\omega}^{*R}$  is nevertheless due mainly to information loss in inverting  $\sum_{\omega} [\dot{Y}_{\omega}^I]$  (or other normalized mobility matrices having similar properties), which results in poor eigenvectors in the iteration. Examination of the computer experiments shows that errors in the  $[Y^R]$  or  $[Y^I]$  matrices are not sufficient to cause as erratic results as have sometimes been observed if the  $\{\gamma\}$  vectors in  $\{\gamma\}^T [Y] \{\gamma\}$  are accurate. In the "bad" cases, the  $\{\gamma\}$  vectors from iteration are invariably very bad. The reason for the occasional negative calculated values of  $\dot{Y}_{i\omega}^{*R}$  is, in part,

that errors in  $[Y]$  can cause the matrix to not be positive definite. For example, in Computer Experiment 188 a nine-point identification with error yielded good results but the same identification with a different seed (Computer Experiment 184) gave poor results which included negative  $\dot{Y}_{i\omega}^{*R}$  for

the seventh mode. The errors by chance happened to act in such a way in Experiment 184 that excessive information was lost in the inverse, as indicated by iterations that failed to converge. The principal minor associated with the eighth and ninth positions in mobilities dominated by the seventh mode was found to be negative in the bad case (Experiment 184), due to a peculiar accumulation of random errors, which, of course, meant that the mobility was no longer positive definite, as in pure theory, and could give negative values of  $\{\gamma\}_i^T [Y_{\omega_i}] \{\gamma_i\}$ . However, precise  $\{\gamma\}$  vectors would not have

caused the negative values of  $\dot{Y}_{i\omega}^{*R}$  even with  $[Y]$  not being positive semidefinite.

Calculation of physically meaningless values of  $\dot{Y}_{i\omega}^{*R}$ , and therefore of  $M^*$ , is caused primarily by information loss in inversion.

The reason for fairly accurate identifications of natural frequencies even when the generalized mass identifications are poor lies in the fact that  $\omega_j$  and  $\omega_k$  in Equation (22) are taken near  $\Omega_i$ ; therefore,

$$\frac{\omega_j \dot{Z}_{i\omega_k}^{*I} - \omega_k \dot{Z}_{i\omega_j}^{*I}}{\omega_k \dot{Z}_{i\omega_k}^{*I} - \omega_j \dot{Z}_{i\omega_j}^{*I}} \approx 1 \quad (73)$$

and

$$\Omega_i^2 = \omega_j \omega_k \frac{\omega_j \dot{Z}_{i\omega_k}^{*I} - \omega_k \dot{Z}_{i\omega_j}^{*I}}{\omega_k \dot{Z}_{i\omega_k}^{*I} - \omega_j \dot{Z}_{i\omega_j}^{*I}} \approx \omega_j \omega_k = (\Omega_i - \delta\omega_j)(\Omega_i + \delta\omega_k)$$

$$\omega_j \omega_k = \Omega_i^2 + \Omega_i(\delta\omega_k - \delta\omega_j) - \delta\omega_j \delta\omega_k$$

But  $\delta\omega_k \ll \delta\omega_j$  so  $\Omega_i^2 \approx \omega_j \omega_k$ .

## IDENTIFIED GENERALIZED MASSES

Typical generalized mass identifications are shown in Tables I through VI. Note in Tables I, III and V that the generalized mass of the first mode identified for a reduced model with no experimental error has always been less than the first mode generalized mass calculated from the modal vector and mass matrix of the specimen. This is not true of other modes.

Tables I and II show results of two different five-point models. No outstanding differences between the models is evident. Model 9A produced acceptable results, as shown in Table III, for different distribution of random error but Model 9B, as shown in Table IV, worked with some seeds and failed with other seeds. The failed experiments of Table IV, Computer Experiments 168 and 184, yielded drastically unrealistic values of generalized mass for most of the modes.

Table V shows a twelve-point model identification which failed only in the eighth mode. Computer Experiment 178 is identical to Computer Experiment 169 except that in the former, the computer was instructed to skip the eighth mode and, instead, operate on tape data for the thirteenth mode which resulted in satisfactory identification.

Using different stations for a twelve-point model, as shown in Table VI, produced proper identification of all models, including the eighth, with various error distributions.

Information loss in the inversion of mobility matrices is the primary cause of such failures, as shown in Computer Experiments 168, 184 and 169. The averaging of mobility test data, properly done, would greatly minimize the chances of such identification failures. Test data averaging is the customary practice. These computer experiments did not take advantage of averaging experiments.

TABLE I. IDENTIFICATION OF GENERALIZED MASSES,  
5 X 5 MODEL\* OF 20 X 20 SPECIMEN

Computer Experiment Number	152	151	157	160	182	1**
Random Amp Error	0	<u>+5%</u>	<u>+5%</u>	<u>+5%</u>	<u>+5%</u>	0
Bias Amp Error	0	+5%	+5%	+5%	+5%	0
Random Phase Error	0	<u>+1</u>	<u>+1°</u>	<u>+1°</u>	<u>+1°</u>	0
Seed	-	246	221	195	327	-

Stations (In.)	Mode	Generalized Masses (Lb-Sec <sup>2</sup> /In.)					
0	1	7.9910	7.3594	7.3834	7.8421	8.2330	8.5341
120	2	4.6248	3.8247	4.5951	4.1440	4.0594	4.4491
220	3	.4951	.4618	.4771	.4653	.4729	.4951
340	4	1.0897	1.0372	1.0657	1.0366	1.0440	1.0872
460	5	.6463	.5869	.6247	.6691	.6131	.6302

\* Model 5A

\*\* From 20 x 20 Model



TABLE II. IDENTIFICATION OF GENERALIZED MASSES,  
5 X 5 MODEL\* OF 20 X 20 SPECIMEN

Computer Experiment Number	159	170	183	1**		
Random Amp Error	<u>+5%</u>	<u>+5%</u>	<u>+5%</u>	0		
Bias Amp Error	+5%	+5%	+5%	0		
Random Phase Error	<u>+1°</u>	<u>+1°</u>	<u>+1°</u>	-		
Seed	221	246	128	-		
	Stations (In.)	Mode	Generalized Masses (Lb-Sec <sup>2</sup> /In.)			
	0	1	7.4385	7.3210	7.8179	8.5341
	100	2	4.4545	4.1824	4.3797	4.4491
	200	3	.4724	.4620	.4596	.4951
	320	4	1.0769	1.0277	1.0233	1.0872
	460	5	.6912	.5945	.6360	.6302
* Model 5B						
** From 20 x 20 Model						

TABLE III. IDENTIFICATION OF GENERALIZED MASSES,  
9 X 9 MODEL\* OF 20 X 20 SPECIMEN

Computer Experiment Number	180	156	162	179	187	1**	
Random Amp Error	0	<u>+5%</u>	<u>+5%</u>	<u>+5%</u>	<u>+5%</u>	0	
Bias Amp Error	0	+5%	+5%	+5%	+5%	0	
Random Phase Error	0	<u>+1°</u>	<u>+1°</u>	<u>+1°</u>	<u>+1°</u>	0	
Seed	-	287	50	315	492	-	
Stations (In.)	Mode	Generalized Masses (Lb-Sec <sup>2</sup> /In.)					
0	1	7.9538	7.3776	7.5946	8.2378	7.4531	8.5342
30	2	4.5889	4.1130	4.2450	4.6233	4.2020	4.4491
100	3	.4938	.4671	.4821	.4614	.4656	.4951
160	4	1.0863	1.0507	1.0368	1.0129	1.0785	1.0872
220	5	.6350	.6164	.6044	.6102	.5971	.6302
280	6	.7457	.7049	.6983	.7227	.7239	.7429
340	7	1.1746	1.1204	1.1332	1.1064	1.0968	1.1769
400	8	1.5002	1.3770	1.4070	1.4193	1.4783	1.4683
460	9	.6593	.6576	.5507	.6235	.5737	.7866
* Model 9A							
** From 20 x 20 Model							

TABLE IV. IDENTIFICATION OF GENERALIZED MASSES,  
9 X 9 MODEL\* OF 20 X 20 SPECIMEN

Computer Experiment Number		161	188	168	184	1**
Random Amp Error		±5%	±5%	±5%	±5%	0
Bias Amp Error		+5%	+5%	+5%	+5%	0
Random Phase Error		±1	±1	±1	±1	-
Seed		287	206	395	619	-
Stations (In.) Mode		Generalized Masses (Lb-Sec <sup>2</sup> /In.)				
0	1	7.4445	7.5891	7.6797	7.0969	8.5341
60	2	4.2851	4.4084	23.2234	4.5730	4.5615
120	3	.4741	.4545	.6876	.4314	.4951
180	4	1.0194	1.0226	28.5896	1.0968	1.0872
240	5	.6343	.6740	.5667	-7.9847	.6302
280	6	.7020	.6987	-8.5143	.5237	.7429
320	7	1.1877	1.0711	-.0080	.0125	1.1769
400	8	1.2510	1.7815	.1256	-.2199	1.4683
460	9	.9347	.9398	-.0159	-.0810	.9836
* Model 9B						
** From 20 x 20 Model						

TABLE V. IDENTIFICATION OF GENERALIZED MASSES,  
12 X 12 MODEL\* OF 20 X 20 SPECIMEN

Computer Experiment Number		169	178	1**
Random Amp Error		+5%	+5%	0
Bias Amp Error		+5%	+5%	0
Random Phase Error		+1°	+1°	0
Seed		492	492	-
Stations (In.)	Mode	Generalized Masses (Lb-Sec <sup>2</sup> /In.)		
0	1	7.4551	7.4629	8.5341
60	2	4.1298	3.9789	4.4491
100	3	.4587	.4657	.4951
120	4	1.0446	1.0376	1.0872
160	5	.5950	.5802	.6302
200	6	.6869	.6975	.7429
240	7	1.2036	1.2044	1.2569
280	8	-7.9616		2.0521
320	9	.9410	.9118	.9836
370	10	.0425	.0428	.0432
430	11	.1718	.1752	.1723
460	12	1.0012	1.0037	1.0480
	13	-	.7924	.5724
* Model 12D				
** From 20 x 20 Model				

TABLE VI. IDENTIFICATION OF GENERALIZED MASSES, 12 X 12 MODEL* OF 20 X 20 SPECIMEN						
Computer Experiment Number		150	149	155	163	1**
Random Amp Error		0	+5%	+5%	+5%	0
Bias Amp Error		0	+5%	+5%	+5%	0
Random Phase Error		0	+1°	+1°	+1°	0
Seed		-	23	492	87	-
Stations (In.)	Mode	Generalized Masses (Lb-Sec <sup>2</sup> /In.)				
0	1	7.9718	7.7160	7.2917	7.4071	8.5342
30	2	4.6071	4.5010	4.2722	4.3406	4.4491
60	3	.4941	.4640	.4682	.4611	.4951
100	4	1.0857	1.0499	1.0625	1.0425	1.0872
140	5	.6348	.6094	.5958	.5936	.6302
180	6	.7441	.7155	.6930	.7097	.7429
220	7	1.1765	1.1433	1.1101	1.1278	1.1769
260	8	1.4158	1.3467	1.3225	1.3454	1.4115
300	9	.7808	.7329	.7395	.7362	.7866
340	10	.0430	.0419	.0422	.0422	.0432
400	11	.1705	.1596	.1665	.1689	.1723
460	12	.9112	.5712	.8417	1.0946	1.3235
* Model 12B						
** From 20 x 20 Model						

## RESPONSE FROM IDENTIFIED MODEL

Figures 2 through 7 portray typical acceleration response obtained from the various models investigated in the present study. In each instance, the exact curve was obtained from the twenty-point structure with zero error. Figure 2 indicates the effect of random number seed for a typical five-point model. Figure 3 presents the results obtained for one of the nine-point models considered in the investigation. Figure 4 portrays the effect of random number seed on the twelve-point model. All the computer experiments which considered error used a  $\pm 5\%$  random, 5% bias and a  $1^\circ$  phase error.

Figure 5 presents the effect of model variation on the acceleration response. The models varied in that different spanwise masses were considered. Model 5A utilized stations 0, 120, 220, 340 and 460 (inches) whereas model 5B consisted of stations 0, 100, 200, 320, and 460 (inches). Figure 6 presents the effect of model for the nine-point model. The model 9A consisted of stations 0, 30, 100, 160, 220, 280, 340, 400 and 460 (inches). Model 9B included stations 0, 60, 120, 180, 240, 280, 320, 400 and 460 (inches). The twelve-point model 12B used stations 0, 30, 60, 100, 140, 180, 220, 260, 300, 340, 400 and 460 (inches) whereas model 12E utilized stations 0, 30, 60, 100, 120, 160, 200, 260, 280, 340, 400, 460 (inches). For each model the computer experiments were executed using the same random number seed and the aforementioned errors were incorporated.

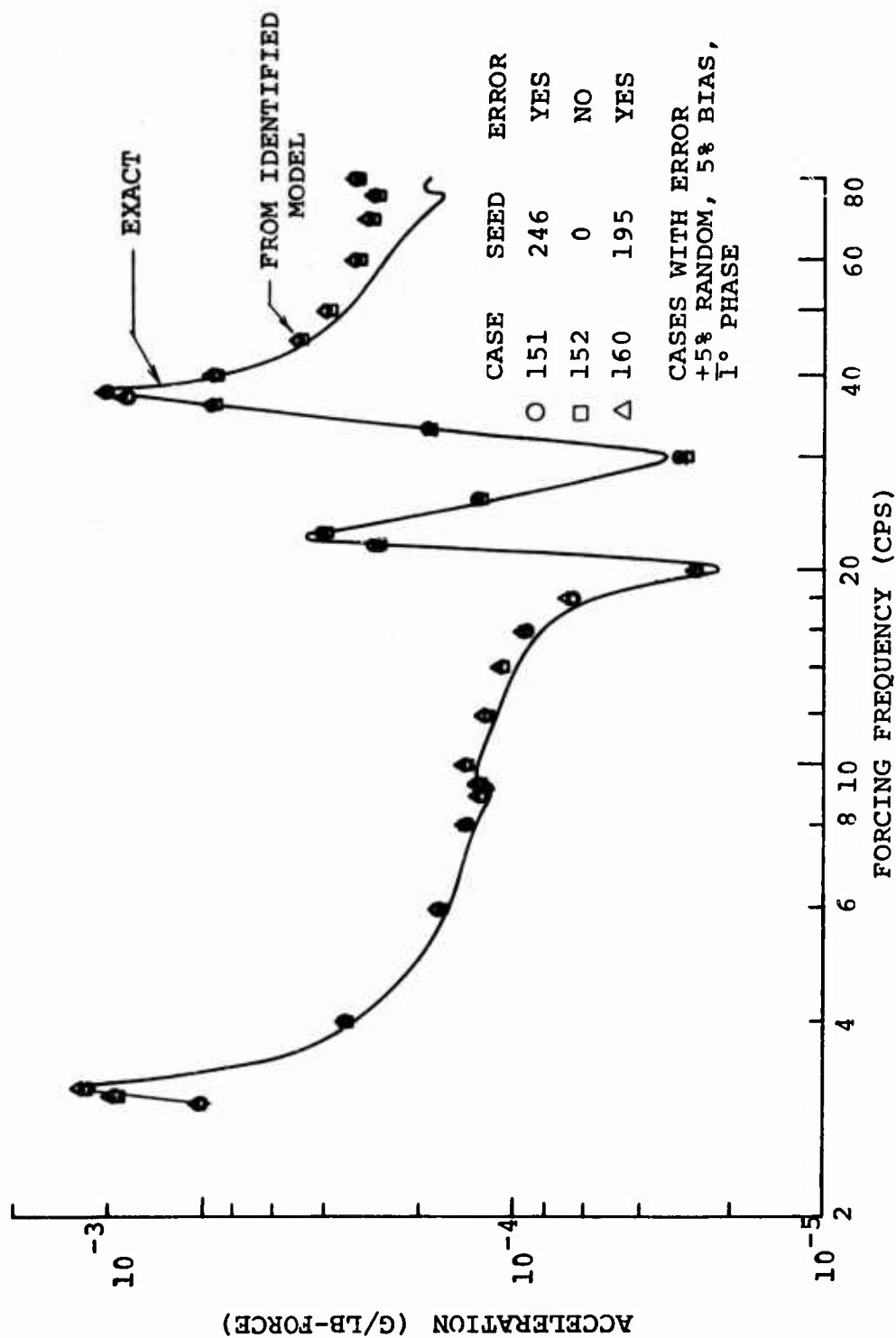


Figure 2. Five-Point Model Response Obtained From Equations With Identified Parameters; Driving Point at Hub.

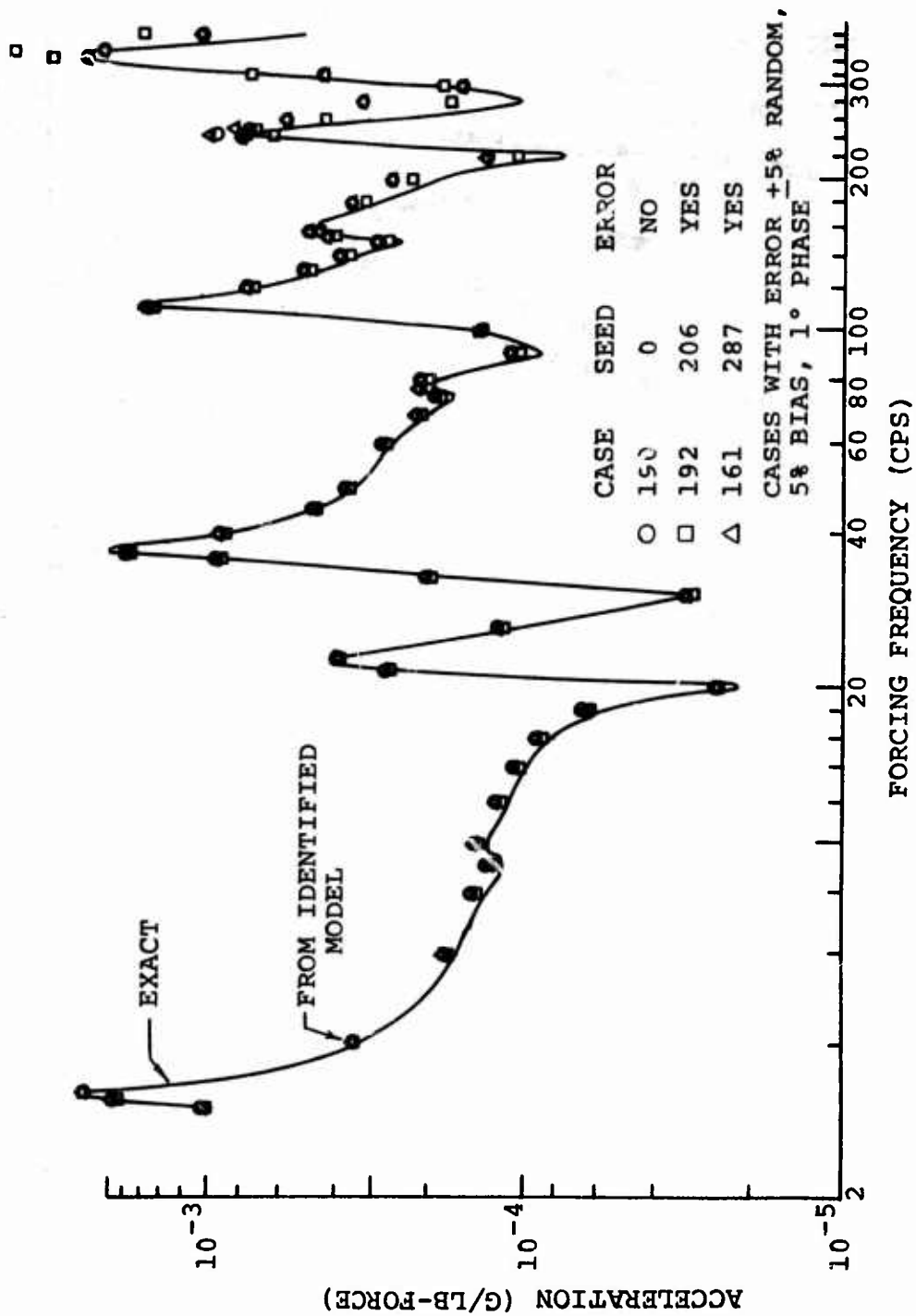


Figure 3. Nine-Point Model Response Obtained From Equations With Identified Parameters; Driving Point at H.v.b.



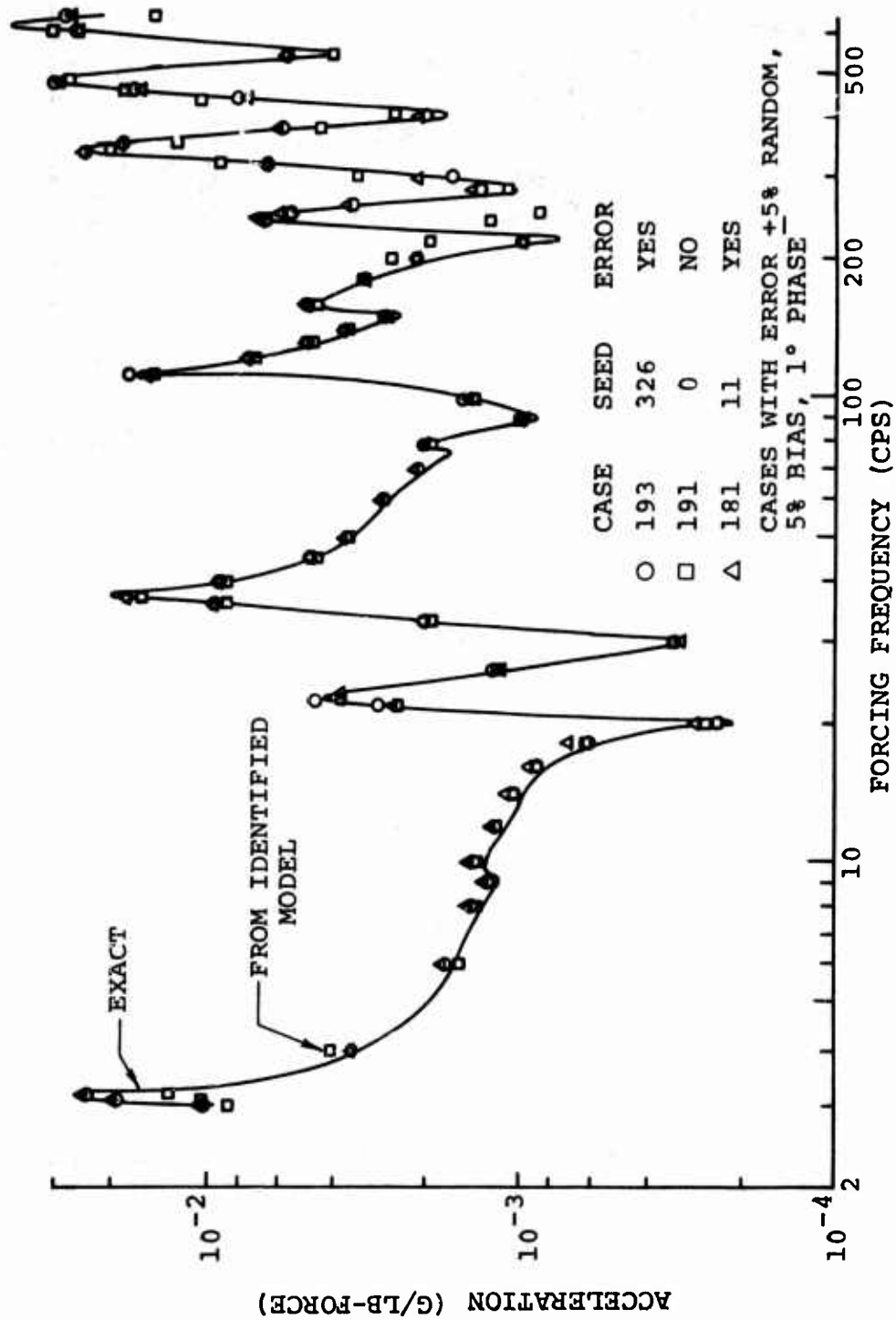


Figure 4. Twelve-Point Model Response Obtained From Equations With Identified Parameters; Driving Point at Hub.

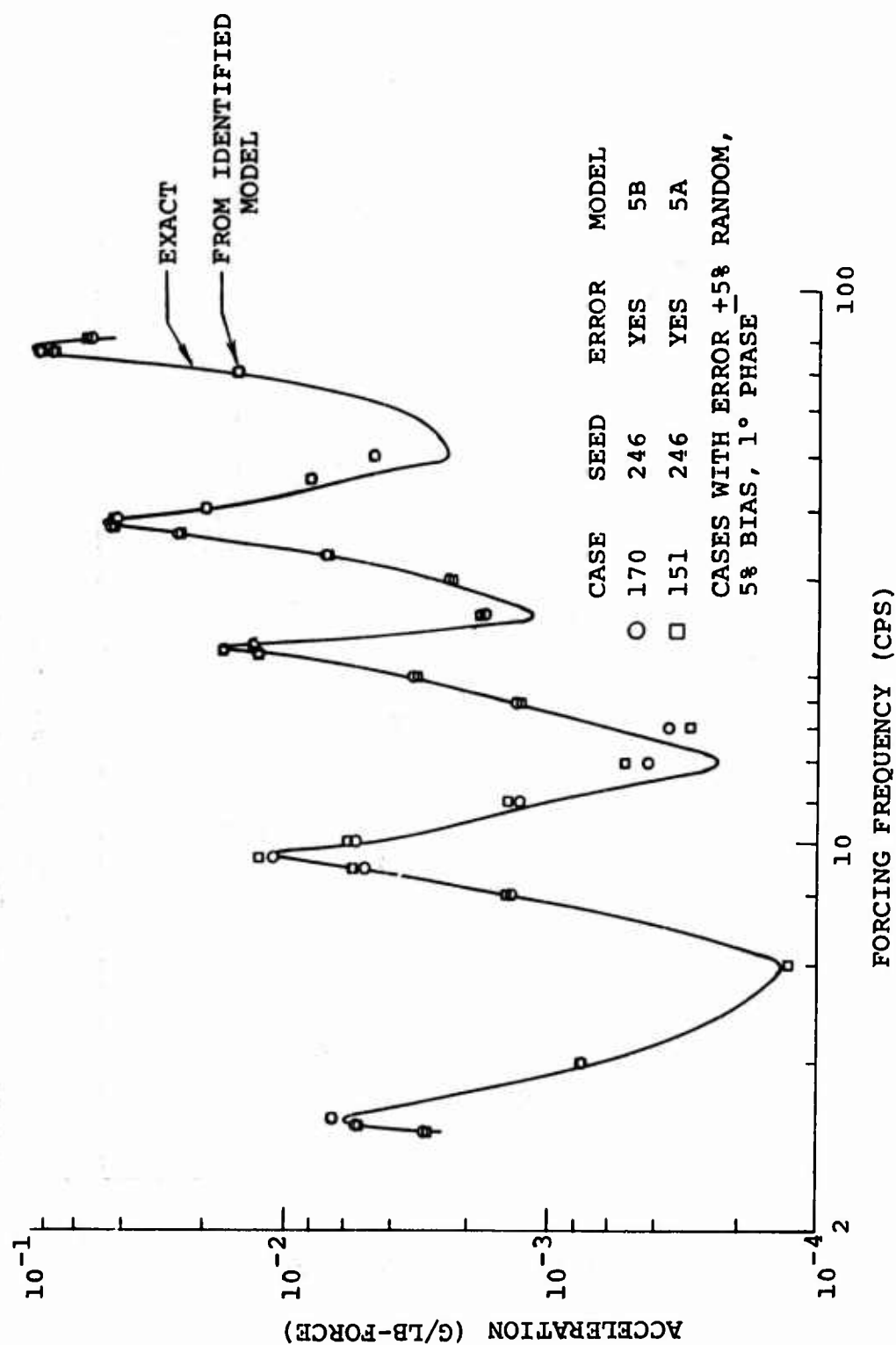


Figure 5. Five-Point Model Response, Effect of Model; Driving Point at Station 1.

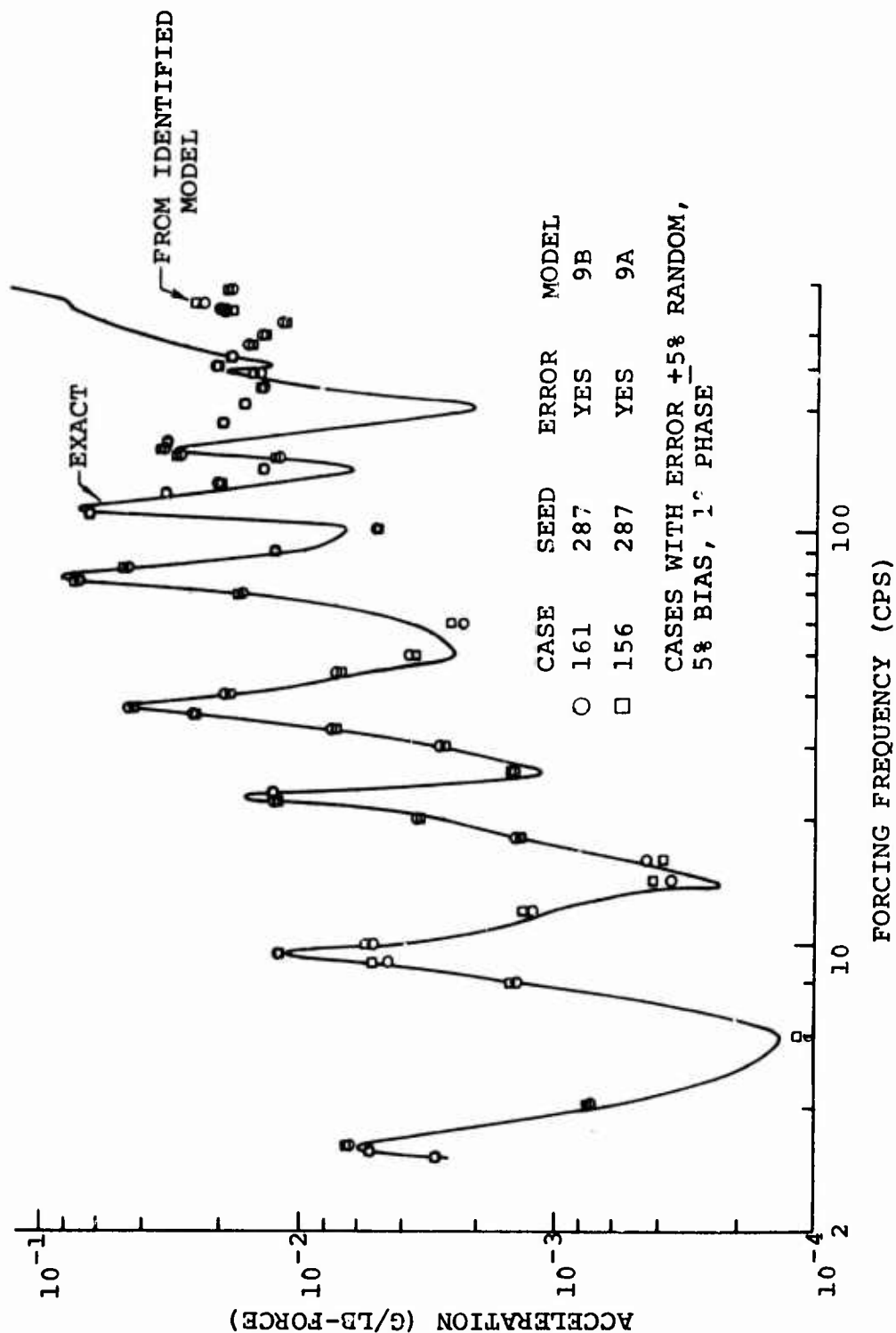


Figure 6. Nine-Point Model Response, Effect of Model; Driving Point at Station 1.

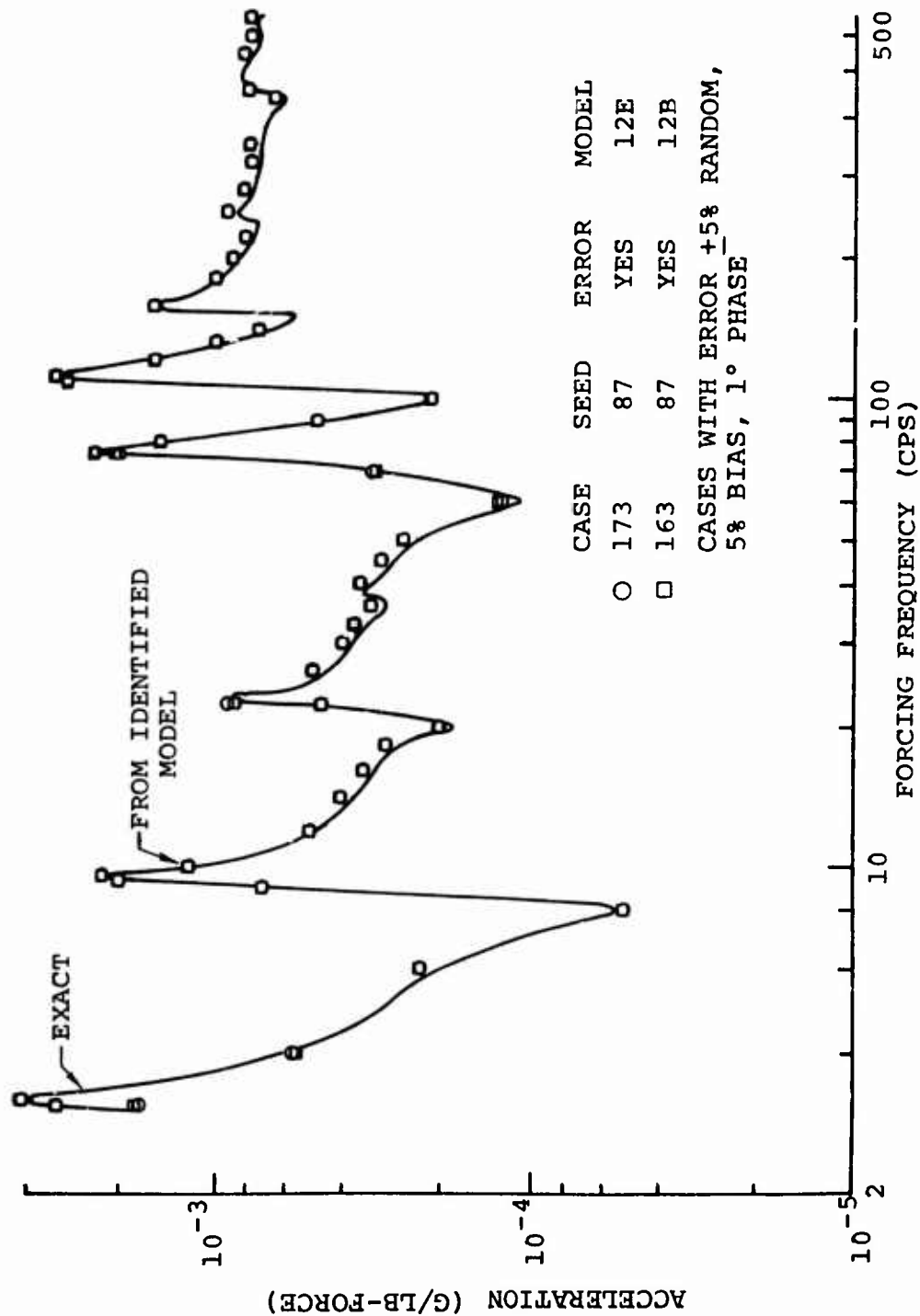


Figure 7. Twelve-Point Model Response, Effect of Model; Driving Point at Station 3.

## CONCLUSIONS

1. The equations of motion for a structure may be obtained using only impedance-type test data without the use of an intuitive mathematical model.
2. The method also yields the eigenvector or mode shape and generalized mass corresponding to each natural frequency.
3. The accuracy of the dynamic response of a structure using impedance-type experimental data is not dependent on the accuracy of the test measurements, provided the data is within the state of the measurement art.
4. The mass matrix assumed for an intuitive mathematical model should be fully populated to yield accurate dynamic response results.
5. To insure minimum information loss in the inversion of mobility matrices, the averaging of mobility test data should be used in practice.
6. There is an upper limit to the size of a physically meaningful reduced complete model yielding minimum loss of information digits. The present report indicates the maximum to be a model of approximately 15 degrees of freedom.

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APPENDIX  
COMPUTER PROGRAM DESCRIPTION

Note: All integer variables must be right justified with no decimal point.

Tape, Card Reader and Printer Assignments.

- 1 Card Reader
- 3 Printer
- 9 Contains influence coefficient matrix for use in XACT.
- 10 Tape assignment in XACT program. Contains mobility data for all degrees of freedom, with no error for specified frequencies for use in INXACT program.
- 11 Tape assignment in INXACT program. Contains mobility data with reduced stations and error (i.e., simulated test data) for use in program IDENTFRE.

All input data must be in the following units.

Mass - lb-sec<sup>2</sup>/in.  
Stiffness - lb/in.  
Frequencies - Hz

PROGRAM XACT

Card	1	Columns	1	IC	Program Control IC $\neq$ 0 End Program IC = 0 Continue Program Case Description
			2-80	HEAD	
Card	2	Columns	1-10 11-20 21-30 31-40	ND G NC NK	Number of Degrees of Freedom ( $\leq 20$ ) Structural Damping Coefficient Number of Modes to be Obtained From Matrix Product [C][M]. If NC = 0, K is not inverted. Number of Modes to be Obtained from Matrix Product $[K]^{-1}[M]$ .
Card(s)	3			M	Mass Matrix. (8E10.0 Format). For full symmetric matrix load lower triangular matrix only starting each row on a new card and ending with the diagonal element. Use as many cards as necessary. For a diagonal mass matrix, load one blank card followed by cards containing diagonal elements in sequence (8E10.0 Format).
Card(s)	4			K	For direct loading of K matrix from cards, proceed as for M matrix as described above.
C Matrix Option				C	To load C matrix from TAPE 9, load one blank card. This will read C matrix from TAPE 9. Unformatted record contains heading (20 words, first character blank); NX (order of matrix). Force deflection influence coefficient matrix.



Card	5	Columns	1-5 6-10	NF IP1	<p>Number of Frequencies Used (&lt; 100)</p> <p>Print Control of Data Written on TAPE 10.</p> <p>IP1 = 0 No Printed Output Except List of Frequencies</p> <p>IP1 = 1 Print Full Mobility Matrix, Real and Imaginary at Each Frequency</p> <p>IP1 = 2 Print Only Diagonal Elements and Row of Mobility Matrix, Real and Imaginary at Each Frequency</p>
		Columns	11-15	IP2	<p>Control on Printed Output</p> <p>IP2 = 0 Same as Written on Tape Above, Complex Velocity Mobility Matrix at Each Frequency</p> <p>IP2 = 1 Print Acceleration Amplitude and Phase Angle</p> <p>This is the row to be printed when IP2 = 2.</p> <p>If NRØW = 0 then only diagonal (driving point) elements are printed as output.</p>
			16-20	NRØW	
Card(s) Omit if NF = 0	6			HZ	<p>Frequencies in Hertz. 10 Columns Per Value, 8 Values Per Card (100 Maximum). Format (8F10.2)</p>
Card	7				<p>Frequency sweep control. This card is the same as Card 5 except that TAPE 10 is not written. To get response data with no tape use a blank card for Card 5 followed by Card 6. To generate TAPE 10 and print no other response data follow Card 5 by one blank card for Card 6. Both options indicated by Card 5 and Card 6 may be used simultaneously.</p>
Card	8	Column			<p>For termination of Case Use 1 in Column 1.</p> <p>Blank card indicates another case to follow, beginning with card 1 again.</p>

# PROGRAM INXACT

Card	1	Column	1	IC HEADN	Program Control Case Description
Card	2	Columns	1-10 11-20 21-30 31-40 71-80	NR PCT PCTB PHE IZ	Number of Points Tested (Number of Degrees of Freedom of the Model) Random Error Applied to Amplitude, Uniform between - and + PCT* Element Amplitude. Bias Error Applied to Amplitude. PCTB* Element Amplitude. Random Error in Degrees Applied to Phase Angle. Uniform Between -PHE and +PHE. Random Number Seed.
Card	3			KEEP	Stations to be used in model. Card 3 is included only if NR < ND (From Program XACT). Five columns per value, maximum of 10 values per card (Format 10I5)
Card	4	Columns	1-5 6-10 11-15 16-20	NFR IP1 IP2 NRØW	Number of Frequencies to be Used (From TAPE 10, XACT Program) IF NFR = 0 all frequencies on TAPE 10 are to be used. Same Definitions as in XACT Program
Card(s)	5			INDX	Indices of Frequencies to be Used from TAPE 10 XACT Program. Indices must be in ascending order. Five columns per value, 16 values per card (Format 16I5).

PROGRAM IDENTRE

Card	1	Columns	1	IC	Program Control IC = 1 Full Program Output IC > 1 Terminate Program Case Description
			2-80	HEAD I	
Card	2	Column	1	NNØR	Control on Normalization of Mobility Matrices
Card(s)	3			INDX	Indices of the Frequencies on TAPE 11 From INXACT Program to be Used in Summation of Real Parts of Mobility Elements (NFR Frequencies. Must be in Ascending Order) Five Columns Per Value, 16 Values Per Card (Format 16I5)
Card(s)	4			INDX	Indices of Frequencies to be Used for PHI Iteration (MODE SHAPE). Same Number of Indices as the Number of Degrees of Freedom of the Model. Indices in Ascending Order.
Card(s)	5			IØM	Indices of Frequencies to be Used in Forming $y_{i(\omega)}$ and $z_{i(\omega)}$ in the Calculation of Generalized Mass and Natural Frequency (2* Number of Degrees of Freedom of the Model). Indices in Ascending Order.

Card	6	Columns	1-5	NF	No. of Frequencies at Which Reidentification of Mobility Matrices is Calculated.
			6-10	IP1	Print Control of Mobility Data IP1 = 0 No printed output except list of frequencies IP1 = 1 Full matrices printed IP1 = 2 Diagonal elements and row printed
			11-15	IP2	IP2 = 0 Complex velocity mobilities printed IP2 = 1 Acceleration mobilities printed Amplitude in g's and phase in degrees
				NROW	This is Row to be Printed when IP1 = 2. If Equal to Zero the Only Diagonal (Driving Point) Elements are Printed
			16-20	NN	Controls Type of Damping Used in Reidentification of Mobilities NN = 0 Use Scalar Structural Damping Coefficient *K Matrix NN = 1 Use Damping Matrix
Card(s)	7			HZ	Frequencies at Which Reidentification is Calculated Ten Columns Per Value, 8 Values Per Card (Format 8F10.0).
Card	8	Column	1		A 2 in Column 1 Terminates Program Otherwise Return to Card 1 for Beginning of New Case

[illegible]

```

K(I,21)=0
DO 250 J=1,ND
250 K(I,21)=K(I,21)+K(I,J)
C
WRITE (3,260) HEAD,ND,G
260 FORMAT ('1',/T5,20(' XACT ',/T5,15('*,5X,A3,19A4,5X,15('*,//
A 110,' DEGREES OF FREEDOM',10X,' STRUCTURAL DAMPING PARAMETER = ',
B F6.3//T50,' MASS MATRIX',/))
CALL MOUT2 (M,ND,ND)
IF (NK.EQ.0.AND.NC.EQ.0) GO TO 290
WRITE (3,270)
270 FORMAT ('1',/T5,15('INFLUENCE COEFFICIENT MATRIX',/))
IF (NK.NE.0) WRITE (3,280) HEAD1
280 FORMAT ('1',/T5,15('FROM TAPE',/T5,15('TAPE HEADING',10X,1H'A3,19A4,1H',/))
CALL MOUT2 (C,ND,ND)
290 WRITE (3,300)
300 FORMAT ('1',/T50,' STIFFNESS MATRIX',/))
CALL MOUT2 (K,ND,ND)
WRITE (3,310) (K(I,21),I=1,ND)
310 FORMAT (/T50,' SPRINGS TO GROUND',/T10,1P10E12.4))
IF (NC.EQ.0) GO TO 350
C
DO 320 I=1,ND
DO 320 J=1,ND
320 B(I,J)=C(I,J)
DO 340 J=1,NC
CALL MHPY (B,M,ND,ND,ND,A)
CALL SITER (A,PHI,FRE,J,ND,ITN,PMAX)
FRE(J)=FRE(J)/6.283185
IT(J)=ITN
DO 330 I=1,ND
LUM(I)=PHI(I,J)
GM(J)=GEN(DUM,M,ND)
CON=PMAX/GM(J)
DO 340 I=1,ND
DO 340 L=1,ND
340 B(I,L)=B(I,L)-DUM(I)*DUM(L)*CON
350 IF (NK.EQ.0) GJ TO 390
C
DO 360 I=1,ND
DO 360 J=1,ND
360 B(I,J)=K(I,J)
CALL INVR5 (M,ND,MU)
CALL MHPY (MU,B,ND,ND,ND,A)
CALL MHPY (A,MU,ND,ND,ND,B)
DO 380 J=1,NK
CALL MHPY (B,M,ND,ND,ND,A)
CALL SITER (A,PHIK,FREK,J,ND,ITN,PMAX)
FREK(J)=1./FREK(IJ)/6.283185
ITK(J)=ITN
DO 370 I=1,ND
DO 370 J=1,ND
370 DUM(I)=PHIK(I,J)
GMK(J)=GEN(DUM,M,ND)
CON=PMAX/GMK(J)
DO 380 I=1,ND

```

```

1XCT 56
2XCT 57
2XCT 58
XCT 59
XCT 60
XCT 61
XCT 62
XCT 63
XCT 64
XCT 65
XCT 66
XCT 67
XCT 68
XCT 69
XCT 70
XCT 71
XCT 72
XCT 73
XCT 74
XCT 75
XCT 76
XCT 77
1XCT 78
2XCT 79
2XCT 80
1XCT 81
1XCT 82
1XCT 83
1XCT 84
1XCT 85
2XCT 86
2XCT 87
1XCT 88
1XCT 89
2XCT 90
3XCT 91
3XCT 92
XCT 93
XCT 94
1XCT 95
2XCT 96
2XCT 97
XCT 98
XCT 99
XCT 100
1XCT 101
1XCT 102
1XCT 103
1XCT 104
1XCT 105
2XCT 106
2XCT 107
1XCT 108
1XCT 109
2XCT 110

```

```

380 B(I,L)=B(I,L)-DUM(I)*DUM(L)*CON
390 IF (NC.EQ.0) GO TO 430
C
WRITE (3,400)
400 FORMAT ('1'//T45,'NORMAL MODES FROM C MATRIX'//)
CALL MOUT2 (PHI,ND,NC)
WRITE (3,410) (FRE(I),I=1,NC)
410 FORMAT ('//T45,'FREQUENCIES - HZ'//T10,10F12.6)
WRITE (3,420) (GM(I),I=1,NC)
420 FORMAT ('//T45,'GENERALIZED MASS'//T10,10F12.6)
430 IF (NKC.EQ.0) GO TO 450
WRITE (3,440)
440 FORMAT ('1'//T45,'NORMAL MODES FROM K MATRIX'//)
CALL MOUT2 (PHIK,ND,NK)
WRITE (3,410) (FREK(I),I=1,NK)
WRITE (3,420) (GMK(I),I=1,NK)
C
READ TAPE CONTROLS
450 TAPE=.TRUE.
460 READ (1,470) NF,IP1,IP2,NROW
470 FORMAT (4I5)
IF (.NOT.TAPE.AND.IP1.EQ.0) GO TO 100
IF (NF.EQ.0) GO TO 690
TORF=NROW-GT.0.AND.NROW.LE.NO
READ (1,130) (HZ(I),I=1,NF)
C
IF (TAPE) WRITE (10) HT,HEAD,NF,ND,(HZ(I),I=1,NF)
DO 570 L=1,NF
CALL MOB (M,K,G,ND,HZ(L),ZR,ZI,YR,YI)
IF (TAPE) WRITE (10) HZ(L),((YR(I,J),YI(I,J),I=1,ND),J=1,ND)
IF (IP1-1) 570,480,550
480 IF (IP2.NE.0) CALL MATAMP (HZ(L),YR,YI,ND)
IF (TAPE) WRITE (3,490)
490 FORMAT ('1'//T10,'COMPLEX MOBILITY WRITTEN ON TAPE'//)
IF (.NOT.TAPE) WRITE (3,540)
IF (IP2.NE.0) GO TO 510
WRITE (3,500) HZ(L)
500 FORMAT ('//T40,'REAL MOBILITY, IMAGINARY MOBILITY FREQ =F10.2,
A = HERTZ'//)
GO TO 530
510 WRITE (3,520) HZ(L)
520 FORMAT ('//T40,'ACCELERATION AMPLITUDE IN G'S, PHASE IN DEG. FREQ
A =F10.2, HERTZ'//)
530 CALL MOUT2 (YR,ND,ND)
WRITE (3,540)
540 FORMAT ('1'//)
CALL MOUT2 (YI,ND,ND)
GO TO 570
550 DO 560 I=1,ND
DPR(I,1)=YR(I,1)
DPI(I,1)=YI(I,1)
IF (.NOT.TORF) GO TO 560
YR(I,1)=YR(NROW,I)
YI(I,1)=YI(NROW,I)
560 TI(L,1)=YI(NROW,I)
570 CONTINUE
3XCT 111
3XCT 112
XCT 113
XCT 114
XCT 115
XCT 116
XCT 117
XCT 118
XCT 119
XCT 120
XCT 121
XCT 122
XCT 123
XCT 124
XCT 125
XCT 126
XCT 127
XCT 128
XCT 129
XCT 130
XCT 131
XCT 132
XCT 133
XCT 134
XCT 135
XCT 136
XCT 137
XCT 138
XCT 139
XCT 140
XCT 141
XCT 142
XCT 143
XCT 144
XCT 145
XCT 146
XCT 147
XCT 148
XCT 149
XCT 150
XCT 151
XCT 152
XCT 153
XCT 154
XCT 155
XCT 156
XCT 157
XCT 158
XCT 159
XCT 160
XCT 161
XCT 162
XCT 163
XCT 164
XCT 165

```

```

IF (IP1-1) 580,690,600
580 WRITE (3,590) (HZ11,I-1,NF)
590 FORMAT (///T10,'MOBILITY MATRICES AT THE FOLLOWING FREQUENCIES (HZ
A) HAVE BEEN WRITTEN ON TAPE'/(T10,10F12.6))
GO TO 690
600 IF (IP2.NE.1) GO TO 620
CALL AMP (HZ,DPR,DPI,NF,ND)
IF (TORF) CALL AMP (HZ,TR,TI,NF,ND)
WRITE (3,610)
610 FORMAT ('1.T40,'DRIVING POINT RESPONSE, AMP IN G'S AND PHASE IN
DEGREES'//)
GO TO 640
620 WRITE (3,630)
630 FORMAT ('1.T40,'DRIVING POINT MOBILITY, REAL AND IMAGINARY'//)
640 CALL YOUT (HZ,DPR,NF,ND,0)
WRITE (3,640)
CALL YOUT (HZ,DPI,NF,ND,IP2)
IF (.NOT.TORF) GO TO 690
IF (IP2.NE.1) GO TO 660
WRITE (3,650) NROW
650 FORMAT ('1.T30,'TRANSFER RESPONSE, ROW '15,' AMP IN G'S AND PHAS
AE IN DEG'//)
GO TO 680
660 WRITE (3,670) NROW
670 FORMAT ('1.T30,'TRANSFER MOBILITY, ROW '15,' REAL AND IMAG'//)
680 CALL YOUT (HZ,TR,NF,ND,0)
WRITE (3,640)
CALL YOUT (HZ,TI,NF,ND,IP2)
690 IF (.NOT.TAPE) GO TO 100
TAPE = .FALSE.
GO TO 460
700 REWIND 10
CALL EXIT
END

```

XCT 166  
 XCT 167  
 XCT 168  
 XCT 169  
 XCT 170  
 XCT 171  
 XCT 172  
 XCT 173  
 XCT 174  
 XCT 175  
 XCT 176  
 XCT 177  
 XCT 178  
 XCT 179  
 XCT 180  
 XCT 181  
 XCT 182  
 XCT 183  
 XCT 184  
 XCT 185  
 XCT 186  
 XCT 187  
 XCT 188  
 XCT 189  
 XCT 190  
 XCT 191  
 XCT 192  
 XCT 193  
 XCT 194  
 XCT 195  
 XCT 196  
 XCT 197  
 XCT 198  
 XCT 199



```

1  SYN
2  SYN
3  SYN
4  SYN
5  1SYN
6  1SYN
7  2SYN
8  2SYN
9  SYN
10 SYN

```

```

C
SUBROUTINE SYN (A,N)
  FORMS SYMMETRIC MATRIX FROM LOWER TRIANGLE
  REAL A(20,21)
  N1=N-1
  DO 100 I=1,N1
    I1=I+1
    DO 100 J=I1,N
      100 A(I,J)=A(J,I)
    RETURN
  END

```

```

SUBROUTINE MOUT2 (A,M,N)
  REAL A(20,21)
  ID=M*NO(N,10)
  WRITE (3,100) (I,I=1,ID)
100 FORMAT (/T5,10I12)
  DO 110 I=1,M
110 WRITE (3,120) I,(A(I,J),J=1,ID)
120 FORMAT (15,5X,1P10E12.4)
  IF (ID=N) 130,150,150
130 WRITE (3,100) (I,I=1,N)
  DO 140 I=1,M
140 WRITE (3,120) I,(A(I,J),J=1,N)
150 RETURN
  END

```

```

MOT 1
MOT 2
MOT 3
MOT 4
MOT 5
MOT 6
MOT 7
MOT 8
MOT 9
MOT 10
MOT 11
MOT 12
MOT 13
MOT 14
MOT 15
MOT 16

```

```

C
C
C
FUNCTION GEN (FUN,A,N)
      GEN = FJNTRANS) * A * FJN
      DIMENSION A(20,21),FUN(20)
      GEN=0
      DO 110 I=1,N
      DUM=0
      DO 100 J=1,N
      100 DUM=DUM+A(I,J)*FUN(J)
      110 GEN=GEN+DUM*FUN(I)
      RETURN
      END

```

```

GEN 1
GEN 2
GEN 3
GEN 4
GEN 5
GEN 6
GEN 7
1GEN 8
1GEN 9
2GEN 10
1GEN 11
GEN 12
GEN 13

```

```

C
SUBROUTINE INVR5 (B,N,A)
A = INVERSE OF B      8 UNDISTURBED
DIMENSION A(20,21),D(20,21),IROW(21),ICOL(21),B(20,21)
DO 100 I=1,N
DO 100 J=1,N
100 A(I,J)=B(I,J)
M=N+1
DO 110 I=1,N
IROW(I)=I
110 ICOL(I)=I
DO 260 K=1,N
AMAX= A(K,K)
DO 130 I=K,N
DO 130 J=K,N
IF(ABS( A(I,J))-ABS(AMAX))136,120,120
120 AMAX= A(I,J)
IC=I
JC=J
130 CONTINUE
KI=ICOL(K)
ICOL(K)=ICOL(IC)
ICOL(IC)=KI
KI=IROW(K)
IROW(K)=IROW(JC)
IROW(JC)=KI
IF(AMAX) 160,140,160
140 WRITE (3,150)
150 FORMAT(' SOLUTION OF EXISTING MATRIX NOT POSSIBLE')
GO TO 330
160 DO 170 J=1,N
E=A(K,J)
A(K,J)=A(IC,J)
170 A(IC,J)=E
DO 180 I=1,N
E=A(I,K)
A(I,K)=A(I,JC)
180 A(I,JC)=E
DO 210 I=1,N
IF(I-K) 200,190,200
190 A(I,M)=1.
GO TO 210
200 A(I,M)=0.
210 CONTINUE
PVT=A(K,K)
DO 220 J=1,M
220 A(K,J)=A(K,J)/PVT
DO 250 I=1,N
IF(I-K)230,250,230
230 AMULT=A(I,K)
DO 240 J=1,M
240 A(I,J)=A(I,J)-AMULT*A(K,J)
250 CONTINUE
260 A(I,K)=A(I,M)
DO 290 I=1,N

```

```

DO 270 L=1,N
  IF(IRO(I))-L)270,280,27C
270 CONTINUE
280 DO 290 J=1,M
290 D(L,J)=A(I,J)
DO 320 J=1,N
DO 300 L=1,N
  IF(ICOL(J)-L) 300,310,300
300 CONTINUE
310 DO 320 I=1,N
320 A(I,L)=D(I,J)
330 RETURN
END

```

```

2INV 56
2INV 57
2INV 58
2INV 59
2INV 60
1INV 61
2INV 62
2INV 63
2INV 64
2INV 65
2INV 66
1INV 67
1INV 68

```

```

C
C
C
C
SUBROUTINE MHPY (A,B,N1,N2,N3,C)
      C = A * B
      A (N1 X N2)  B (N2 X N3)  C (N1 X N3)
      REAL A(20,21),B(20,21),C(20,21)
      DO 100 I=1,N1
      DO 100 J=1,N3
      C(I,J)=0.
      DO 100 K=1,N2
      C(I,J)=C(I,J)+A(I,K)*B(K,J)
      100 C(I,J)=C(I,J)+A(I,K)*B(K,J)
      RETURN
      END
      1
      2
      3
      4
      5
      6
      7
      8
      9
      10
      11
      12
      13
      MMY
      MMY
      MMY
      MMY
      MMY
      1MMY
      2MMY
      3MMY
      MMY
      MMY
      MMY

```



1	MOB
2	MOB
3	MOB
4	MOB
5	MOB
6	MOB
7	MOB
8	MOB
9	MOB
10	MOB
11	MOB
12	MOB
13	MOB
14	MOB
15	MOB
16	MOB
17	MOB
18	MOB
19	MOB
20	MOB
21	MOB
22	MOB
23	MOB
24	MOB
25	MOB
26	MOB



```

SUBROUTINE SITER (A,PHI,FRE,J,ND,ITN,PMAX)
REAL A(20,21),PHI(20,21),FRE(20),DUM(20)
K=ND-J+1
ANK=3.14159*K/(ND-1)
AN=3.14159*J/(ND-1)
DO 100 I=1,ND
  ANG=AN*(I-1)
  ANGK=ANK*(I-1)
  PHI(I,J)=(SIN(ANG)+SIN(ANGK)+1.0)/5.0
  ITN=0
  PMO=100.
110 DO 120 I=1,ND
  DUM(I)=0.
  DO 120 L=1,ND
    DUM(I)=DUM(I)+A(I,L)*PHI(L,J)
    PMAX=0.
    DO 130 I=1,ND
      PMO=AMAX1(PMAX,ABS(DUM(I)))
      DU 140 I=1,ND
140 PHI(I,J)=DUM(I)/PMAX
  IF(ABS(PMAX/PMO-1.0)-.000001) 160,160,150
150 ITN=ITN+1
  PMO=PMAX
  IF(ITN-100) 110,110,160
160 FRE(J)=1.0/SQRT(ABS(PMAX))
  RETURN
END

```

```

SIT 1
SIT 2
SIT 3
SIT 4
SIT 5
SIT 6
SIT 7
SIT 8
SIT 9
SIT 10
SIT 11
SIT 12
SIT 13
SIT 14
SIT 15
SIT 16
SIT 17
SIT 18
SIT 19
SIT 20
SIT 21
SIT 22
SIT 23
SIT 24
SIT 25
SIT 26
SIT 27

```

```

SUBROUTINE YOUT (OMH,A,NINC,ND,NAMP)
REAL OMH(100),A(100,20)
J1=1
ID=MINO(ND,10)
IL=MINO(NINC,50)
100 J1=1
110 WRITE (3,120) (I,I=J1,ID)
120 FORMAT (T5,'HERTZ',16,9I12)
130 WRITE (3,130)
140 FORMAT (1X)
150 IF(NAMP) 140,140,170
160 DO 150 I=1,IL
170 WRITE(3,160) OMH(I),(A(I,J),J=J1,ID)
180 FORMAT (1X,F9.3,1P10E12.4)
190 GO TO 200
200 DO 180 I=1,IL
210 WRITE(3,190) OMH(I),(A(I,J),J=J1,ID)
220 FORMAT (1X,F9.3,10F12.2)
230 IF(IL=NINC) 210,230,230
240 WRITE (3,220)
250 FORMAT ('1.//')
260 J1=51
270 IL=NINC
280 GO TO 110
290 IF(ID=ND) 240,250,250
300 J1=11
310 ID=ND
320 WRITE (3,190)
330 GO TO 100
340 RETURN
350 END

```

```

YOT 1
YOT 2
YOT 3
YOT 4
YOT 5
YOT 6
YOT 7
YOT 8
YOT 9
YOT 10
YOT 11
YOT 12
YOT 13
YOT 14
YOT 15
YOT 16
YOT 17
YOT 18
YOT 19
YOT 20
YOT 21
YOT 22
YOT 23
YOT 24
YOT 25
YOT 26
YOT 27
YOT 28
YOT 29
YOT 30
YOT 31

```

```

C
C
C
C
C
SUBROUTINE MATAMP (OMH,A,B,NR)
      CONVERTS MOBILITY, A + I*08 IN VEL UNITS TO
      AMP (IN A) IN G'S AND PHASE (IN B) IN DEG
      MATRICES ARE AT FREQUENCY OMH IN HERTZ

      DIMENSION A(20,21),B(20,21)
      OM=OMH*0.01626
      DO 210 I=1,NR
      DO 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SQRT(R**2+C**2)*OM
      IF(C) 140,100,160
      100 IF(R) 110,120,130
      110 B(I,J)=270.
      GO TO 210
      120 B(I,J)=0
      GO TO 210
      130 B(I,J)=90.
      GO TO 210
      140 P=ATAN(ABS(R/C))*57.2958
      IF(C) 150,150,180
      150 IF(R) 160,160,170
      160 B(I,J)=180.+P
      GO TO 210
      170 B(I,J)=180.-P
      GO TO 210
      180 IF(R) 190,190,200
      190 B(I,J)=360.-P
      GO TO 210
      200 B(I,J)=P
      210 CONTINUE
      RETURN
      END
MAT 1
MAT 2
MAT 3
MAT 4
MAT 5
MAT 6
MAT 7
MAT 8
MAT 9
MAT 10
MAT 11
MAT 12
MAT 13
MAT 14
MAT 15
MAT 16
MAT 17
MAT 18
MAT 19
MAT 20
MAT 21
MAT 22
MAT 23
MAT 24
MAT 25
MAT 26
MAT 27
MAT 28
MAT 29
MAT 30
MAT 31
MAT 32
MAT 33
MAT 34
MAT 35

```

```

C
C
C
C
C
SUBROUTINE AMP (OMH,A,B,NINC,NR)
      CONVERTS A + I*B IN VELOCITY UNITS TO
      AMP (IN A) IN G'S AND PHASE (IN B) IN DEG
      EACH ROW IS AT A FREQUENCY OMH(I) IN HERTZ

      DIMENSION OMH(100),A(100,20),B(100,20)
      DO 210 I=1,NINC
      OM=OMH(I)*0.01626
      DO 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SQRT(R*R+C*C)*OM
      IF(C) 140,100,140
      100 IF(R) 110,120,130
      110 B(I,J)=270.
      GO TO 210
      120 B(I,J)=0
      GO TO 210
      130 B(I,J)=90.
      GO TO 210
      140 P=ATAN(ABS(R/C))*57.2958
      IF(C) 150,150,180
      150 IF(R) 160,160,170
      160 B(I,J)=180.+P
      GO TO 210
      170 B(I,J)=180.-P
      GO TO 210
      180 IF(R) 190,190,200
      190 B(I,J)=360.-P
      GO TO 210
      200 B(I,J)=P
      210 CONTINUE
      RETURN
      END
AMP 1
AMP 2
AMP 3
AMP 4
AMP 5
AMP 6
AMP 7
AMP 8
1AMP 9
1AMP 10
2AMP 11
2AMP 12
2AMP 13
2AMP 14
2AMP 15
2AMP 16
2AMP 17
2AMP 18
2AMP 19
2AMP 20
2AMP 21
2AMP 22
2AMP 23
2AMP 24
2AMP 25
2AMP 26
2AMP 27
2AMP 28
2AMP 29
2AMP 30
2AMP 31
2AMP 32
2AMP 33
AMP 34
AMP 35

```



CALL MOUT2(VI, NR, NR)	11XT 56
INFR=INFR+1	11XT 57
GO TO 290	11XT 58
260 J=INFR	11XT 59
DO 270 I=1, NR	21XT 60
DPR(J, I)=VR(I, I)	21XT 61
DPI(J, I)=VI(I, I)	21XT 62
IF (.NOT. TORF) GO TO 270	21XT 63
TR(J, I)=VR(NROW, I)	21XT 64
270 TI(J, I)=VI(NROW, I)	21XT 65
280 HZ(INFR)=HZ(I)	11XT 66
INFR=INFR+1	11XT 67
IF (INFR.GT. NFR) GO TO 300	11XT 68
290 CONTINUE	11XT 69
300 IF (IP1.NE.2) GO TO 390	11XT 70
IF (IP2.NE.1) GO TO 320	11XT 71
CALL AMP (HZ, DPR, DPI, NFR, NR)	11XT 72
IF (TORF) CALL AMP (HZ, TR, TI, NFR, NR)	11XT 73
WRITE (3, 310)	11XT 74
310 FORMAT ('1'//T30, 'DRIVING POINT RESPONSE, AMP IN G'S AND PHASE IN	11XT 75
A DEG'//)	11XT 76
GO TO 340	11XT 77
320 WRITE (3, 330)	11XT 78
330 FORMAT ('1'//T30, 'DRIVING POINT MOBILITY, REAL AND IMAGINARY'//)	11XT 79
340 CALL YOUT (HZ, DPR, NFR, 0)	11XT 80
WRITE (3, 250)	11XT 81
CALL YOUT (HZ, DPI, NFR, IP2)	11XT 82
IF (.NOT. TORF) GO TO 390	11XT 83
IF (IP2.NE.1) GO TO 360	11XT 84
WRITE (3, 350) NROW	11XT 85
350 FORMAT ('1'//T30, 'TRANSFER RESPONSE, ROW'15,' , AMP IN G'S AND PH	11XT 86
AASE IN DEG'//)	11XT 87
GO TO 380	11XT 88
360 WRITE (3, 370) NROW	11XT 89
370 FORMAT ('1'//T30, 'TRANSFER MOBILITY, ROW'15,' REAL AND IMAG'//)	11XT 90
380 CALL YOUT (HZ, TR, NFR, 0)	11XT 91
WRITE (3, 250)	11XT 92
CALL YOUT (HZ, TI, NFR, NR, IP2)	11XT 93
390 REWIND 10	11XT 94
REWIND 11	11XT 95
CALL EXIT	11XT 96
END	11XT 97



```

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```

```

C
SUBROUTINE RANDU (IX,IY,YFL)
  THIS SUBROUTINE IS FROM SSP VERS. II
  IY=IX*65539
  IF(IY) 100,110,110
100 IY=IY*2147483647+1
110 YFL=IY
  YFL=YFL*.4656613E-9
  RETURN
  END

```

```


```



```

C
C
C
SUBROUTINE MATAMP (OMH,A,B,NR)
  CONVERTS MOBILITY, A → 100 IN VEL UNITS TO
  AMP (IN A) IN G'S AND PHASE (IN B) IN DEG
  MATRICES ARE AT FREQUENCY OMH IN HERTZ

  DIMENSION A(20,21),B(20,21)
  OM=OMH*0.01626
  DO 210 I=1,NR
    DO 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SQRT(R*R+C*C)*OM
      IF(C) 140,100,140
100 IF(I) 110,120,130
110 B(I,J)=270.
      GO TO 210
120 B(I,J)=0
      GO TO 210
130 B(I,J)=90.
      GO TO 210
140 P=ATAN(ABS(R/C))*57.2958
      IF(C) 150,150,180
150 IF(I) 160,160,170
160 B(I,J)=180.+P
      GO TO 210
170 B(I,J)=180.-P
      GO TO 210
180 IF(I) 190,190,200
190 B(I,J)=360.-P
      GO TO 210
200 B(I,J)=P
210 CONTINUE
      RETURN
      END

```

```

MAT 1
MAT 2
MAT 3
MAT 4
MAT 5
MAT 6
MAT 7
MAT 8
MAT 9
MAT 10
MAT 11
MAT 12
MAT 13
MAT 14
MAT 15
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MAT 17
MAT 18
MAT 19
MAT 20
MAT 21
MAT 22
MAT 23
MAT 24
MAT 25
MAT 26
MAT 27
MAT 28
MAT 29
MAT 30
MAT 31
MAT 32
MAT 33

```

```

C
C
C
SUBROUTINE AMP (OMH,A,B,NINC,NR)
  CONVERTS A + I*B IN VELOCITY UNITS TO
  AMP (IN A) IN G'S AND PHASE (IN B) IN DEG
  EACH ROW IS AT A FREQUENCY OMH(I) IN HERTZ
  DIMENSION OMH(100),A(100,20),B(100,20)
  DO 210 I=1,NINC
    OM=OMH(I)*0.01626
    DO 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SQRT(R**2+C**2)*OM
      IF(C) 140,100,140
      IF(C) 110,120,130
      GO TO 210
      100 B(I,J)=0
      GO TO 210
      110 B(I,J)=90.
      GO TO 210
      120 P=ATAN(ABS(R/C))*5/.2958
      IF(C) 150,150,180
      IF(C) 160,160,170
      GO TO 210
      130 B(I,J)=180.-P
      GO TO 210
      140 IF(R) 190,190,200
      IF(R) 200,200,210
      GO TO 210
      150 B(I,J)=P
      200 B(I,J)=P
      210 CONTINUE
  RETURN
  END
AMP 1
AMP 2
AMP 3
AMP 4
AMP 5
AMP 6
AMP 7
AMP 8
AMP 9
AMP 10
AMP 11
AMP 12
AMP 13
AMP 14
AMP 15
AMP 16
AMP 17
AMP 18
AMP 19
AMP 20
AMP 21
AMP 22
AMP 23
AMP 24
AMP 25
AMP 26
AMP 27
AMP 28
AMP 29
AMP 30
AMP 31
AMP 32
AMP 33

```

```

SUBROUTINE RED (A,B,NO,NR,KEEP )
  INTEGER KEEP (20)
  REAL A(20,21) , B(20,21)
  DO 100 I=1,NR
    DO 100 J=1,NR
      A(I,J) = A(KEEP(I),KEEP(J))
      B(I,J) = B(KEEP(I),KEEP(J))
    RETURN
  END
100

```

```

RED 1
RED 2
RED 3
RED 4
RED 5
RED 6
RED 7
RED 8
RED 9

```

```

SUBROUTINE YOUT (OMH,A,NINC,ND,NAMP)
REAL OMH(100),A(100,20)
J1=1
ID=MINO(MD,10)
IL=MINO(NINC,50)
100 J1=1
110 WRITE (3,120) (1,1-J1,10)
120 FORMAT (15,'HERTZ',16,9(112)
WRITE (3,130)
130 FORMAT (1X)
140 IF(NAMP) 140,140,170
150 DO 150 I=1,IL
160 WRITE(3,160) OMH(I),(A(I,J),J=J1,10)
170 FORMAT (1X,F9.3,1P10E12.4)
GO TO 200
170 DO 180 I=1,IL
180 WRITE(3,190) OMH(I),(A(I,J),J=J1,10)
190 FORMAT (1X,F9.3,10E12.2)
200 IF(IL-NINC) 210,230,230
210 WRITE (3,220)
220 FORMAT ('1'//)
J1=J1+1
IL=NINC
GO TO 110
230 IF(ID-ND) 240,250,250
240 J1=1
ID=ND
WRITE (3,190)
GO TO 100
250 RETURN
END

```

```

YOUT 1
YOUT 2
YOUT 3
YOUT 4
YOUT 5
YOUT 6
YOUT 7
YOUT 8
YOUT 9
YOUT 10
YOUT 11
YOUT 12
YOUT 13
YOUT 14
YOUT 15
YOUT 16
YOUT 17
YOUT 18
YOUT 19
YOUT 20
YOUT 21
YOUT 22
YOUT 23
YOUT 24
YOUT 25
YOUT 26
YOUT 27
YOUT 28
YOUT 29
YOUT 30
YOUT 31

```

```

SUBROUTINE MOUTZ (A,M,N)
  REAL A(20,21)
  ID=MIND(N,10)
  WRITE (3,100) (I,I=1,10)
  100 FORMAT (/Y5,10I12)
  WRITE (3,100)
  DO 110 I=1,M
  110 WRITE (3,120) I,(A(I,J),J=1,10)
  120 FORMAT (15,5X,1P10E12.4)
  IF (10-N) 130,150,150
  130 WRITE (3,100) (I,I=11,N)
  WRITE (3,100)
  DO 140 I=1,M
  140 WRITE (3,120) I,(A(I,J),J=11,N)
  150 RETURN
  END

```

```

NOT 1
NOT 2
NOT 3
NOT 4
NOT 5
NOT 6
NOT 7
NOT 8
NOT 9
NOT 10
NOT 11
NOT 12
NOT 13
NOT 14
NOT 15
NOT 16

```



```

INFR=INFR+1
IF (INFR.GT.NFR) GO TO 320
310 CONTINUE
320 CALL INVR (YRS,NR,YRSIN)
      REMIND 11
      READ (11)
      IF (IC.EQ.0) GO TO 350
      WRITE (3,330)
330 FORMAT ('1'//T30,'SUM OF REAL MODULITIES'//)
      CALL MOUT2 (YRS,NR,NR)
      WRITE (3,340)
340 FORMAT ('1'//T30,'INVERSE OF SUM OF REAL MOD'//)
      CALL MOUT2 (YRSIN,NR,NR)
C      ITERATE FOR PHI (SECOND PASS)
350 READ (1,130) (INDEX(I),I=1,NR)
      WRITE (3,360) (HZ(INDEX(I)),I=1,NR)
360 FORMAT ('//T25,'SECOND PASS FREQUENCIES'//((T10,10F10.2))
      INFR=1
      DO 380 L=1,NFR
        READ (11) FREQ((YR(I),J),YI(I,J),I=1,NR),J=1,NR)
        IF (L.NE.INDEX(INFR)) GO TO 380
        CALL MITER (YR,YRSIN,NR,.0001,99,DUM,VAL,ITN)
        ITP(INFR)=ITN
        CALL MITER (YRSIN,YR,NR,.0001,99,DUM1,VAL,ITN)
        IT(INFR)=ITN
      DO 370 I=1,NR
        GAM1(I,INFR)=SUM(I)
370 PHI(I,INFR)=DUM(I)
      INFR=INFR+1
      IF (INFR.GT.NR) GO TO 390
380 CONTINUE
390 DO 420 I=1,NR
      SUM=0.
      DO 400 J=1,NR
        SUM=SUM+GAM1(J,I)*PHI(J,I)
400 SUM=SUM+GAM1(J,I)*PHI(J,I)
      DO 410 J=1,NR
        GAM1(J,I)=GAM1(J,I)/SUM
420 CONTINUE
      WRITE (3,430)
430 FORMAT ('1'//T40,'ITERATED PHI'//)
      CALL MOUT2 (PHI,NR,NR)
      WRITE (3,440) (ITP(I),I=1,NR)
440 FORMAT ('//T40,'ITERATIONS'//((T5,10I12))
      WRITE (3,450)
450 FORMAT ('1'//T40,'ITERATED GAMMA'//)
      CALL MOUT2 (GAM1,NR,NR)
      WRITE (3,460) (IT(I),I=1,NR)
      EXCD = .FALSE.
      DO 460 I=1,NR
        IF (IT(I).GT.99) EXCD=.TRUE.
460 CONTINUE
      IF (EXCD) WRITE (3,470)
470 FORMAT ('//T10,'*** WARNING - ITERATION NOT CONVERGED ***')
      DO 480 I=1,NR
        DO 480 J=1,NR

```

```

11DN 56
11DN 57
11DN 58
11DN 59
11DN 60
11DN 61
11DN 62
11DN 63
11DN 64
11DN 65
11DN 66
11DN 67
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11DN 69
11DN 70
11DN 71
11DN 72
11DN 73
11DN 74
11DN 75
11DN 76
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11DN 80
21DN 81
21DN 82
21DN 83
11DN 84
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11DN 86
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11DN 88
21DN 89
21DN 90
21DN 91
21DN 92
11DN 93
11DN 94
11DN 95
11DN 96
11DN 97
11DN 98
11DN 99
11DN 100
11DN 101
11DN 102
11DN 103
11DN 104
11DN 105
11DN 106
11DN 107
11DN 108
11DN 109
21DN 110

```

```

480 YI(I,J)=PHI(J,I)
CALL INVR5 (YI,NR,GAMMA)
WRITE (3,490)
490 FORMAT (///T40,'GAMMA = PHI INVERSE TRANSPOSE'//)
CALL MOUT2 (GAMMA,NR,NR)
C READ THIRD PASS FREQ
500 REWIND 11
READ (11)
READ (1,130) (OM(I,1),OM(I,2),I=1,NR)
WRITE (3,510) (HZ(OM(I,1)),HZ(OM(I,2)),I=1,NR)
510 FORMAT ('1'//T25,'THIRD PASS FREQUENCIES'//T10,10F10.2)
C FORM ALL Y STAR
I12=1
INFR=1
DO 550 L=1,NFR
READ (11) FREQ,((YR(I,J),YI(I,J),I=1,NR),J=1,NR)
IF (L.NE.OM(INFR,I12)) GO TO 550
OM(INFR,I12)=FREQ
IF (I12.EQ.2) GO TO 530
DO 520 I=1,NR
GAMMA(I,INFR)=GAM1(I,INFR)
520 DUM(I)=GAMMA(I,INFR)
530 YRSTAR(INFR,I12)=GEN(DUM,YR,NR)
YISTAR(INFR,I12)=GEN(DUM,YI,NR)
IF (I12.EQ.2) GO TO 540
I12=2
GO TO 550
540 I12=1
INFR=INFR+1
IF (INFR.GT.NR) GO TO 560
550 CONTINUE
C FORM Z STAR
560 DO 570 L=1,NR
DO 570 LL=1,2
CON=YRSTAR(L,LL)*2+YISTAR(L,LL)*2
ZRSTAR(L,LL)=YRSTAR(L,LL)/CON
ZISTAR(L,LL)=YISTAR(L,LL)/CON
570 ZISTAR(L,LL)=YISTAR(L,LL)/CON
WRITE (3,580)
580 FORMAT ('1',T40,'YSTAR USING ITERATED GAMMA'//)
WRITE (3,590)
590 FORMAT ('1',T40,'YSTAR (MODE)*T98, *ZSTAR (MODE)*T3,*MODE
A OM 2 REAL (OM 1) IMAG (OM 2) (OM 2) REAL
B (OM 1) (OM 2) IMAG (OM 1) (OM 2) (OM 2) (OM 2) (OM 2)
WRITE (3,600) (I,OM(I,1),YRSTAR(I,1),YISTAR(I,1),ZRSTAR(I,1),
A ZISTAR(I,1),OM(I,2),YRSTAR(I,2),YISTAR(I,2),ZRSTAR(I,2),
B ZISTAR(I,2),I=1,NR)
600 FORMAT ('15,OPF10.2,1P4E24.4/T18,OPF10.2,1P4E24.4)
C IDENTIFY GEN MASS, NAT FREQ
DO 610 I=1,NR
GHI(I)=(OM(I,1)*ZISTAR(I,1)-OM(I,2)*ZISTAR(I,2))/(OM(I,1)*2-
A OM(I,2)*2)/6.283185
OMEGA(I)=OM(I,1)*OM(I,2)*(OM(I,2)*ZISTAR(I,1)-OM(I,1)*ZISTAR(I,2))
A / (OM(I,1)*ZISTAR(I,1)-OM(I,2)*ZISTAR(I,2))
GKI(I)=OMEGA(I)*GHI(I)*39.4784
IF (OMEGA(I).GT.0) OMEGA(I)=SQRT(OMEGA(I))

```



```

      G(I)=OM(I,1)*ZRSSTAR(I,1)/IOMEGA(I)*OMEGA(I)*CM(I)*6.283185)
610 CONTINUE
      WRITE (3,620) (I,CM(I),OMEGA(I),I=1,NR)
620 FORMAT ('1'//T40,'GENERALIZED MASSES AND NATURAL FREQUENCIES'//
      A T50,'MODE GEN MASS',5X,'NAT FREQ'/(T30,I3,F10.4,F15.5),
      --- REIDN (NR,CM,OMEGA,PHI,GAM1,GK,G )
      CALL REIDN (NR,CM,OMEGA,PHI )
      REWIND 11
      GO TO 100
      END
C

```

```

IDN 166
IDN 167
IDN 168
IDN 169
IDN 170
IDN 171
IDN 172
IDN 173
IDN 174
IDN 175

```

```

SUBROUTINE MITER (A,B,N,TOL,ITMAX,FUN,VAL,IT)
  ITERATES ON A+B FOR DOMINANT EIGENFUNCTION (FUN)
  AND EIGENVALUE (VAL).
  N IS ORDER
  TOL IS DECIMAL (.01 PERCENT) TOLERANCE ON VAL.
  ITMAX IS MAX NO OF ITERATIONS.
  IT IS NUMBER OF ITERATIONS PERFORMED.
  A,B ARE SQUARE OF ORDER N (DIMENSIONED (20,21) ).
  USES MNPY (A,B,N1,N2,N3,C1)

  REAL A(20,21),B(20,21),C(20,21),DUM(20),FUN(20)
  CALL MNPY (A,B,N,N,N,C)
  VALO=100.
  IT=1
  DO 100 I=1,N
    100 FUN(I)=1.C
    110 CALL MNPY (C,FUN,N,N,1,0,DUM)
    VAL=DUM(I)
    DO 130 I=2,N
      IF(ABS(VAL)-ABS(DUM(I))) 120,130,130
      120 VAL=DUM(I)
      130 CONTINUE
    DO 140 I=1,N
      FUN(I)=DUM(I)/VAL
      IF(ABS(VAL/VALO-1.0)-TOL) 160,160,150
    140 VAL=VAL
    150 IT=IT+1
    IF(I-ITMAX) 110,110,160
  160 RETURN
  END

```

```

SUBROUTINE MOUTZ (A,M,N)
  REAL A(20,21)
  ID=MINO(N,10)
  WRITE (3,100) (I,I=1,ID)
100 FORMAT (/T5,10I12)
  WRITE (3,100)
  DO 110 I=1,M
110 WRITE (3,120) I,(A(I,J),J=1,ID)
120 FORMAT (15,5X,1P10E12.4)
  IF (ID=N) 130,150,150
130 WRITE (3,100) (I,I=11,N)
  WRITE (3,100)
  DO 140 I=1,M
140 WRITE (3,120) I,(A(I,J),J=11,N)
150 RETURN
  END

```

```

MOT 1
MOT 2
MOT 3
MOT 4
MOT 5
MOT 6
MOT 7
MOT 8
MOT 9
MOT 10
MOT 11
MOT 12
MOT 13
MOT 14
MOT 15
MOT 16

```

```

C
C
FUNCTION GEN (FUN,A,N)      GEN = FJN(TRANS) * A * FJN
DIMENSION A(20,21),FUN(20)
GEN=0
DO 110 I=1,N
  SUM=0
  DO 100 J=1,N
    100 SUM=SUM+A(I,J)*FUN(J)
  110 GEN=GEN+SUM*FUN(I)
  RETURN
END

```

```

GEN 1
GEN 2
GEN 3
GEN 4
GEN 5
1GEN 6
1GEN 7
2GEN 8
2GEN 9
1GEN 10
GEN 11
GEN 12

```

```

C      SUBROUTINE INVR5 (B,N,A)
C      A = INVERSE OF B      B UNDISTURBED
C
C      DIMENSION A(20,21),D(20,21),IROW(21),ICOL(21),M(20,21)
C      DO 100 I=1,N
C      DO 100 J=1,N
C      100 A(I,J)=B(I,J)
C      M=N+1
C      DO 110 I=1,N
C      IROW(I)=I
C      110 ICOL(I)=I
C      DO 260 K=1,M
C      AMAX= A(K,K)
C      DO 130 I=K,N
C      DO 130 J=K,N
C      IF(ABS( A(I,J) )-ABS(AMAX))130,120,120
C      120 AMAX= A(I,J)
C      IC=I
C      JC=J
C      130 CONTINUE
C      KI=ICOL(K)
C      ICOL(K)=ICOL(IC)
C      ICOL(IC)=KI
C      KI=IROW(K)
C      IROW(K)=IROW(JC)
C      IROW(JC)=KI
C      IF(AMAX) 160,140,160
C      140 WRITE (3,150)
C      150 FORMAT(' SOLUTION OF EXISTING MATRIX NOT POSSIBLE')
C      GO TO 330
C      160 DO 170 J=1,N
C      E=A(K,J)
C      A(K,J)=A(IC,J)
C      170 A(IC,J)=E
C      DO 180 I=1,N
C      E=A(I,KI)
C      A(I,KI)=A(I,JC)
C      180 A(I,JC)=E
C      DO 210 I=1,N
C      IF(I-K) 200,190,200
C      190 A(I,M)=1.
C      GO TO 210
C      200 A(I,M)=0.
C      210 CONTINUE
C      PVT=A(K,K)
C      DO 220 J=1,M
C      DO 220 J=1,M
C      220 A(K,J)=A(K,J)/PVT
C      DO 250 I=1,N
C      IF(I-K)230,250,230
C      230 AMULT=A(I,K)
C      DO 240 J=1,M
C      240 A(I,J)=A(I,J)-AMULT*A(K,J)
C      250 CONTINUE
C      DO 260 I=1,N
C      260 A(I,K)=A(I,M)

```

```

INV 1
INV 2
INV 3
INV 4
INV 5
INV 6
INV 7
INV 8
INV 9
INV 10
INV 11
INV 12
INV 13
INV 14
INV 15
INV 16
INV 17
INV 18
INV 19
INV 20
INV 21
INV 22
INV 23
INV 24
INV 25
INV 26
INV 27
INV 28
INV 29
INV 30
INV 31
INV 32
INV 33
INV 34
INV 35
INV 36
INV 37
INV 38
INV 39
INV 40
INV 41
INV 42
INV 43
INV 44
INV 45
INV 46
INV 47
INV 48
INV 49
INV 50
INV 51
INV 52
INV 53
INV 54
INV 55

```

```

DO 290 I=1,N
DO 270 L=1,N
IF(IKOW(I)-L)270,280,270
270 CONTINUE
280 DO 290 J=1,N
290 O(L,J)=A(I,J)
DO 320 J=1,N
DO 300 L=1,N
IF(ICOL(J)-L) 300,310,300
300 CONTINUE
310 DO 320 I=1,N
320 A(L,I)=O(I,J)
330 RETURN
END

```

```

1INV 56
2INV 57
2INV 58
2INV 59
2INV 60
2INV 61
1INV 62
2INV 63
2INV 64
2INV 65
2INV 66
2INV 67
1INV 68
INV 69

```

```

C
C
C
C
SUBROUTINE MNPY (A,B,N1,N2,N3,C)
      C = A * B
      A (N1 X N2)   B (N2 X N3)   C (N1 X N3)
      REAL A(20,21),B(20,21),C(20,21)
      DO 100 I=1,N1
      DO 100 J=1,N3
      C(I,J)=0.
      DO 100 K=1,N2
      100 C(I,J)=C(I,J)+A(I,K)*B(K,J)
      RETURN
      END
MPY 1
MPY 2
MPY 3
MPY 4
MPY 5
MPY 6
1MPY 7
2MPY 8
2MPY 9
3MPY 10
3MPY 11
MPY 12
MPY 13

```

```

SUBROUTINE YNRM (YR,NR )
  DIMENSION YR(20,21)
  VAL=YR(1,1)
  DO 110 I=1,NR
    DO 110 J=1,NR
      IF( ABS(VAL)-ABS(YR(I,J))) 100,110,110
    100 VAL=YR(I,J)
    110 CONTINUE
    DO 120 I=1,NR
      DO 120 J=1,NR
        120 YR(I,J)=YR(I,J)/ABS(VAL)
      RETURN
    END
  SUBROUTINE YRRMS ( YR,NR )
    YR NORMALIZATION BY RMS OF YR
    DIMENSION YR(20,21)
    RMS=0.
    DO 130 I=1,NR
      DO 130 J=1,NR
        130 RMS=YR(I,J)*YR(I,J)+RMS
      RMS=SQR(RMS/(NR*NR))
      DO 140 I=1,NR
        DO 140 J=1,NR
          140 YR(I,J)=YR(I,J)/RMS
        RETURN
      END

```

```

1 YRN
2 YRN
3 YRN
4 1YRN
5 2YRN
6 2YRN
7 2YRN
8 2YRN
9 1YRN
10 2YRN
11 2YRN
12 YRN
13 YRN
14 YRN
15 YRN
16 YRN
17 YRN
18 1YRN
19 2YRN
20 2YRN
21 YRN
22 1YRN
23 2YRN
24 2YRN
25 YRN
26 YRN

```





```

C
C
C
C
C
SUBROUTINE CINV (A,B,N,C,D)
      C+I*D = INVERSE OF A+I*B
      I=SQRT(-1)
      B ASSUMED NON SINGULAR
      REAL A(20,21),B(20,21),C(20,21),D(20,21),E(20,21)
      CALL INVRSEB,N,C)
      CALL HMPY(C,A,N,N,N,E)
      CALL HMPY(A,E,N,N,N,C)
      DO 100 I=1,N
      DO 100 J=1,N
      100 C(I,J)=C(I,J)+B(I,J)
      CALL INVRSC,N,D)
      CALL HMPY(E,D,N,N,N,C)
      DO 110 I=1,N
      DO 110 J=1,N
      110 D(I,J)=D(I,J)
      RETURN
      END
CIN 1
CIN 2
CIN 3
CIN 4
CIN 5
CIN 6
CIN 7
CIN 8
CIN 9
CIN 10
ICIN 11
2CIN 12
2CIN 13
CIN 14
CIN 15
ICIN 16
2CIN 17
2CIN 18
CIN 19
CIN 20

```

```

SUBROUTINE YAFREQ (YR,FREQ,NR )
  DIMENSION YR(20,21)
  DO 100 I=1,NR
    DO 100 J=1,NR
      100 YR(I,J)=YR(I,J)*FREQ
  RETURN
END

```

```

YRF 1
YRF 2
1YRF 3
2YRF 4
2YRF 5
YRF 6
YRF 7

```

```

SUBROUTINE YOUT (OMH,A,NINC,NO,NAMP)
REAL OMH(100),A(100,20)
J1=1
ID=MNO(MO,10)
IL=MNO(NINC,50)
100 I1=1
110 WRITE (3,120) (I,I=J1,10)
120 FORMAT (15,'HERTZ',16,9I12)
130 WRITE (3,130)
130 FORMAT (1X)
140 IF(NAMP) 140,140,170
140 DO 150 I=I1,IL
150 WRITE(3,160) OMH(I),(A(I,J),J=J1,10)
160 FORMAT (1X,F9.3,1P10E12.4)
GO TO 200
170 DO 180 I=I1,IL
180 WRITE(3,190) OMH(I),(A(I,J),J=J1,10)
190 FORMAT (1X,F9.3,10F12.2)
200 IF(IL-NINC) 210,230,230
210 WRITE (3,220)
220 FORMAT ('1'//)
I1=51
IL=NINC
GO TO 110
230 IF(ID-MO) 240,250,250
240 J1=11
ID=MO
WRITE (3,190)
GO TO 100
250 RETURN
END

```

```

YOT 1
YOT 2
YOT 3
YOT 4
YOT 5
YOT 6
YOT 7
YOT 8
YOT 9
YOT 10
YOT 11
YOT 12
YOT 13
YOT 14
YOT 15
YOT 16
YOT 17
YOT 18
YOT 19
YOT 20
YOT 21
YOT 22
YOT 23
YOT 24
YOT 25
YOT 26
YOT 27
YOT 28
YOT 29
YOT 30
YOT 31

```

```

C
C
C
C
C
SUBROUTINE NATAMP (OMH,A,B,NR)
      CONVERTS MOBILITY, A * 108 IN VEL UNITS TO
      AMP (IN A ) IN G'S AND PHASE (IN B ) IN DEG
      MATRICES ARE AT FREQUENCY OMH IN HERTZ

      DIMENSION A(20,21),B(20,21)
      OM=OMH*0.01626
      DO 210 I=1,NR
      DO 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SORT(R*R+C*C)*OM
      IF(C) 140,100,140
      100 IF(R) 110,120,130
      110 B(I,J)=270.
      GO TO 210
      120 B(I,J)=0
      GO TO 210
      130 B(I,J)=90.
      GO TO 210
      140 P=ATAN(ABS(R/C))*57.2958
      IF(C) 150,150,180
      150 IF(R) 160,160,170
      160 B(I,J)=180.+P
      GO TO 210
      170 B(I,J)=180.-P
      GO TO 210
      180 IF(R) 190,190,200
      190 B(I,J)=360.-P
      GO TO 210
      200 B(I,J)=P
      210 CONTINUE
      RETURN
      END
MAT 1
MAT 2
MAT 3
MAT 4
MAT 5
MAT 6
MAT 7
MAT 8
MAT 9
MAT 10
MAT 11
MAT 12
MAT 13
MAT 14
MAT 15
MAT 16
MAT 17
MAT 18
MAT 19
MAT 20
MAT 21
MAT 22
MAT 23
MAT 24
MAT 25
MAT 26
MAT 27
MAT 28
MAT 29
MAT 30
MAT 31
MAT 32
MAT 33
MAT 34
MAT 35

```

```

C
C
C
C
C
SUBROUTINE AMP (OMH,A,B,NINC,NR)
      CONVERTS A + I*B IN VELOCITY UNITS TO
      AMP (IN A) IN G*5 AND PHASE (IN B) IN DEG
      EACH ROW IS AT A FREQUENCY OMH(I) IN HERTZ

      DIMENSION OMH(100),A(100,20),B(100,20)
      DO 210 I=1,NINC
      OM=OMH(I)*0.01626
      DU 210 J=1,NR
      R=A(I,J)
      C=B(I,J)
      A(I,J)=SQRT(R*R+C*C)*OM
      IF(C) 140,100,140
      100 IF(R) 110,120,130
      110 B(I,J)=270.
      GO TO 210
      120 B(I,J)=0
      GO TO 210
      130 B(I,J)=90.
      GO TO 210
      140 P=ATAN(ABS(R/C))*57.2958
      IF(C) 150,150,180
      150 IF(R) 160,160,170
      160 B(I,J)=180.+P
      GO TO 210
      170 B(I,J)=180.-P
      GO TO 210
      180 IF(R) 190,190,200
      190 B(I,J)=360.-P
      GO TO 210
      200 B(I,J)=P
      210 CONTINUE
      RETURN
      END
AMP 1
AMP 2
AMP 3
AMP 4
AMP 5
AMP 6
AMP 7
AMP 8
AMP 9
AMP 10
AMP 11
AMP 12
AMP 13
AMP 14
AMP 15
AMP 16
AMP 17
AMP 18
AMP 19
AMP 20
AMP 21
AMP 22
AMP 23
AMP 24
AMP 25
AMP 26
AMP 27
AMP 28
AMP 29
AMP 30
AMP 31
AMP 32
AMP 33
AMP 34
AMP 35

```

```

C
SUBROUTINE REIDN (NR,GM,OM,PHI,GAM1,GK,G )
IDENTIFICATION OF MASS, STIFFNESS, DAMPING MATRICES
DIMENSION GM(20),OM(20),PHI(20,21),AM(20,21),AK(20,21)
DIMENSION GAM1(20,21),GK(20),AD(20,21),GI(20)
DIMENSION C(20,21),U(20,21),CONA(20),AMG(20,21)
DIMENSION ZR(20,21),ZI(20,21),YR(20,21),YI(20,21),MZ(100)
DIMENSION DPR(100,20),DPI(100,20),YR(100,20),YI(100,20)
LOGICAL TORF
DO 100 I=1,NR
DO 100 J=1,NR
AD(I,J)=0.
AMG(I,J)=0.
U(I,J)=0.
100 C(I,J)=0.
DO 120 I=1,NR
CONA(I)=1./((GM(I)*OM(I)*OM(I)*39.4784)
DO 110 J=1,NR
DO 110 K=1,NR
CALC=PHI(K,I)*PHI(J,I)
CAL=GAM1(K,I)*GAM1(J,I)
AD(K,J)=CAL*G(I)*GK(I)+AD(K,J)
AMG(K,J)=CAL*GM(I)+AMG(K,J)
U(K,J)=CAL*GM(I)+U(K,J)
110 C(K,J)=CAL*CONA(I)+C(K,J)
120 CONTINUE
CALL INVR5 (C,NR,AK)
CALL INVR5 (U,NR,AM)
WRITE (3,130)
CALL MOUT2 (AM,NR,NR)
130 FORMAT ('1',T50,'IDENTIFIED MASS MATRIX'//)
WRITE (3,140)
CALL MOUT2 (AK,NR,NR)
140 FORMAT ('1',T50,'IDENTIFIED STIFFNESS MATRIX'//)
WRITE (3,150)
CALL MOUT2 (AD,NR,NR)
150 FORMAT ('1',T50,'IDENTIFIED DAMPING MATRIX'//)
SUM=0.
WRITE (3,160)
160 FORMAT (' MODE NUMBER',10X,'STRUCTURAL DAMPING'//)
DO 170 I=1,NR
WRITE (3,180) I,G(I)
170 SUM=SUM+G(I)
GS=SUM/NR
180 FORMAT (18,F22.4)
WRITE (3,190) GS
190 FORMAT ('/' ' AVG STRUCTURAL DAMPING='F8.4)
200 READ (1,201) NF,IP1,IP2,NROW,NN
210 FORMAT (5I10)
IF (NF.EQ.0) GO TO 410
TORF=NROW-CT.O.AND.NROW.LE.NR
READ (1,230) (H2(I),I=1,NF)
DO 300 L=1,NF
OMF=H2(L)
CALL MOB2 (AM,AK,GS,NR,OMF,ZR,ZI,YR,YI,AD,NN)
IF (IP1) 220,220,280

```

```

REI 1
REI 2
REI 3
REI 4
REI 5
REI 6
REI 7
REI 8
REI 9
REI 10
REI 11
REI 12
REI 13
REI 14
REI 15
REI 16
REI 17
REI 18
REI 19
REI 20
REI 21
REI 22
REI 23
REI 24
REI 25
REI 26
REI 27
REI 28
REI 29
REI 30
REI 31
REI 32
REI 33
REI 34
REI 35
REI 36
REI 37
REI 38
REI 39
REI 40
REI 41
REI 42
REI 43
REI 44
REI 45
REI 46
REI 47
REI 48
REI 49
REI 50
REI 51
REI 52
REI 53
REI 54
REI 55

```

```

220 IF(IP2.NE.0) CALL MATAMP (HZ(L),YR,YI,NP)
    IF(IP2.NE.0) GO TO 250
    WRITE (3,240) HZ(L)
230 FORMAT (8F10.0)
240 FORMAT ('1.140,'REAL MOBILITY, IMAGINARY MOBILITY   FREQ ='F10.2, IREI 56
    A 'HERTZ'//)
    GO TO 270
    IREI 57
    IREI 58
    IREI 59
    IREI 60
    IREI 61
    IREI 62
    IREI 63
250 WRITE (3,260) HZ(L)
260 FORMAT('1.140,'ACCELERATION AMPLITUDE IN G'S, PHASE IN DEG.   FREQ IREI 64
    A ='F10.2,'HERTZ'//)
    IREI 65
    IREI 66
    IREI 67
270 CALL MOUT2 (YR,NR,NR)
    CALL MOUT2 (YI,NR,NR)
    GO TO 300
    IREI 68
    IREI 69
    IREI 70
    IREI 71
    IREI 72
    IREI 73
    IREI 74
    IREI 75
280 DO 290 I=1,NR
    DPR(L,I)=YR(I,I)
    DPI(L,I)=YI(I,I)
    IF(.NOT.TORF) GO TO 290
    TR(L,I)=YR(NROW,I)
    TI(L,I)=YI(NROW,I)
    290 CONTINUE
    IREI 76
    IREI 77
    IREI 78
    IREI 79
    IREI 80
    IREI 81
    IREI 82
    IREI 83
    IREI 84
    IREI 85
    IREI 86
    IREI 87
    IREI 88
    IREI 89
    IREI 90
    IREI 91
    IREI 92
    IREI 93
    IREI 94
    IREI 95
    IREI 96
    IREI 97
    IREI 98
    IREI 99
    IREI 100
    IREI 101
    IREI 102
300 CONTINUE
    IF(IP1) 410,410,310
310 IF(IP2.NE.1) GO TO 330
    CALL AMP (HZ,DPI,NF,NR)
    IF(TORF) CALL AMP (HZ,TR,TI,NF,NR)
    WRITE (3,320)
    320 FORMAT ('1.140,'DRIVING POINT RESPONSE, AMP IN G'S AND PHASE IN
    ADEGREES'//)
    GO TO 350
    IREI 80
    IREI 81
    IREI 82
    IREI 83
    IREI 84
    IREI 85
    IREI 86
    IREI 87
    IREI 88
    IREI 89
    IREI 90
    IREI 91
    IREI 92
    IREI 93
    IREI 94
    IREI 95
    IREI 96
    IREI 97
    IREI 98
    IREI 99
    IREI 100
    IREI 101
    IREI 102
330 WRITE (3,340)
340 FORMAT ('1.140,'DRIVING POINT MOBILITY, REAL AND IMAGINARY'//)
350 CALL YOUT (HZ,DPI,NF,NR,0)
    WRITE (3,360)
360 FORMAT ('1,'//)
    CALL YOUT (HZ,DPI,NF,NR,IP2)
    IF(.NOT.TORF) GO TO 410
    IF (IP2.NE.1) GO TO 380
    WRITE (3,370) NPON
370 FORMAT ('1.130,'TRANSFER RESPONSE, ROW 'IS,' AMP IN G'S AND PHAS
    A IN DEG'//)
    GO TO 400
    IREI 93
    IREI 94
    IREI 95
    IREI 96
    IREI 97
    IREI 98
    IREI 99
    IREI 100
    IREI 101
    IREI 102
390 WRITE (3,390) NROW
390 FORMAT ('1.130,'TRANSFER MOBILITY, ROW 'IS,' REAL AND IMAG'//)
400 CALL YOUT (HZ,TR,NF,NR,0)
    WRITE (3,360)
    CALL YOUT (HZ,TI,NF,NR, IP2)
410 RETURN
    END

```



## LIST OF FORTRAN SUBROUTINES

AMP	Converts mobility from velocity units to acceleration as amplitude (in g's) and phase angle (in degrees)
CINV	Complex inverse of complex matrix
ERR	Incorporates measurement errors into simulated measurements
GEN	Generalized function of form $f^T A f$ where $f$ is a vector and $A$ is a square matrix
INVR	Inverse of a matrix
ITER	Matrix iteration for eigenvalues and eigenvectors
MITER	More general iteration on product of two matrices; used for gamma iteration
MMPX	Matrix multiplication
MØB	Calculates complex impedance and mobility
MØUT	Special output for square matrix
RANDU	Random number generator
RED	Removes rows and columns from matrix
YØUT	Special matrix output
SYM	Forms symmetric matrix from lower triangle
MØUT2	Special output for nonsquare matrix
MMPY	Matrix multiplication
SITER	Matrix iteration for eigenvalues and eigenvectors
MATAMP	Converts velocity mobility to amplitude (g's) and phase (degrees)
YRNM	Performs normalization of mobility matrix on absolute value of largest element of mobility matrix

YRRMS    Performs normalization of mobility matrix on root  
         mean square value of mobility matrix

MØB2    Calculates complex impedance and mobility

YRFREQ   Multiplies each velocity mobility matrix by  
         its respective frequency to give acceleration  
         mobility

REIDN    Identification of mass stiffness and damping  
         matrices

# SAMPLE OUTPUT

INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT INXACT

INXACT 9 POINT MODEL 20 POINT STRUCTURE 12/11/70

TAPE READING

EXACT DATA SIMULATED TEST  
 INXACT 20 POINT UN2 8/19/70  
 20 DEGREES OF FREEDOM  
 FREQUENCIES (1-2) ON TAPE

3.06	3.40	9.63	10.00	22.32	23.00	37.40	39.00	76.59	78.00
110.52	112.00	152.90	156.00	242.00	245.80	336.28	344.00	453.00	462.34
177.00	180.00	612.00	615.00	798.00	801.00	992.00	995.00	1226.00	1230.00
1456.00	1460.00	1779.00	1783.00	2442.00	2447.00	3561.00	3565.00	5440.00	5445.00

9 POINTS TESTED

MAX RAND ERROR = 0.050. BIAS ERROR = 0.050 OF ELEMENTS. MAX RAND PHASE ERROR = 1.00 DEG. SEED = 206

STATIONS USED

1 3 5 8 11 13 15 18 20

FREQUENCIES USED

3.0600	3.4000	9.6300	10.0000	22.3200	23.0000	37.4000	39.0000	76.5300	78.0000
110.5200	112.0000	152.9000	155.0000	242.0000	245.8000	336.2798	344.0000	544.0000	

IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE IDENTRE

9 PT MODEL 20 PT STRUCTURE 11/13/70

TAPE READING

INEXACT SIMULATED TEST DATA  
 INACT 20 POINT UM2 8/19/70  
 INACT 9 POINT MODEL 20 POINT STRUCTURE 17/11/70  
 ORDER OF MATRICES = 9

FREQUENCIES ON TAPE	22.32	23.70	37.40	76.59	78.00
3.00	10.00	23.70	37.40	39.00	
110.52	156.00	245.80	336.28	344.00	

FIRST PASS FREQUENCIES

3.00	3.40	9.63	10.00	22.32	23.00	37.40	76.59	78.00
110.52	112.00	152.90	156.00	242.00	245.80	336.28	39.00	344.00

SUMMATION OF ACCELERATION MOBILITIES

ACCELERATION MOBILITY FREQ= 5.060HZ									
1	2	3	4	5	6	7	8	9	
1	1.193E-01	1.1885E-01	8.9663E-02	6.9142E-02	4.5834E-02	3.3404E-02	1.9091E-02	-6.2770E-03	-2.4512E-02
2	1.1885E-01	9.3381E-02	7.3176E-02	5.5235E-02	3.6437E-02	2.6138E-02	1.5512E-02	-5.1528E-03	-2.0037E-02
3	8.9663E-02	7.3176E-02	5.7054E-02	4.2072E-02	2.8929E-02	1.9590E-02	1.1578E-02	-3.5627E-03	-1.6062E-02
4	6.9142E-02	5.5235E-02	4.2072E-02	3.1556E-02	2.1548E-02	1.4896E-02	8.7571E-03	-2.7923E-03	-1.1725E-02
5	5.5235E-02	4.2072E-02	3.1556E-02	2.1548E-02	1.4401E-02	1.0729E-02	6.3877E-03	-1.8185E-03	-7.8399E-03
6	4.2072E-02	3.1556E-02	2.1548E-02	1.4401E-02	1.0729E-02	7.6637E-03	4.4155E-03	-1.2225E-03	-5.3266E-03
7	3.1556E-02	2.1548E-02	1.4401E-02	1.0729E-02	7.6637E-03	4.4155E-03	2.6473E-03	-5.6178E-04	-3.1240E-03
8	2.1548E-02	1.4401E-02	1.0729E-02	7.6637E-03	4.4155E-03	2.6473E-03	-5.6178E-04	5.8744E-04	1.4308E-03
9	1.4401E-02	1.0729E-02	7.6637E-03	4.4155E-03	2.6473E-03	-5.6178E-04	5.8744E-04	1.4308E-03	4.9317E-03
	-6.2770E-03	-5.1528E-03	-3.9627E-03	-2.7923E-03	-1.8185E-03	-1.2225E-03	-5.6178E-04	5.8744E-04	1.4308E-03
	-2.4512E-02	-2.0037E-02	-1.6062E-02	-1.1725E-02	-7.8399E-03	-5.3266E-03	-3.1240E-03	1.4308E-03	4.9317E-03

ACCELERATION MOBILITY FREQ = 3.400HZ

	1	2	3	4	5	6	7	8	9
1	4.9517E-02	3.8895E-02	2.9817E-02	2.3722E-02	1.6067E-02	1.1031E-02	6.5267E-03	-2.3836E-03	-9.2107E-03
2	3.8895E-02	3.1853E-02	2.5367E-02	1.8923E-02	1.2607E-02	9.2471E-03	5.1950E-03	-1.8770E-03	-7.3050E-03
3	2.9817E-02	2.5367E-02	1.8712E-02	1.4385E-02	1.0263E-02	6.8905E-03	4.0612E-03	-1.4133E-03	-5.5762E-03
4	2.3722E-02	1.8923E-02	1.4385E-02	1.1277E-02	7.4347E-03	5.2159E-03	3.4579E-03	-9.4974E-04	-4.0151E-03
5	1.6067E-02	1.2607E-02	1.0263E-02	7.4347E-03	5.2159E-03	3.7728E-03	2.3305E-03	-5.1095E-04	-2.5540E-03
6	1.1031E-02	9.2471E-03	6.8905E-03	5.2159E-03	3.7728E-03	2.6990E-03	1.7381E-03	-2.1509E-04	-1.7582E-03
7	6.5267E-03	5.1950E-03	4.0612E-03	3.4579E-03	2.3305E-03	1.7381E-03	1.2282E-03	4.6168E-04	9.5873E-04
8	-2.3836E-03	-1.8770E-03	-1.4133E-03	-9.4974E-04	-5.1095E-04	-2.1509E-04	1.2282E-04	9.5873E-04	2.2743E-03
9	-9.2107E-03	-7.3050E-03	-5.5762E-03	-4.0151E-03	-2.5540E-03	-1.7582E-03	-8.1639E-04	9.5873E-04	2.2743E-03

ACCELERATION VIBILITY		FREQ = 9.630HZ							
1	2	3	4	5	6	7	8	9	
1	5.3285E-01	2.2601E-01	-2.5050E-02	-2.1061E-01	-3.6045E-01	-4.2201E-01	-4.7685E-01	-4.9652E-01	-5.1413E-01
2	2.4601E-01	9.7757E-02	-1.0312E-02	-9.1107E-02	-1.5183E-01	-1.8930E-01	-2.0964E-01	-2.2340E-01	-2.2840E-01
3	-2.5050E-02	-1.0312E-02	1.2001E-03	9.4715E-03	1.6455E-02	2.0479E-02	2.2793E-02	2.3613E-02	2.4217E-02
4	-2.1061E-01	-9.1107E-02	9.4715E-03	8.2921E-02	1.4693E-01	1.7400E-01	1.9261E-01	2.0940E-01	2.1136E-01
5	-3.6045E-01	-1.5183E-01	1.6455E-02	1.4693E-01	2.4386E-01	2.9491E-01	3.3350E-01	3.5820E-01	3.4955E-01
6	-4.2201E-01	-1.8930E-01	2.0479E-02	1.7400E-01	2.9491E-01	3.4664E-01	3.8567E-01	4.1743E-01	4.2575E-01
7	-4.7685E-01	-2.0964E-01	2.2793E-02	1.9261E-01	3.3350E-01	3.8567E-01	4.5184E-01	4.6989E-01	4.5295E-01
8	-4.9652E-01	-2.2340E-01	2.3613E-02	2.0940E-01	3.5820E-01	4.1743E-01	4.6989E-01	5.0327E-01	5.3413E-01
9	-5.1413E-01	-2.2840E-01	2.4217E-02	2.1136E-01	3.4955E-01	4.2575E-01	4.8295E-01	5.3413E-01	5.1896E-01

ACCELERATION VIBILITY FREQ= 10.00MHZ

1	2	3	4	5	6	7	8	9
1	1.2720E-01	5.6919E-02	-5.8938E-03	-5.1901E-02	-8.4441E-02	-1.0464E-01	-1.1353E-01	-1.2500E-01
2	5.6919E-02	2.3756E-02	-2.6440E-03	-2.7296E-02	-3.9799E-02	-4.6932E-02	-5.1327E-02	-5.3489E-02
3	-5.6919E-02	-2.3756E-02	2.6440E-03	2.7296E-02	3.9799E-02	4.6932E-02	5.1327E-02	5.3489E-02
4	-5.1901E-02	-2.2296E-02	2.4210E-03	2.4210E-03	3.6137E-02	4.2779E-02	4.8703E-02	5.1147E-02
5	-8.4441E-02	-3.9799E-02	4.3186E-03	3.6137E-02	5.9799E-02	6.8708E-02	8.3266E-02	9.8020E-02
6	-1.0464E-01	-4.6932E-02	4.9332E-03	4.2779E-02	5.8708E-02	8.5224E-02	1.0570E-01	1.2078E-01
7	-1.1353E-01	-5.1327E-02	5.3029E-03	4.8703E-02	8.3266E-02	9.7038E-02	1.1205E-01	1.1335E-01
8	-1.2500E-01	-5.3489E-02	5.5197E-03	5.1147E-02	8.8020E-02	1.0570E-01	1.1335E-01	1.2078E-01
9	-1.2500E-01	-5.3489E-02	5.7280E-03	5.1756E-02	9.0634E-02	1.0709E-01	1.1389E-01	1.2732E-01



ACCELERATION VIBILITY FREQ= 22.320HZ

	1	2	3	4	5	6	7	8	9
1	9.0943E-01	2.0592E-01	-1.4744E-01	-1.3395E-01	5.0079E-02	2.4630E-01	5.1907E-01	1.3771E 00	2.2491E 00
2	2.0592E-01	4.7550E-02	-3.2826E-02	-2.9808E-02	1.0763E-02	5.7770E-02	1.1476E-01	2.5688E-01	5.4112E-01
3	-1.4744E-01	-3.2826E-02	2.3849E-02	2.1832E-02	-8.4392E-03	-4.0427E-02	-8.5937E-02	-2.2158E-01	-3.7340E-01
4	-1.3395E-01	-2.9808E-02	2.1832E-02	2.0406E-02	-6.8461E-03	-3.8214E-02	-7.6723E-02	-2.0406E-01	-3.3329E-01
5	5.0079E-02	1.0763E-02	-8.4392E-03	-6.8461E-03	3.3310E-03	1.4339E-02	2.9377E-02	7.2997E-02	1.2343E-01
6	2.4630E-01	5.7770E-02	-4.0427E-02	-3.8214E-02	1.4339E-02	7.3883E-02	1.4762E-01	3.7437E-01	6.2337E-01
7	5.1907E-01	1.1476E-01	-2.2158E-01	-2.0406E-01	2.9377E-02	1.4762E-01	3.0351E-01	7.5935E-01	1.3658E 00
8	1.3771E 00	2.5688E-01	-2.2158E-01	-2.0406E-01	7.2997E-02	3.7437E-01	7.5935E-01	2.1286E 00	3.6141E 00
9	2.2491E 00	5.4112E-01	-3.7340E-01	-3.3329E-01	1.2343E-01	6.2337E-01	1.3658E 00	3.6141E 00	6.1281E 00

ACCELERATION MOBILITY FREQUENCY = 23.000HZ

1	2	3	4	5	6	7	8	9
1	6.2170E-01	1.4326E-01	-1.0557E-01	3.4671E-02	1.6881E-01	3.6934E-01	9.1123E-01	1.5577E 00
2	1.4326E-01	3.1780E-02	-2.2937E-02	7.4432E-03	3.7747E-02	8.2834E-02	2.0373E-01	3.5535E-01
3	-1.0557E-01	-2.2937E-02	1.7353E-02	-5.7474E-03	-2.7116E-02	-5.9004E-02	-1.5041E-01	-2.5157E-01
4	-9.2963E-02	-2.0792E-02	1.4192E-02	-4.8172E-03	-2.4314E-02	-5.4437E-02	-1.4155E-01	-2.3698E-01
5	3.4671E-02	7.4432E-03	-5.7474E-03	2.5880E-03	1.0261E-02	2.1118E-02	5.0205E-02	8.5107E-02
6	1.6881E-01	3.7747E-02	-5.9004E-02	1.0261E-02	5.0301E-02	1.0011E-01	2.6034E-01	4.2224E-01
7	3.6934E-01	8.2834E-02	-5.4437E-02	2.1118E-02	1.0011E-01	2.0759E-01	5.2815E-01	9.0001E-01
8	9.1123E-01	2.0373E-01	-1.5041E-01	5.0205E-02	2.6034E-01	5.2815E-01	1.4774E 00	2.4234E 00
9	1.5577E 00	3.5535E-01	-2.5157E-01	8.5107E-02	4.2224E-01	9.0001E-01	2.4934E 00	4.1646E 00

ACCELERATION MOBILITY FREQ= 37.430MHZ

	1	2	3	4	5	6	7	8	9
1	1.209E-00	1.110E-01	-6.110E-01	1.3757E-01	8.5721E-01	1.1216E-00	9.6725E-01	-5.8325E-01	-2.4252E-00
2	1.140E-01	4.1582E-03	-2.2889E-02	4.8512E-03	3.2356E-02	4.0710E-02	3.8694E-02	-2.0334E-02	-1.0730E-01
3	-6.110E-01	-2.2889E-02	1.2907E-01	-2.3536E-02	-1.3340E-01	-2.2588E-01	-2.0615E-01	1.1914E-01	6.0089E-01
4	1.3757E-01	4.8512E-03	-2.8536E-02	6.6101E-03	4.0766E-02	4.9245E-02	4.3704E-02	-2.7312E-02	-1.2901E-01
5	8.5721E-01	3.2356E-02	-1.3340E-01	4.0766E-02	2.6412E-01	3.1705E-01	2.8415E-01	-1.7174E-01	-8.4540E-01
6	1.1216E-00	4.0710E-02	-2.2588E-01	4.9245E-02	3.1705E-01	4.1285E-01	3.5644E-01	-2.0991E-01	-1.0908E-00
7	9.6725E-01	3.8694E-02	-2.0334E-01	4.3704E-02	2.8415E-01	3.5644E-01	3.2823E-01	-1.8334E-01	-9.8655E-01
8	-5.8325E-01	-2.0334E-02	1.1914E-01	-2.7312E-02	-1.7174E-01	-2.0991E-01	-1.8334E-01	1.1840E-01	5.8552E-01
9	-2.4252E-00	-1.0730E-01	6.0089E-01	-1.2901E-01	-8.4540E-01	-1.0908E-00	-9.8655E-01	5.8552E-01	2.8134E-00

ACCELERATION VIBILITY FREQ= 39.000HZ

1	2	3	4	5	6	7	8	9
1	9.4367E-01	3.0306E-02	-1.7166E-01	3.8472E-02	2.4890E-01	2.9813E-01	2.7501E-01	-1.5701E-01 -6.1298E-01
2	3.0306E-02	1.7030E-03	-6.2990E-03	1.1618E-03	8.9874E-03	1.1377E-02	1.0195E-02	-5.0455E-03 -2.7800E-02
3	-1.7166E-01	-6.2990E-03	3.6168E-02	-7.4747E-03	-5.0047E-02	-6.4370E-02	-5.8873E-02	3.2891E-02 1.7087E-01
4	3.8472E-02	1.1618E-03	-7.4747E-03	1.9404E-03	1.0431E-02	1.3824E-02	1.1711E-02	-7.8808E-03 -3.3933E-02
5	2.4890E-01	8.9874E-03	-5.0047E-02	1.0431E-02	7.4925E-02	8.9583E-02	7.7492E-02	-4.6953E-02 -2.4520E-01
6	2.9813E-01	1.1377E-02	-6.4370E-02	1.3824E-02	8.9583E-02	1.1277E-01	9.7422E-02	-5.7365E-02 -2.5629E-01
7	2.7501E-01	1.0195E-02	-5.8873E-02	1.1711E-02	7.7492E-02	9.7422E-02	9.3349E-02	-4.8909E-02 -2.5750E-01
8	-1.5701E-01	-5.0455E-03	3.2891E-02	-7.8808E-03	-4.6953E-02	-5.7365E-02	-4.8909E-02	4.0968E-02 1.6138E-01
9	-6.1298E-01	-2.7800E-02	1.7087E-01	-3.3933E-02	-2.4520E-01	-2.9629E-01	-2.5950E-01	1.6138E-01 3.0850E-01

ACCELERATION MOBILITY FREQ= 76.590HZ

1	2	3	4	5	6	7	8	9
5.0301E 00	-6.9881E-01	1.3672E-01	6.2396E-01	-8.4057E-01	-2.0260E 00	-2.8206E 00	-1.5860E 00	2.4639E 00
-8.9881E-01	1.3930E-01	-2.4500E-02	-1.0183E-01	1.3951E-01	3.4456E-01	4.8781E-01	2.7234E-01	-4.7007E-01
1.3672E-01	-2.4500E-02	4.3212E-03	1.5345E-02	-2.2851E-02	-5.2457E-02	-7.2415E-02	-3.5019E-01	7.8324E-02
6.2396E-01	-1.0183E-01	1.5345E-02	7.3283E-02	-9.5463E-02	-2.2851E-02	-3.4314E-01	-1.5073E-01	3.3964E-01
-8.4057E-01	1.3951E-01	-2.2851E-02	-9.5463E-02	1.3709E-01	3.1234E-01	4.6268E-01	2.4120E-01	-4.4836E-01
-2.0260E 00	3.4456E-01	-5.2457E-02	-2.2851E-01	3.1234E-01	7.6603E-01	1.1098E 00	6.1442E-01	-1.1325E 00
-2.8206E 00	4.8781E-01	-7.2415E-02	-3.4314E-01	4.6268E-01	1.1098E 00	1.6628E 00	8.5519E-01	-1.5710E 00
-1.5860E 00	2.7234E-01	-3.9019E-02	-1.9083E-02	2.6120E-01	6.1442E-01	8.9519E-01	5.0555E-01	-9.0144E-01
2.4639E 00	-4.7007E-01	7.8324E-02	3.3964E-01	-4.4836E-01	-1.1325E 00	-1.5710E 00	-9.0144E-01	1.6658E 00

ACCELERATION, MUSILITY      FREQ= 78.000HZ

1	2	3	4	5	6	7	8	9
4.209E 00	-6.837E-01	1.1061E-01	4.5610E-01	-6.2667E-01	-1.5165E 00	-2.2052E 00	-1.2112E 00	2.2209E 00
-6.837E-01	1.1669E-01	-1.8803E-02	-7.8450E-02	1.0401E-01	2.4990E-01	3.6650E-01	1.5545E-01	-3.6934E-01
1.1061E-01	-1.8803E-02	3.8547E-03	1.2335E-02	-1.6988E-02	-4.0954E-02	-5.5847E-02	-3.1256E-02	5.7637E-C2
4.5610E-01	-7.8450E-02	1.2335E-02	5.7151E-02	-7.4451E-02	-1.8074E-01	-2.6330E-01	-1.4619E-01	2.5737E-01
-6.2667E-01	1.0401E-01	-1.6988E-02	-7.4451E-02	1.0401E-01	2.3613E-01	3.4917E-01	1.5228E-01	-3.5265E-01
-1.5165E 00	2.4990E-01	-4.0954E-02	-1.8074E-01	2.3613E-01	5.9319E-01	8.7111E-01	4.7585E-01	-8.9499E-01
-2.2052E 00	3.6650E-01	-5.5847E-02	-2.6330E-01	3.4917E-01	8.7111E-01	1.2812E 00	7.1406E-01	-1.2712E 00
-1.2112E 00	1.9545E-01	-3.1256E-02	-1.4619E-01	1.9228E-01	4.7586E-01	7.1406E-01	4.6458E-01	-4.8542E-01
2.2209E 00	-3.6934E-01	5.7637E-02	2.5737E-01	-3.5266E-01	-8.8498E-01	-1.2712E 00	-6.8542E-01	1.2928E 00

ACCELERATION MOBILITY FREQ= 110.520HZ

1	2	3	4	5	6	7	8	9
1	4.510E 00	-9.6081E-01	6.8829E-01	-3.6599E-01	-1.9719E-01	1.0550E 00	2.6669E 00	2.5389E 00 -3.3442E 00
2	-9.8061E-01	1.9666E-01	-1.4265E-01	7.8335E-02	4.0030E-02	-2.2732E-01	-5.6650E-01	-6.0944E-01 6.9688E-01
3	6.8829E-01	-1.4265E-01	9.9511E-02	-5.6276E-02	-2.8174E-02	1.6106E-01	3.9934E-01	4.5505E-01 -4.9533E-01
4	-3.6699E-01	7.8335E-02	-5.6276E-02	3.1951E-02	1.4953E-02	-9.0285E-02	-2.3054E-01	-2.4007E-01 2.7008E-01
5	-1.9719E-01	4.0030E-02	-2.8174E-02	1.4953E-02	1.1589E-02	-4.4569E-02	-1.1591E-01	-1.3532E-01 1.4387E-01
6	1.0550E 00	-2.2732E-01	1.6106E-01	-9.0285E-02	-4.4569E-02	2.6858E-01	6.8191E-01	7.1882E-01 -8.2026E-01
7	2.6669E 00	-5.6650E-01	3.9934E-01	-2.3054E-01	-1.1591E-01	6.8191E-01	1.6806E 00	1.8631E 00 -2.0225E 00
8	2.5389E 00	-6.0944E-01	4.5505E-01	-2.4007E-01	-1.3532E-01	7.1882E-01	1.8631E 00	2.0225E 00 -2.1999E 00
9	-3.3442E 00	6.9688E-01	-4.9533E-01	2.7008E-01	1.4387E-01	-8.2026E-01	-2.0225E 00	-2.1999E 00 2.3689E 00

ACCELERATION MOBILITY FREQ= 112.000HZ

1	2	3	4	5	6	7	8	9
3.8025E 00	-8.4398E-01	6.0902E-01	-3.2547E-01	-1.6721E-01	9.5518E-01	2.3507E 00	2.5649E 00	-2.8961E 00
-8.4398E-01	1.7383E-01	-1.2529E-01	6.8903E-02	3.5504E-02	-1.9648E-01	-5.0709E-01	-5.4303E-01	5.3112E-01
6.0902E-01	-1.2529E-01	8.7655E-02	-5.0907E-02	-2.5721E-02	1.4537E-01	3.7258E-01	3.7607E-01	-4.4245E-01
-3.2647E-01	6.8903E-02	-5.0907E-02	2.7255E-02	1.2028E-02	-8.2950E-02	-1.9745E-01	-2.1340E-01	2.3555E-01
-1.6721E-01	3.6506E-02	-2.5721E-02	1.2828E-02	9.9629E-03	-3.8399E-02	-1.0337E-01	-1.1922E-01	1.2682E-01
9.5518E-01	-1.9648E-01	1.4537E-01	-8.2950E-02	-3.8399E-02	2.3070E-01	5.9611E-01	6.1406E-01	-6.7230E-01
2.3507E 00	-5.0709E-01	3.7258E-01	-1.9745E-01	-1.0337E-01	5.9611E-01	1.4258E 00	1.5358E 00	-1.8061E 00
2.5649E 00	-5.4303E-01	3.7607E-01	-2.1340E-01	-1.1922E-01	6.1406E-01	1.5358E 00	1.6782E 00	-1.9021E 00
-2.8961E 00	5.3112E-01	-4.4245E-01	2.3555E-01	1.2682E-01	-6.7230E-01	-1.8061E 00	-1.5021E 00	2.0694E 00



ACCELERATION MOBILITY FREQ = 152.90CHZ

1	2	3	4	5	6	7	5	9
1	1.4437E-00	-3.0527E-01	1.5075E-01	-2.8419E-01	5.9813E-01	5.1921E-01	-3.3623E-01	-1.7748E-00
2	-3.0527E-01	6.6368E-02	-3.1588E-02	6.1162E-02	-1.2141E-01	-1.0571E-01	7.1591E-02	3.8497E-01
3	1.5075E-01	-3.1588E-02	1.4998E-02	-3.0282E-02	5.9512E-01	5.3577E-02	-3.2973E-02	-1.7347E-01
4	-2.8419E-01	6.1162E-02	-3.0282E-02	5.6899E-02	-1.2007E-01	1.0704E-01	7.1943E-02	3.5294E-01
5	5.9813E-01	-1.2141E-01	5.9512E-01	-1.2007E-01	2.4263E-01	2.1379E-01	-1.4700E-01	-7.6655E-01
6	5.1921E-01	-1.0571E-01	5.3577E-01	-1.0704E-01	2.1379E-01	1.9539E-01	-1.1694E-01	-6.4345E-01
7	-3.3623E-01	7.1591E-02	-3.2973E-02	7.1943E-02	-1.4700E-01	-1.1694E-01	1.1246E-01	4.5984E-01
8	-1.7748E-00	3.8497E-01	-1.7347E-01	3.5294E-01	-7.6655E-01	-6.4345E-01	4.5984E-01	-2.3319E-00
9	1.5790E-00	-3.4493E-01	1.6380E-01	-3.2588E-01	6.5721E-01	6.0804E-01	-4.2411E-01	-2.1089E-00
								1.8778E-00

ACCELERATION VIBILITY FREQ= 156.00CHZ

1	2	3	4	5	6	7	8	9
1.7013E 00	-3.5091E-01	1.6937E-01	-3.3707E-01	6.8326E-01	5.8932E-01	-3.9962E-01	-2.0208E 00	1.7927E 00
-3.5091E-01	7.0943E-02	-3.5169E-02	5.9292E-02	-1.4327E-01	-1.2652E-01	8.3085E-02	4.1769E-01	-3.9087E-01
1.6937E-01	-3.5169E-02	1.8707E-02	-3.2317E-02	6.8260E-02	6.2447E-02	-3.8364E-02	-2.0134E-01	1.8996E-01
-3.3707E-01	6.9292E-02	-3.2317E-02	6.3047E-02	-1.3711E-01	-1.1935E-01	8.1059E-02	4.1256E-01	-3.6202E-01
6.8326E-01	-1.4327E-01	6.8260E-02	-1.3711E-01	2.9087E-01	2.3847E-01	-1.7219E-01	-8.5338E-01	7.8982E-01
5.8932E-01	-1.2652E-01	6.2447E-02	-1.1935E-01	2.3847E-01	2.3039E-01	-1.3976E-01	-7.7002E-01	6.8901E-01
-3.9962E-01	8.3085E-02	-3.8364E-02	8.1059E-02	-1.7219E-01	-1.3976E-01	1.2492E-01	5.2734E-01	-4.4563E-01
-2.0208E 00	4.1769E-01	-2.0134E-01	4.1256E-01	-8.5338E-01	-7.7002E-01	5.2734E-01	2.5955E 00	-2.3354E 00
1.7927E 00	-3.9087E-01	1.8996E-01	-3.6202E-01	7.8982E-01	6.8901E-01	-4.8563E-01	-2.3354E 00	2.1013E 00

ACCELERATION, MOBILITY FREQ= 242.070HZ

1	2	3	4	5	6	7	8	9
3.2002E-01	-5.7173E-02	-1.6759E-01	1.4652E-01	-1.5034E-01	-1.0438E-01	-1.1317E-00	1.2022E-00	-1.3744E-00
-5.7173E-02	1.4048E-02	2.3114E-02	-2.0100E-02	1.9645E-02	1.4197E-01	1.5839E-01	-1.6635E-01	1.4595E-01
-1.0759E-01	2.3114E-02	4.7196E-02	-3.9824E-02	4.1048E-02	2.6814E-01	3.1307E-01	-3.1920E-01	2.9409E-01
1.4652E-01	-2.0100E-02	-3.9824E-02	3.4260E-02	-3.4849E-02	-2.3209E-01	-2.6389E-01	2.7035E-01	-2.4421E-01
-1.5034E-01	1.9645E-02	4.1048E-02	-3.4849E-02	3.6079E-02	-2.2676E-01	2.4294E-01	-2.7506E-01	2.5012E-01
-1.0438E-01	1.4197E-01	2.6814E-01	-2.3209E-01	2.2676E-01	1.6179E-00	1.7889E-00	-1.5441E-00	1.7120E-00
-1.1317E-00	1.5839E-01	3.1307E-01	-2.6389E-01	2.4294E-01	1.7889E-00	2.1355E-00	-2.2607E-00	1.9865E-00
1.2022E-00	-1.6635E-01	-3.1920E-01	2.7035E-01	-2.7506E-01	-1.9441E-00	-2.2607E-00	2.3530E-00	-2.1291E-00
-1.0442E-00	1.4595E-01	2.9409E-01	-2.4421E-01	2.5012E-01	1.7120E-00	1.9865E-00	-2.1291E-00	1.8228E-00

ACCELERATION QUALITY FREQ= 245.800HZ

1	2	3	4	5	6	7	8	9
6.2103E-01	-6.0854E-02	-1.2893E-01	1.0655E-01	-1.0526E-01	-7.4786E-01	-8.6382E-01	9.2350E-01	-8.0851E-01
-6.0854E-02	8.3969E-02	1.7448E-02	-1.4507E-02	1.4135E-02	1.0602E-01	1.2296E-01	-1.2431E-01	1.1095E-01
-1.2893E-01	1.7448E-02	3.4205E-02	-3.0219E-02	2.9449E-02	2.0764E-01	2.2620E-01	-2.4572E-01	2.1543E-01
1.0655E-01	-1.4607E-02	-3.0219E-02	2.6249E-02	-2.6494E-02	-1.7506E-01	-1.8882E-01	2.0107E-01	-1.7945E-01
-1.0526E-01	1.4135E-02	2.9449E-02	-2.6494E-02	2.5438E-02	1.6954E-01	1.8604E-01	-2.0139E-01	1.8069E-01
-7.4786E-01	1.0655E-01	1.0802E-01	-1.7506E-01	1.6954E-01	1.1959E-00	1.3707E-00	-1.4702E-00	1.2703E-00
9.2350E-01	-8.0851E-01	1.0802E-01	-1.7506E-01	1.6954E-01	1.1959E-00	1.3707E-00	-1.4702E-00	1.2703E-00
-8.0851E-01	1.0802E-01	2.2620E-01	-1.8882E-01	1.8604E-01	1.3707E-00	1.5105E-00	-1.6423E-00	1.4880E-00
1.0802E-01	-1.4607E-02	-3.0219E-02	2.6249E-02	-2.6494E-02	1.6954E-01	1.8604E-01	-2.0139E-01	1.8069E-01
-1.4607E-02	1.4135E-02	2.9449E-02	-2.6494E-02	2.5438E-02	1.6954E-01	1.8604E-01	-2.0139E-01	1.8069E-01
1.4135E-02	3.4205E-02	1.7448E-02	-1.4507E-02	1.4135E-02	1.0602E-01	1.2296E-01	-1.2431E-01	1.1095E-01
3.4205E-02	1.7448E-02	3.4205E-02	-1.4507E-02	1.4135E-02	1.0602E-01	1.2296E-01	-1.2431E-01	1.1095E-01
-1.4507E-02	1.4135E-02	2.9449E-02	-2.6494E-02	2.5438E-02	1.6954E-01	1.8604E-01	-2.0139E-01	1.8069E-01
1.4135E-02	2.9449E-02	-2.6494E-02	2.5438E-02	1.6954E-01	1.8604E-01	-2.0139E-01	1.8069E-01	1.8069E-01
2.9449E-02	-2.6494E-02	2.5438E-02	1.6954E-01	1.8604E-01	-2.0139E-01	1.8069E-01	1.8069E-01	1.8069E-01
-2.6494E-02	2.5438E-02	1.6954E-01	1.8604E-01	-2.0139E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01
2.5438E-02	1.6954E-01	1.8604E-01	-2.0139E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01
1.6954E-01	1.8604E-01	-2.0139E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01
1.8604E-01	-2.0139E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01
-2.0139E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01
1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01	1.8069E-01

ACCELERATION MOBILITY FREQ= 336.280HZ

	1	2	3	4	5	6	7	8	9
1	1.2718E 00	-2.8444E-02	-2.8397E-01	2.6913E-01	-3.7221E-01	2.2589E-01	1.2460E 00	-5.8919E-01	8.3975E-01
2	-2.8444E-02	1.9030E-03	1.5833E-02	-1.5807E-02	2.2582E-02	-1.2607E-02	-7.4099E-02	3.4405E-02	-4.9299E-02
3	-2.8397E-01	1.5833E-02	1.2582E-01	-1.1532E-01	1.7727E-01	-9.9707E-02	-5.4298E-01	2.6262E-01	-3.6921E-01
4	2.6913E-01	-1.5807E-02	-1.1532E-01	1.1712E-01	-1.7182E-01	9.5037E-02	5.3327E-01	-2.5400E-01	3.7992E-01
5	-3.4221E-01	2.2582E-02	1.7727E-01	-1.7182E-01	2.5973E-01	-1.4389E-01	-8.1980E-01	3.4409E-01	-5.8096E-01
6	2.2589E-01	-1.2607E-02	-9.9707E-02	9.5027E-02	-1.4389E-01	9.7694E-02	4.6736E-01	-2.2702E-01	3.0985E-01
7	1.2460E 00	-7.4099E-02	-5.4298E-01	5.3327E-01	-8.1980E-01	4.6736E-01	2.5672E 00	-1.2598E 00	1.7365E 00
8	-5.8919E-01	3.4405E-02	2.6262E-01	-2.5400E-01	3.9409E-01	-2.2702E-01	-1.2598E 00	6.2600E-01	-8.3557E-01
9	8.3975E-01	-4.9299E-02	-3.6921E-01	3.7992E-01	-5.8096E-01	3.0985E-01	1.7365E 00	-8.3557E-01	1.2573E 00

ACCELERATION MOBILITY FREQ= 344.000HZ

	1	2	3	4	5	6	7	8	9
1	1.4848E 00	-3.4613E-02	-3.1697E-01	3.0634E-01	-4.5154E-01	2.7267E-01	1.4385E 00	-6.8701E-01	7.6742E-01
2	-3.4613E-02	4.8338E-03	1.7505E-02	-1.8028E-02	2.5828E-02	-1.4060E-02	-8.6340E-02	3.8348E-02	-5.7468E-02
3	-3.1697E-01	1.7505E-02	1.3867E-01	-1.3308E-01	2.0320E-01	-1.1225E-01	-6.5877E-01	3.0103E-01	-4.4151E-01
4	3.0634E-01	-1.8028E-02	1.3308E-01	1.3439E-01	-1.9488E-01	1.0824E-01	6.5069E-01	-2.5280E-01	4.4419E-01
5	-4.5154E-01	2.5828E-02	2.0320E-01	-1.9488E-01	3.0409E-01	-1.6370E-01	-9.4750E-01	4.3668E-01	-6.5066E-01
6	2.7267E-01	-1.4060E-02	1.1225E-01	1.0824E-01	-1.6370E-01	1.086E-01	5.0545E-01	-2.5535E-01	3.5617E-01
7	1.4385E 00	-8.6340E-02	6.5877E-01	6.5069E-01	-9.4750E-01	5.0545E-01	3.0855E 00	-1.4322E 00	2.0706E 00
8	-6.8701E-01	3.8348E-02	3.0103E-01	-2.9280E-01	4.3668E-01	-2.5535E-01	-1.4322E 00	7.2430E-01	-9.6030E-01
9	9.6762E-01	-5.7468E-02	-4.4151E-01	4.4419E-01	-6.5066E-01	3.5617E-01	2.0706E 00	-9.6030E-01	1.3548E 00

SUM OF REAL MOBILITIES

1	2	3	4	5	6	7	8	9
3.139E 01	-3.3294E 00	2.0249E-02	3.7338E-01	-8.4168E-01	-3.6424E-01	1.5105E 00	6.4850E-01	1.6468E 00
-3.3294E 00	1.1229E 00	-3.0346E-01	-5.4966E-02	5.4729E-02	1.0650E-01	5.6523E-02	9.0789E-02	3.1609E-01
2.0249E-02	-3.0346E-01	8.5829E-01	-3.9219E-01	2.9734E-01	2.8688E-01	-4.5644E-01	1.8857E-01	-5.9186E-01
3.7338E-01	-5.4966E-02	-3.9219E-01	8.1177E-01	-5.7577E-01	-7.7532E-01	2.7618E-02	-2.4436E-01	3.2975E-01
-8.4168E-01	5.4729E-02	2.9734E-01	-5.7577E-01	2.0948E 00	1.8158E 00	-2.2332E-01	-7.1829E-01	-3.4659E-01
-3.0424E-01	1.0650E-01	2.8688E-01	-7.7532E-01	1.8158E 00	6.3986E 00	8.3252E 00	-2.0027E 00	1.6194E 00
1.5105E 00	5.6523E-02	-4.5644E-01	2.7618E-02	-2.2332E-01	8.3252E 00	1.7087E 01	1.0783E 00	1.3137E 00
6.4850E-01	9.0789E-02	1.8857E-01	-2.4436E-01	-7.1829E-01	-2.0027E 00	1.0783E 00	1.5403E 01	-8.0701E 00
1.6468E 00	3.1609E-01	-5.9186E-01	3.2975E-01	-3.3659E-01	1.6194E 00	1.3137E 00	-8.0701E 00	3.1338E 01

INVERSE OF SUM OF REAL MOB

1	2	3	4	5	6	7	8	9
5.0050E-04	1.6690E-01	6.0446E-02	3.2791E-02	2.7644E-03	1.2553E-02	-8.9994E-03	-2.5677E-03	-4.3368E-03
1.8690E-01	1.6438E 00	8.3188E-01	2.7081E-01	3.1200E-01	-3.8553E-01	1.9853E-01	-6.3128E-02	-1.3536E-02
6.0444E-02	8.3188E-01	2.1800E 00	4.8681E-01	9.2302E-01	-1.2304E 00	6.6695E-01	-1.3721E-01	3.4035E-02
3.2791E-02	2.7081E-01	4.8682E-01	2.5925E 00	-8.6144E-01	1.7123E 00	-8.5079E-01	2.0168E-01	-3.2060E-02
2.7644E-03	3.1200E-01	9.2302E-01	-8.6144E-01	3.4414E 00	-3.7410E 00	1.9084E 00	-3.5140E-01	8.1431E-02
1.2553E-02	-3.8553E-01	-1.2304E 00	1.7123E 00	-3.7410E 00	4.9497E 00	-2.5205E 00	5.0556E-01	-9.6286E-02
6.6695E-01	1.9853E-01	6.6695E-01	-8.5079E-01	1.9084E 00	-2.5205E 00	1.3438E 00	-2.6197E-01	4.6083E-02
-8.5079E-01	-8.5079E-01	2.0168E-01	2.0168E-01	-3.5140E-01	5.0556E-01	-2.6197E-01	1.1173E-01	5.8028E-03
-3.5140E-01	-3.5140E-01	8.1431E-02	-3.2060E-02	8.1431E-02	-9.6286E-02	4.6083E-02	5.8028E-03	3.7941E-02

SECOND PASS FREQUENCIES

5.06	9.63	22.32	37.40	75.59	110.52	152.90	242.00	336.28
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# ITERATED PHI

1	2	3	4	5	6	7	8	9
1.0000E 00	-9.9511E-01	3.7916E-01	1.0000E 00	1.0000E 00	1.0000E 00	-7.6925E-01	5.8849E-01	4.9849E-01
8.0187E-01	-4.3311E-01	8.6950E-02	3.6254E-02	-1.6546E-01	-2.0947E-01	1.6460E-01	-8.0119E-02	-2.8434E-02
6.2392E-01	4.5990E-02	-6.1828E-02	-2.0193E-01	2.6481E-02	1.4995E-01	-7.7471E-02	-1.3341E-01	-2.1495E-01
4.6676E-01	4.0241E-01	-5.6207E-02	4.4657E-02	1.2476E-01	-8.2675E-02	1.5151E-01	1.2067E-01	2.0989E-01
3.1594E-01	6.8590E-01	2.0501E-02	2.8493E-01	-1.6857E-01	-4.3212E-02	-3.1909E-01	-1.2413E-01	-3.2167E-01
2.2012E-01	8.2276E-01	1.0470E-01	3.6712E-01	-3.9323E-01	2.4179E-01	-2.7802E-01	-8.8929E-01	1.7800E-01
1.3013E-01	9.1414E-01	2.2319E-01	3.2192E-01	-5.8562E-01	6.0595E-01	1.9310E-01	-6.5086E-01	1.0000E 00
-4.3010E-02	9.4684E-01	5.8933E-01	-1.9167E-01	-3.2138E-01	6.6418E-01	1.0000E 00	1.0000E 00	-4.8853E-01
-1.7275E-01	1.0000E 00	1.0000E 00	-9.4863E-01	5.4342E-01	-7.3634E-01	-8.5744E-01	-8.7196E-01	6.4380E-01

# ITERATIONS

4	5	7	9	8	6	19	5
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# ITERATED GAMMA

1	2	3	4	5	6	7	8	9
5.4240E-02	-7.2297E-02	2.2378E-01	2.3126E-01	2.5506E-01	1.7703E-01	-9.9581E-02	3.6486E-02	2.8518E-02
4.8192E-01	-9.8314E-01	1.3357E 00	4.7087E-01	-5.9484E-01	-8.1112E-01	7.1786E-01	-1.3483E-01	-2.1428E-01
5.3050E-01	3.1730E-03	-7.8115E-01	-1.3570E 00	3.7819E-01	1.0712E 00	3.3268E-02	-1.6758E-01	-7.2743E-01
3.0227E-01	7.4488E-01	-8.4920E-01	7.1016E-01	2.0865E 00	-6.2510E-01	1.5099E 00	-8.8758E-01	3.1871E-01
2.0060E-01	2.1202E-01	-1.4160E-02	3.9665E-03	-2.0016E 00	-3.7478E-01	-1.6714E 00	1.5814E 00	-1.4864E-01
1.5159E-01	2.6002E-01	3.3167E-01	1.1359E 00	2.0193E 00	4.3968E-02	1.2724E 00	-2.8089E 00	-6.5871E-01
8.0815E-02	-9.1222E-02	-1.3295E-01	-5.1532E-01	-1.3776E 00	2.9727E-01	-5.7806E-01	1.1001E 00	6.6482E-01
-1.0227E-02	1.0366E-01	5.1809E-01	2.6782E-02	1.0607E-01	1.9122E-01	3.8372E-01	-7.4289E-02	-2.2756E-01
-1.1870E-03	9.0643E-02	4.1899E-01	-2.0958E-01	1.0644E-01	-1.4503E-01	-1.4859E-01	-1.0119E-02	6.1432E-02

# ITERATIONS

4	5	4	6	12	7	6	18	7
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# GAMMA = PHI INVERSE TRANSPOSE

1	2	3	4	5	6	7	8	9
4.4771E-02	-7.6608E-02	2.4300E-01	2.3003E-01	2.5964E-01	1.9220E-01	-9.0876E-02	3.4732E-02	1.3339E-02
4.8532E-01	-4.7800E-01	9.6815E-01	3.7568E-01	-6.4113E-01	-7.2242E-01	3.3136E-01	-8.1442E-02	-3.4852E-02
6.0253E-01	2.9031E-02	-9.7707E-01	-1.6855E 00	-4.5391E-01	1.1964E 00	-8.6034E-01	1.5453E-01	-3.1642E-02
2.7103E-01	7.0803E-01	-6.3625E-01	9.2060E-01	2.3581E 00	-6.8761E-01	1.8234E 00	-7.5178E-01	-9.6828E-02
4.3780E-01	3.1369E-01	-5.4040E-01	-8.4248E-01	-3.4520E 00	-2.4200E-01	-3.1878E 00	2.6800E 00	1.1763E 00
-4.4492E-01	1.2515E-01	9.8582E-01	2.0911E 00	3.8741E 00	-1.4160E-01	3.2939E 00	-3.3677E 00	-2.2313E 00
2.4787E-01	-2.4423E-02	-5.0417E-01	-1.0336E 00	-2.3797E 00	3.9980E-01	-1.7018E 00	1.5347E 00	1.5049E 00
-5.8670E-02	9.1982E-02	5.9028E-01	1.6330E-01	3.7812E-01	5.9109E-01	5.9319E-01	-1.8539E-01	-4.1266E-01
9.02540E-03	4.3910E-02	4.0002E-01	-2.1290E-01	3.5203E-02	-1.3499E-01	-1.9435E-01	2.3353E-02	1.0512E-01

THIRD PASS FREQUENCIES	
5.06	3.40
110.52	112.00
	9.63
	152.90
	10.00
	156.00
	22.32
	242.00
	23.00
	245.80
	37.40
	336.28
	39.00
	344.00
	76.55
	78.00

# YSTAR USING ITERATED GAMMA

MODE	UM 1	JM 2	REAL (OM 1)	YSTAR (MODE) (OM 2)		IMAG (OM 1)	(OM 2)	REAL (OM 1)	ZSTAR (MODE) (OM 2)		IMAG (OM 1)	(OM 2)
1	3.00		4.7717E-02	1.4685E-02	6.0094E-02	-4.0989E-02		8.1046E 00	7.7461E 00	-1.0202E 01		
2	9.03	3.40	5.4232E-02	1.2909E-02	-3.5918E-02	-2.9742E-02		1.2817E 01	1.2279E 01	8.4888E 00		2.1621E 01
3	22.32	10.00	2.7338E-01	1.8165E-01	9.5339E-02	-1.5671E-01		3.2613E 00	3.1561E 00	-1.1373E 00		2.8293E 01
4	37.40	23.00	8.2268E-02	2.1644E-02	1.2287E-03	-3.7290E-02		1.2153E 01	1.1696E 01	-1.8151E-01		2.7227E 00
5	76.59	59.00	6.2450E-02	4.6673E-02	9.8975E-03	-2.7702E-02		1.5620E 01	1.5844E 01	-2.4756E 00		1.9966E 01
6	110.22	70.00	4.0707E-02	3.5185E-02	4.7320E-03	-1.4738E-02		2.4238E 01	2.4179E 01	-2.8176E 00		9.4042E 00
7	152.40	112.00	1.5707E-02	1.7655E-02	7.5959E-03	-5.8243E-03		5.1599E 01	5.1082E 01	-2.4954E 01		1.0123E 01
8	242.00	150.00	7.5210E-03	3.8804E-03	1.8826E-03	-8.4479E-04		1.2511E 02	2.4604E 02	-3.1333E 01		1.6852E 01
9	336.28	245.80	7.5837E-03	8.9333E-03	4.1601E-03	-3.2302E-03		1.0136E 02	9.8997E 01	-5.5602E 01		5.3564E 01
		344.00										3.5796E 01

# GENERALIZED MASSES AND NATURAL FREQUENCIES

MODE	GEN MASS	NAT FREQ
1	7.5891	3.16517
2	4.4084	9.47552
3	0.4545	22.51922
4	1.0226	37.41409
5	0.6743	76.89167
6	0.6987	110.84041
7	1.0711	154.74289
8	1.7815	243.19555
9	0.9399	340.95532

IDENTIFIED MASS MATRIX

	1	2	3	4	5	6	7	8	9
1	2.0622E-01	2.8535E-01	-1.1160E-01	1.0187E-01	-3.3755E-01	4.7988E-01	-2.7881E-01	6.1291E-02	-1.1709E-02
2	2.8535E-01	3.9931E 00	2.6458E-01	-3.6572E-01	3.7867E-01	-4.6971E-01	2.4693E-01	-6.1698E-02	3.1051E-03
3	-1.1160E-01	2.6458E-01	8.0997E 00	-3.1569E 00	8.2370E 00	-1.1269E 01	5.2004E 00	-1.3643E 00	3.2976E-01
4	1.0187E-01	-3.6572E-01	-3.1569E 00	1.2585E 01	-1.4237E 01	1.8767E 01	-1.5020E 01	2.1376E 00	-4.8045E-01
5	-3.3755E-01	3.7867E-01	8.2370E 00	-1.4237E 01	3.5797E 01	-4.2128E 01	2.2099E 01	-4.6189E 00	1.0253E 00
6	4.7988E-01	-4.6971E-01	-1.1269E 01	1.8767E 01	-4.2128E 01	5.3115E 01	-2.7970E 01	5.8978E 00	-1.2660E 00
7	-2.7881E-01	2.4693E-01	5.2004E 00	-1.0020E 01	2.2099E 01	-2.7970E 01	1.5096E 01	-3.1643E 00	6.4651E-01
8	6.1291E-02	-6.1698E-02	-1.3643E 00	2.1376E 00	-4.6189E 00	5.8978E 00	-3.1643E 00	9.5834E-01	-9.5173E-02
9	-1.1709E-02	3.1052E-03	3.2976E-01	-4.8045E-01	1.0253E 00	-1.2660E 00	6.4651E-01	-9.5173E-02	2.0270E-01

# IDENTIFIED STIFFNESS MATRIX

1	2	3	4	5	6	7	8	9
4.1727E 04	-1.1068E 05	1.3412E 05	-2.2876E 05	6.0884E 05	-7.9678E 05	4.1736E 05	-8.6880E 04	2.6446E 04
-1.1368E 05	4.0608E 05	-6.2683E 05	8.3343E 05	-1.7693E 06	2.2743E 06	-1.2000E 06	2.5555E 05	-6.1074E 04
1.3413E 05	-6.2682E 05	1.5403E 06	-2.6108E 06	4.5686E 06	-5.2657E 06	2.6983E 06	-5.6418E 05	1.3120E 05
-2.2674E 05	8.3331E 05	-2.6106E 06	7.1041E 06	-1.6450E 07	1.9660E 07	-9.8426E 06	1.5858E 06	-4.4839E 05
6.0977E 05	-1.7689E 06	4.5679E 06	-1.6450E 07	4.8033E 07	-6.1636E 07	3.1518E 07	-6.2949E 06	1.4202E 06
-7.9669E 05	2.2738E 06	-5.2648E 06	1.9659E 07	-6.1636E 07	8.2183E 07	-4.3214E 07	8.7836E 06	-1.9802E 06
4.1731E 05	-1.1998E 06	2.6978E 06	-9.8422E 06	3.1518E 07	-4.3214E 07	2.3485E 07	-5.0115E 06	1.1456E 06
-8.6870E 04	2.5550E 05	-5.6407E 05	1.9857E 06	-6.2949E 06	8.7835E 06	-5.0115E 06	1.2678E 06	-3.2644E 05
2.0444E 04	-6.1061E 04	1.3118E 05	-4.4837E 05	1.4202E 06	-1.9802E 06	1.1455E 06	-3.2644E 05	9.9031E 04

# IDENTIFIED DAMPING MATRIX

	1	2	3	4	5	6	7	8	9
1	2.1150E 03	-8.8975E 03	-2.8250E 03	-7.5527E 03	1.6145E 04	-2.3640E 04	1.2446E 04	-2.5518E 03	7.1254E 02
2	-8.8975E 03	5.3912E 04	2.0394E 04	5.4405E 04	-8.9568E 04	1.3436E 05	-7.7911E 04	2.3160E 04	-6.4418E 03
3	-2.8250E 03	2.0394E 04	1.4470E 05	-3.9253E 04	-5.520E 04	1.8988E 05	-1.3610E 05	4.1913E 04	-1.1143E 04
4	-7.5527E 03	5.4405E 04	-3.9253E 04	2.7207E 05	-4.2286E 05	4.1935E 05	-1.6184E 05	2.0511E 04	-2.4453E 03
5	1.6145E 04	-8.9568E 04	-5.520E 04	-4.2286E 05	9.2250E 05	-1.0986E 06	4.6051E 05	-5.5435E 04	5.8816E 03
6	-2.3640E 04	1.3436E 05	1.8988E 05	4.1935E 05	-1.0986E 06	1.5021E 06	-6.9855E 05	9.5109E 04	-1.2117E 04
7	1.2446E 04	-7.7911E 04	-1.3610E 05	-1.6184E 05	4.6051E 05	-6.9855E 05	3.5796E 05	-5.5163E 04	9.3391E 03
8	-2.5518E 03	2.3160E 04	4.1913E 04	2.0511E 04	-5.5435E 04	9.5109E 04	-5.9163E 04	2.0273E 04	-5.9727E 03
9	7.1254E 02	-6.4418E 03	-1.1143E 04	-2.4453E 03	5.8816E 03	-1.2117E 04	9.3391E 03	-5.9727E 03	2.5683E 03

MODE NUMBER STRUCTURAL DAMPING

1	0.0519
2	0.0496
3	0.0503
4	0.0505
5	0.0478
6	0.0497
7	0.0490
8	0.0457
9	0.0497

AVG STRUCTURAL DAMPING= 0.0493



DRIVING POINT RESPONSE, AMP IN G'S AND PHASE IN DEGREES

HERTZ	1	2	3	4	5	6	7	8	9
3.000	2.4025E-03	1.7516E-03	1.0540E-03	5.9907E-04	2.9621E-04	1.7662E-04	1.0808E-04	1.0734E-04	2.631E-04
3.100	5.2659E-03	3.3468E-03	2.0232E-03	1.1407E-03	5.3777E-04	2.9136E-04	1.4564E-04	1.1686E-04	3.2406E-04
3.200	6.4026E-03	4.1386E-03	2.5072E-03	1.4012E-03	6.2369E-04	2.9125E-04	1.0800E-04	1.1451E-04	2.2927E-04
3.300	7.1206E-04	5.5818E-04	3.5120E-04	1.7673E-04	2.9005E-05	5.0846E-05	1.0899E-04	1.5244E-04	3.1881E-04
4.000	8.9205E-05	2.2788E-04	1.7849E-04	3.8584E-05	1.4314E-04	2.5859E-04	3.5774E-04	5.4075E-04	8.3323E-04
6.000	1.3054E-03	4.8205E-05	1.4574E-04	1.4803E-04	6.3975E-04	9.7385E-04	1.2536E-03	1.6868E-03	2.3594E-03
9.000	4.6816E-03	6.7221E-04	1.3125E-04	6.9225E-04	2.1912E-03	3.2039E-03	4.0119E-03	4.9526E-03	5.8686E-03
9.400	1.1323E-02	2.0998E-03	1.3349E-04	1.8252E-03	5.3654E-03	7.7334E-03	9.5773E-03	1.1259E-02	1.1909E-02
9.500	1.1683E-03	2.2676E-03	1.4274E-04	1.9246E-03	5.5348E-03	7.9636E-03	9.8276E-03	1.1428E-02	1.1700E-02
10.000	4.9459E-03	1.1736E-03	1.4299E-04	8.9302E-04	2.4121E-03	3.4152E-03	4.1512E-03	4.5006E-03	3.7243E-03
12.000	1.1490E-03	5.0421E-04	1.2416E-04	3.2089E-04	7.3705E-04	9.8373E-04	1.1070E-03	6.9013E-04	1.0623E-03
14.000	3.2545E-04	3.9977E-04	1.0998E-04	2.3837E-04	5.0357E-04	6.2700E-04	6.1626E-04	3.0346E-04	2.9557E-03
16.000	4.4265E-04	3.4419E-04	9.2664E-05	1.9994E-04	4.0319E-04	4.5169E-04	3.2072E-04	1.2442E-03	5.4997E-03
18.000	1.4027E-03	2.9109E-04	6.7174E-04	1.6985E-04	3.3997E-04	3.0831E-04	9.3041E-05	2.8255E-03	1.0009E-02
20.000	3.3699E-03	2.0192E-04	2.6386E-05	1.2730E-04	2.8703E-04	1.3197E-04	6.8183E-04	6.6528E-03	2.1062E-02
22.000	1.2279E-02	4.5845E-04	2.6649E-04	1.8998E-04	2.2899E-04	7.6085E-04	3.8706E-03	2.7790E-02	8.1487E-02
22.500	1.6412E-02	8.4462E-04	4.2282E-04	3.6339E-04	2.3720E-04	1.2217E-03	5.5453E-03	3.8726E-02	1.1200E-01
22.500	1.6751E-02	9.2527E-04	4.4852E-04	4.0216E-04	2.4346E-04	1.2997E-03	5.7628E-03	4.0003E-02	1.1530E-01
23.000	1.4600E-02	9.4042E-04	3.9862E-04	4.2602E-04	2.5339E-04	1.1555E-03	4.6794E-03	3.1747E-02	9.0001E-02
26.000	1.2438E-03	4.9015E-04	1.2129E-04	2.3939E-04	1.5005E-04	3.0501E-04	1.1771E-03	8.1654E-03	2.0842E-02
30.000	2.7793E-03	4.0122E-04	2.9547E-05	1.9679E-04	7.2951E-05	1.8298E-04	3.7205E-04	4.6717E-03	9.2729E-03
33.000	7.5268E-03	3.6683E-04	1.9622E-04	1.7539E-04	4.1703E-04	8.0507E-04	3.7773E-04	3.6202E-03	3.8052E-03
36.000	2.5803E-02	3.2818E-04	9.6681E-04	1.4180E-04	1.9071E-03	3.2967E-03	2.4298E-03	2.6601E-03	1.7009E-02
37.000	4.9655E-02	3.3047E-04	1.8106E-04	1.6500E-04	3.6159E-03	6.0741E-03	4.6635E-03	3.0421E-03	3.8698E-02
37.400	5.1394E-02	3.5340E-04	2.0948E-03	2.0481E-04	4.1659E-03	6.9251E-03	5.3682E-03	3.7356E-03	4.7072E-02
38.000	5.1495E-02	3.5997E-04	2.1138E-03	2.1402E-04	4.2044E-03	6.9665E-03	5.4036E-03	3.8983E-03	4.8069E-02
38.000	4.4894E-02	3.7860E-04	1.9056E-03	2.3682E-04	3.7772E-03	6.1741E-03	4.8232E-03	4.2637E-03	4.5539E-02
40.000	1.9233E-02	3.5438E-04	9.2353E-04	2.0636E-04	1.8087E-03	2.8044E-03	2.1688E-03	3.5666E-03	2.5030E-02
45.000	7.3461E-03	3.0825E-04	4.7324E-04	1.7041E-04	9.0052E-04	1.2070E-03	7.1039E-04	2.7243E-03	1.4333E-02
50.000	3.8222E-03	2.6644E-04	3.6708E-04	1.4737E-04	6.7252E-04	7.2404E-04	1.5703E-04	2.2239E-03	1.1308E-02
60.000	1.9723E-03	1.4263E-04	2.8176E-04	8.5181E-05	4.3294E-04	1.5591E-04	1.5244E-03	1.2306E-03	7.9836E-03
70.000	1.5585E-02	2.7912E-04	2.2468E-04	1.3350E-04	1.5579E-04	2.1400E-03	6.1146E-03	9.7195E-04	3.5148E-03
76.000	6.8763E-02	1.9140E-03	1.9424E-04	1.0227E-03	1.8219E-03	1.0475E-02	2.3906E-02	7.1277E-03	1.8795E-02
76.800	7.7944E-02	2.2448E-03	2.1056E-04	1.2136E-03	2.2476E-03	1.2024E-02	2.6763E-02	8.2220E-03	2.3998E-02
76.900	7.8294E-02	2.2630E-03	2.1291E-04	1.2270E-03	2.2760E-03	1.2082E-02	2.6824E-02	8.2588E-03	2.4421E-02
77.000	7.8418E-02	2.2770E-03	2.1519E-04	1.2338E-03	2.3070E-03	1.2133E-02	2.6837E-02	8.2960E-03	2.4797E-02
80.000	4.3808E-02	1.4087E-03	2.1239E-04	7.9682E-04	1.6465E-03	7.1156E-03	1.4338E-02	4.5955E-03	1.8923E-02
90.000	1.1338E-02	4.5017E-04	1.1624E-04	3.2637E-04	8.0054E-04	2.3909E-03	3.0764E-03	8.3866E-04	8.6587E-03
100.000	5.3042E-03	2.1009E-04	1.2610E-04	1.8657E-04	5.8629E-04	1.1263E-03	2.5118E-03	5.4126E-03	3.3502E-03
110.000	6.8940E-02	3.0100E-03	1.5409E-03	4.8431E-04	4.6377E-04	4.0645E-03	2.5434E-02	3.1321E-02	3.7079E-02
110.800	7.0696E-02	3.2420E-03	1.7043E-03	5.7152E-04	4.9252E-04	4.7074E-03	2.7823E-02	3.3407E-02	4.1874E-02
111.900	7.6588E-02	3.3661E-03	1.7159E-03	5.7959E-04	4.9602E-04	4.7653E-03	2.7971E-02	3.3479E-02	4.2255E-02
111.000	7.6972E-02	3.3853E-03	1.7248E-03	5.8697E-04	4.9945E-04	4.8144E-03	2.7971E-02	3.3509E-02	4.2578E-02
120.000	3.2799E-02	1.5195E-03	7.6751E-04	3.5379E-04	3.6342E-04	2.6157E-03	1.1281E-02	1.0136E-02	2.0806E-02
130.000	2.0261E-02	9.8153E-04	5.0949E-04	2.1836E-04	1.4684E-04	1.6539E-03	6.9037E-03	3.3867E-03	1.3238E-02
140.000	1.3692E-02	6.9724E-04	3.9551E-04	1.0743E-04	4.8933E-04	9.6226E-04	5.0937E-03	4.6548E-03	8.0210E-03
150.000	1.2236E-02	5.8931E-04	2.9894E-04	4.8890E-04	2.5326E-03	1.5641E-02	3.7682E-03	2.5197E-02	1.4246E-02
154.000	3.0490E-02	1.4277E-03	4.4810E-04	1.0833E-03	4.7835E-03	3.7606E-03	4.3405E-03	4.7195E-02	3.5425E-02
154.700	3.3768E-02	1.5876E-03	4.9380E-04	1.1614E-03	5.0224E-03	4.0718E-03	4.6136E-03	4.5455E-02	3.8603E-02
154.900	3.4159E-02	1.6070E-03	4.9971E-04	1.1697E-03	5.0440E-03	4.1057E-03	4.6501E-03	4.5651E-02	3.8962E-02

HERTZ	1	2	3	4	5	6	7	8	9
155.000	3.4874E-02	1.6427E-03	5.1096E-04	1.1837E-03	5.0757E-03	4.1653E-03	4.7209E-03	4.5936E-02	3.9604E-02
160.000	3.4462E-02	1.5746E-04	5.3426E-03	9.3189E-04	3.5905E-03	3.3743E-03	4.7566E-03	3.4722E-02	3.1325E-02
130.000	1.9506E-02	1.0083E-04	3.7608E-04	4.4653E-04	1.6862E-03	1.2308E-03	2.7899E-03	1.2969E-02	1.8277E-02
200.000	1.8506E-02	8.8236E-04	3.0332E-04	3.3097E-04	1.0803E-03	4.1713E-04	1.1389E-03	9.2277E-03	1.4443E-02
220.000	1.3932E-02	8.0887E-04	2.1048E-04	2.3225E-04	8.4533E-04	3.0229E-04	2.6854E-03	4.0329E-03	1.0763E-02
240.000	1.3554E-02	7.5421E-04	4.1691E-04	3.5455E-04	7.2441E-04	1.9141E-02	2.9510E-02	2.2740E-02	1.7961E-02
242.000	1.6311E-02	7.9821E-04	5.4052E-04	4.6181E-04	8.4200E-04	2.2846E-02	2.9233E-02	2.8311E-02	2.3641E-02
242.100	1.6538E-02	8.0104E-04	5.4223E-04	4.6888E-04	8.4877E-04	2.2406E-02	2.2740E-02	2.8576E-02	2.3314E-02
242.200	1.6694E-02	8.0391E-04	5.5184E-04	4.7186E-04	8.5535E-04	2.2522E-02	2.2522E-02	2.8815E-02	2.4183E-02
243.000	1.7114E-02	8.2766E-04	5.9241E-04	5.0833E-04	9.0755E-04	2.3253E-02	2.3539E-02	3.0470E-02	2.6163E-02
250.000	2.0650E-02	9.0733E-04	5.8190E-04	5.1413E-04	9.9639E-04	1.7342E-02	1.7342E-02	2.7384E-02	2.7610E-02
260.000	1.8066E-02	8.6101E-04	4.0931E-04	3.6877E-04	8.0997E-04	1.0530E-02	9.3493E-03	1.8717E-02	2.1452E-02
280.000	1.5560E-02	8.1493E-04	2.3609E-04	2.1273E-04	5.2604E-04	5.4322E-03	3.5687E-03	1.3004E-02	1.6632E-02
300.000	1.3900E-02	7.5107E-04	1.0659E-04	9.7989E-05	2.4463E-04	5.1141E-03	2.7847E-03	1.0348E-02	1.3523E-02
320.000	1.1495E-02	7.7046E-04	4.9877E-04	4.4335E-04	1.0388E-03	4.1170E-03	1.2964E-02	7.6147E-03	9.5365E-03
340.000	1.9516E-02	7.7241E-04	2.5427E-03	2.4242E-03	5.6982E-03	4.4965E-03	5.5096E-02	1.6381E-02	2.9328E-02
340.600	2.0334E-02	7.7541E-04	2.5943E-03	2.4726E-03	5.8167E-03	4.6102E-03	5.6020E-02	1.7067E-02	3.0519E-02
340.700	2.0455E-02	7.7591E-04	2.6018E-03	2.4796E-03	5.8339E-03	4.6286E-03	5.6145E-02	1.7176E-02	3.0703E-02
341.000	2.0845E-02	7.7744E-04	2.6216E-03	2.4982E-03	5.8794E-03	4.6842E-03	5.6486E-02	1.7490E-02	3.1244E-02
350.000	2.3452E-02	7.9450E-04	2.1639E-03	2.0548E-03	4.8824E-03	5.1594E-03	4.4594E-02	1.8376E-02	3.2643E-02
330.000	1.8235E-02	7.7407E-04	1.0486E-03	9.8909E-04	2.3929E-03	4.2791E-03	1.9726E-02	1.2466E-02	2.2370E-02
410.000	1.6903E-02	7.6359E-04	8.1770E-04	7.6853E-04	1.8762E-03	3.9401E-03	1.4616E-02	1.1932E-02	1.5851E-02
440.000	1.6246E-02	7.5700E-04	7.2211E-04	5.7710E-04	1.6619E-03	3.7489E-03	1.2485E-02	1.1289E-02	1.3739E-02
455.000	1.6054E-02	7.5445E-04	6.9268E-04	4.4892E-04	1.5958E-03	3.5795E-03	1.1825E-02	1.1054E-02	1.3325E-02
457.900	1.6020E-02	7.5402E-04	6.8801E-04	4.4455E-04	1.5854E-03	3.5680E-03	1.1720E-02	1.1019E-02	1.3324E-02
457.900	1.6019E-02	7.5400E-04	6.8765E-04	4.4429E-04	1.5853E-03	3.5674E-03	1.1716E-02	1.1014E-02	1.3322E-02
460.000	1.5949E-02	7.5369E-04	6.8449E-04	4.4107E-04	1.5775E-03	3.5586E-03	1.1639E-02	1.0906E-02	1.3280E-02
475.000	1.5836E-02	7.5154E-04	6.6348E-04	4.2094E-04	1.5302E-03	3.6043E-03	1.1168E-02	1.0803E-02	1.3013E-02
477.000	1.5816E-02	7.5131E-04	6.6103E-04	4.1859E-04	1.5247E-03	3.5972E-03	1.1111E-02	1.0787E-02	1.2987E-02
430.000	1.5790E-02	7.5093E-04	6.5748E-04	4.1518E-04	1.5167E-03	3.5876E-03	1.1031E-02	1.0756E-02	1.2942E-02
550.000	1.5499E-02	7.4411E-04	6.0331E-04	5.6315E-04	1.3945E-03	3.4230E-03	9.8044E-03	1.0251E-02	1.7230E-02
600.000	1.5145E-02	7.4087E-04	5.8258E-04	5.4319E-04	1.3475E-03	3.3504E-03	9.3318E-03	1.0038E-02	1.6943E-02
612.000	1.5126E-02	7.4022E-04	5.7877E-04	5.3351E-04	1.3386E-03	3.3362E-03	9.2440E-03	9.5973E-03	1.6839E-02
615.000	1.5126E-02	7.4007E-04	5.7786E-04	5.3364E-04	1.3368E-03	3.3329E-03	9.2231E-03	9.5877E-03	1.6876E-02
616.000	1.5117E-02	7.4001E-04	5.7757E-04	5.3336E-04	1.3361E-03	3.3320E-03	9.2171E-03	9.5845E-03	1.6872E-02
650.000	1.5033E-02	7.3843E-04	5.6869E-04	5.2979E-04	1.3159E-03	3.2981E-03	9.0133E-03	9.8477E-03	1.6745E-02

HERTZ	1	2	3	4	5	6	7	8	9
3.000	25.15	25.81	25.97	25.57	23.57	19.86	12.56	4.08	9.86
3.100	49.09	50.38	50.53	50.07	47.73	42.79	29.75	6.43	23.49
3.200	113.30	114.10	114.30	113.88	111.17	104.20	74.24	8.54	54.00
4.000	173.08	174.91	175.23	174.30	157.51	130.41	4.49	3.31	3.77
6.000	33.21	177.00	178.77	169.23	6.46	5.05	4.59	4.18	3.81
8.000	11.32	79.29	179.05	15.55	10.22	9.67	9.35	8.52	7.11
9.000	26.25	35.09	177.31	28.95	26.28	25.82	25.47	24.18	21.31
9.400	74.77	81.23	169.46	77.37	75.22	74.77	74.40	72.79	64.64
9.500	46.64	104.82	169.07	101.27	99.22	98.78	98.38	96.69	92.08
10.000	150.45	161.53	177.74	159.14	157.48	157.06	156.59	154.39	146.37
12.000	171.38	177.04	179.03	176.19	175.24	174.73	173.90	164.92	150.23
14.000	158.21	178.33	178.64	177.93	177.31	176.49	174.57	164.12	146.95
16.000	20.44	178.37	177.50	178.24	177.90	176.30	169.72	10.56	6.62
18.000	11.34	177.21	173.53	177.55	177.92	173.61	80.60	10.03	8.15
20.000	13.55	168.91	126.29	172.21	177.32	147.16	22.36	14.43	13.17
22.000	44.94	80.54	60.47	92.53	170.49	62.47	50.54	48.04	46.97
22.400	75.05	101.06	88.12	107.12	165.56	89.01	80.65	78.71	77.64
22.500	84.96	109.46	97.61	114.68	165.26	98.28	90.60	88.78	87.71
23.000	127.06	146.62	138.23	149.37	170.29	138.11	133.03	131.80	130.73
25.000	146.34	176.75	169.46	177.03	171.10	165.64	169.87	171.74	170.26
30.000	16.72	178.60	78.16	178.44	47.74	34.31	159.59	175.94	171.24
33.000	15.59	178.58	23.74	177.76	22.01	19.56	40.91	176.01	145.02
36.000	34.77	175.61	39.03	167.01	38.50	36.97	40.01	165.48	52.26
37.000	66.57	169.39	70.39	148.63	69.96	68.50	70.07	147.05	78.22
37.400	89.60	168.03	93.29	149.17	92.90	91.44	92.56	147.66	99.84
37.500	95.85	168.28	99.52	150.75	99.13	97.68	98.69	149.24	105.78
38.000	122.90	171.59	126.45	160.84	126.09	124.64	125.17	159.50	131.57
40.000	160.55	177.66	163.88	175.72	163.56	162.06	160.97	174.65	166.46
45.000	171.27	178.33	175.36	178.16	174.95	172.78	165.98	177.32	175.96
50.000	169.72	177.58	177.38	177.73	176.71	172.50	104.12	177.03	177.30
60.000	30.66	168.89	178.19	170.75	175.32	69.17	12.12	171.65	176.61
70.000	19.72	33.16	177.03	170.75	102.89	21.86	17.07	42.99	151.47
76.000	68.32	71.27	163.87	72.84	79.32	68.01	54.55	69.14	86.29
76.800	88.39	92.92	162.10	94.47	99.89	90.10	86.68	90.28	105.01
76.900	91.41	95.90	162.26	97.43	102.76	93.13	89.70	93.16	107.69
77.000	94.38	98.82	162.39	100.37	105.55	96.10	92.62	95.99	110.32
80.000	144.93	152.29	172.53	154.04	157.12	150.87	147.26	146.35	158.71
90.000	164.80	167.05	166.65	173.17	175.12	171.11	160.23	64.42	171.82
100.000	64.80	71.93	47.97	166.93	175.59	160.93	43.87	21.85	130.70
110.000	81.55	82.77	81.77	101.27	159.18	94.63	79.55	74.33	86.80
110.800	97.04	98.20	97.39	113.21	158.67	107.90	95.24	90.17	101.66
110.900	99.06	100.22	99.43	114.85	158.78	109.68	97.28	92.22	103.61
111.000	101.07	102.22	101.45	116.49	158.94	111.46	99.32	94.27	105.55
120.000	107.33	168.16	168.34	170.10	169.20	169.29	166.71	160.09	168.72
130.000	172.91	173.72	174.63	170.15	132.35	171.87	173.41	148.20	172.39
140.000	169.02	170.54	174.50	137.13	36.54	160.36	173.15	43.42	161.22
150.000	108.91	116.84	155.24	62.05	48.36	77.54	164.64	48.26	78.80
154.000	112.00	115.42	139.13	94.05	86.39	99.32	152.01	85.84	101.24
154.700	119.09	121.97	141.73	103.50	96.48	107.87	152.73	95.89	109.88
154.800	120.16	122.99	142.23	104.88	97.94	109.13	152.94	97.35	111.15

HERTZ	1	2	3	4	5	6	7	8	9
155.000	124.54	125.05	143.31	107.64	100.87	111.67	153.44	100.26	113.71
160.000	160.44	161.71	167.35	153.86	149.88	154.25	168.74	149.05	150.95
165.000	170.86	171.44	177.16	175.21	174.45	174.45	172.69	173.09	176.14
170.000	177.84	178.62	176.62	176.03	175.56	77.90	131.77	173.20	176.83
175.000	176.44	178.48	168.89	170.64	175.44	26.27	14.49	150.91	171.66
180.000	144.06	168.51	98.00	105.26	147.46	67.12	96.68	81.74	106.84
185.000	142.71	166.49	107.61	112.74	145.32	83.38	82.53	95.73	114.50
190.000	142.04	166.45	108.24	113.28	145.37	84.29	83.41	96.53	115.04
195.000	142.45	166.41	108.87	113.83	145.44	85.19	84.30	97.34	115.60
200.000	144.50	166.36	114.23	118.57	146.48	92.61	91.56	104.02	120.42
205.000	160.24	174.11	153.56	155.13	164.98	142.89	140.43	149.93	157.56
210.000	175.25	178.07	167.76	168.21	171.96	164.16	159.19	168.19	171.82
215.000	177.72	179.25	166.21	165.73	167.97	173.46	153.97	174.81	176.04
220.000	177.06	179.44	111.06	105.21	117.49	175.58	56.08	174.76	174.54
225.000	170.26	179.23	42.81	41.96	44.04	174.13	35.01	164.60	157.83
230.000	135.32	176.38	93.55	93.30	93.96	155.31	90.58	125.49	118.11
235.000	140.83	176.35	97.44	97.20	97.85	155.65	94.53	127.64	120.78
240.000	137.11	176.34	98.10	97.86	98.50	155.73	95.20	128.02	121.24
245.000	137.97	176.34	100.07	99.83	100.47	155.98	97.20	129.17	122.64
250.000	162.54	176.19	145.16	144.99	145.45	169.28	142.97	159.64	157.20
255.000	177.63	179.66	172.91	172.83	173.05	177.87	171.73	176.67	176.44
260.000	178.98	179.80	176.73	176.68	176.81	178.80	175.95	178.45	178.47
265.000	179.39	179.85	178.04	178.00	178.09	179.15	177.47	179.02	179.09
270.000	179.50	179.87	178.40	178.36	178.44	179.26	177.90	179.18	179.26
275.000	174.51	179.87	178.45	178.42	178.49	179.28	177.96	179.20	179.28
280.000	179.51	179.87	178.45	178.42	178.49	179.28	177.96	179.20	179.28
285.000	179.52	179.87	178.49	178.46	178.53	179.29	178.01	179.22	179.30
290.000	174.59	179.88	178.73	178.70	178.76	179.37	178.30	179.33	179.41
295.000	174.60	179.88	178.76	178.73	178.79	179.38	178.33	179.34	179.42
300.000	179.61	179.89	178.80	178.77	178.82	179.39	178.38	179.36	179.44
305.000	179.77	179.92	179.33	179.31	179.35	179.61	179.07	179.61	179.68
310.000	179.83	179.94	179.51	179.50	179.52	179.69	179.31	179.70	179.76
315.000	179.84	179.94	179.55	179.53	179.55	179.71	179.35	179.72	179.78
320.000	174.84	179.94	179.55	179.54	179.56	179.71	179.36	179.72	179.78
325.000	179.84	179.94	179.55	179.54	179.56	179.71	179.36	179.72	179.78
330.000	179.86	179.95	179.63	179.61	179.63	179.75	179.46	179.76	179.82

# TRANSFER RESPONSE, ROW 5 AMP IN G'S AND PHASE IN DEG

HERTZ	1	2	3	4	5	6	7	8	9
3.000	9.1107E-04	9.5303E-04	5.3022E-04	4.1245E-04	2.9621E-04	2.2312E-04	1.5093E-04	1.7764E-05	1.0919E-04
3.100	1.6012E-03	1.2439E-03	1.0188E-03	7.7551E-04	5.3777E-04	3.9025E-04	2.4817E-04	5.1174E-05	2.5116E-04
3.200	2.0410E-03	1.6311E-03	1.2628E-03	9.3802E-04	6.2369E-04	4.3215E-04	2.5071E-04	1.1501E-04	3.6898E-04
4.000	3.6265E-04	2.6489E-04	1.7574E-05	9.8715E-05	2.9005E-05	1.6165E-05	4.7116E-05	9.8283E-05	1.3112E-04
6.000	3.9358E-04	2.3132E-04	6.4274E-05	3.8821E-05	1.4314E-05	1.9615E-05	2.3361E-04	2.7001E-04	2.8113E-04
8.000	1.0507E-03	5.1617E-04	4.2748E-05	3.3302E-04	6.3975E-04	7.8957E-04	8.8956E-04	9.6713E-04	9.7586E-04
9.000	3.2737E-03	1.4768E-03	8.2376E-05	1.2470E-03	2.1912E-03	2.6483E-03	2.9526E-03	3.1888E-03	3.2219E-03
9.400	7.8021E-03	3.4164E-03	3.3655E-04	3.1333E-03	5.3654E-03	6.4422E-03	7.1643E-03	7.7338E-03	7.8318E-03
9.500	8.0080E-03	3.4811E-03	3.8907E-04	3.2513E-03	5.5348E-03	6.6377E-03	7.3741E-03	7.9587E-03	8.0633E-03
10.000	3.4166E-03	1.4318E-03	2.4099E-04	2.4121E-03	2.5121E-03	2.8728E-03	3.1825E-03	3.4386E-03	3.4992E-03
12.000	1.0204E-03	3.7851E-04	1.4002E-04	4.8349E-04	7.3705E-04	8.5872E-04	9.4456E-04	1.0343E-03	1.0793E-03
14.000	7.9977E-04	2.4518E-04	1.3274E-04	3.5422E-04	5.0357E-04	5.7523E-04	6.3046E-04	7.0813E-04	7.6001E-04
16.000	6.6206E-04	1.9931E-04	1.3774E-04	3.0562E-04	4.0319E-04	4.4970E-04	4.9089E-04	5.6951E-04	6.4260E-04
18.000	7.0037E-04	1.8342E-04	1.5079E-04	2.8363E-04	3.9977E-04	3.6498E-04	3.9334E-04	4.6889E-04	5.5049E-04
20.000	8.0690E-04	1.9016E-04	1.7629E-04	2.7859E-04	2.8703E-04	2.8236E-04	2.8686E-04	3.2655E-04	3.8109E-04
22.000	1.2317E-03	2.6819E-04	2.5066E-04	3.1131E-04	2.2899E-04	2.0623E-04	2.9555E-04	7.2572E-04	1.2274E-03
22.400	1.2548E-03	2.7146E-04	2.4771E-04	2.9441E-04	2.3720E-04	3.2355E-04	5.5466E-04	1.3478E-03	2.4439E-03
22.500	1.1881E-03	2.5650E-04	2.3372E-04	2.7675E-04	2.4346E-04	3.5840E-04	6.1434E-04	1.4302E-03	2.4639E-03
23.000	6.2419E-04	1.2798E-04	1.3878E-04	1.8501E-04	2.5339E-04	3.9774E-04	6.5177E-04	1.5189E-03	2.5185E-03
26.000	7.7262E-04	1.1139E-04	1.9688E-04	1.9641E-04	1.5005E-04	1.7467E-04	2.9295E-04	8.5239E-04	1.5563E-03
30.000	1.4162E-03	1.4792E-04	3.3173E-04	1.7852E-04	7.2951E-05	1.2302E-04	6.9362E-05	8.3850E-04	1.9391E-03
33.000	2.5692E-03	1.9527E-04	5.7178E-04	3.3381E-04	4.1703E-04	5.7816E-04	4.1572E-04	1.0219E-03	3.0023E-03
36.000	7.5529E-03	3.7108E-04	1.5783E-04	1.8713E-04	1.9071E-03	2.4963E-03	2.2179E-03	1.9002E-03	7.6616E-03
37.000	1.3062E-02	5.3131E-04	2.6672E-03	5.3036E-04	3.6159E-03	4.6777E-03	4.0730E-03	2.7512E-03	1.2649E-02
37.400	1.4593E-02	5.4335E-04	2.9520E-03	6.8820E-04	4.1659E-03	5.3682E-03	4.7130E-03	2.8638E-03	1.3904E-02
37.500	1.4600E-02	5.3378E-04	2.9461E-03	7.1724E-04	4.2044E-03	5.4108E-03	4.7608E-03	2.7968E-03	1.3820E-02
39.000	1.2581E-02	4.1155E-04	2.5233E-03	7.2939E-04	3.7772E-03	4.8374E-03	4.2956E-03	2.1928E-03	1.1707E-02
40.000	5.4257E-03	9.3972E-05	1.0271E-03	4.8805E-04	1.8087E-03	2.2767E-03	2.0772E-03	5.8344E-04	4.3625E-03
45.000	2.3897E-03	5.3370E-05	3.8669E-04	3.6817E-04	9.0052E-04	1.0919E-03	1.0276E-03	4.1419E-05	1.3469E-03
50.000	1.8381E-03	9.8141E-05	2.4696E-04	3.5695E-04	6.7252E-04	7.7999E-04	7.3452E-04	1.4522E-04	8.5445E-04
60.000	1.9035E-03	2.0085E-04	1.4278E-04	4.0763E-04	4.3294E-04	3.9818E-04	3.1474E-04	1.1810E-04	2.2431E-04
70.000	3.2468E-04	5.4541E-04	6.4704E-05	6.5393E-04	1.5579E-03	5.4756E-04	9.5967E-04	4.5565E-04	1.0120E-03
76.000	1.1922E-02	1.9875E-03	2.8317E-04	1.5933E-03	1.8213E-03	4.3385E-03	6.4800E-03	3.4941E-03	6.0331E-03
76.800	1.3116E-02	2.2192E-03	3.6565E-03	1.6701E-03	2.2476E-03	5.1662E-03	7.6643E-03	4.1967E-03	7.1003E-03
76.900	1.3124E-02	2.2240E-03	3.7182E-04	1.6622E-03	2.2760E-03	5.2147E-03	7.7321E-03	4.2406E-03	7.1569E-03
77.000	1.3101E-02	2.2254E-03	3.7996E-04	1.6465E-03	2.3070E-03	5.2590E-03	7.7898E-03	4.2822E-03	7.2007E-03
80.000	6.7084E-03	1.2000E-03	3.0183E-04	6.7592E-04	1.6465E-03	3.4525E-03	5.0526E-03	2.5367E-03	4.5698E-03
90.000	1.7128E-03	1.5247E-04	1.6398E-04	4.0535E-05	8.0054E-04	1.5729E-03	2.3777E-03	1.9498E-03	2.1349E-03
100.000	7.3521E-04	1.7827E-04	1.0351E-04	9.8857E-05	5.8629E-04	1.2588E-03	2.1583E-03	1.7978E-03	2.0698E-03
110.000	2.9327E-03	6.1794E-04	4.3965E-04	2.7122E-04	4.6377E-04	1.3176E-03	2.7851E-03	2.8883E-03	3.1268E-03
110.800	3.4397E-03	7.3005E-04	5.2166E-04	3.3961E-04	4.5252E-04	1.1793E-03	2.5493E-03	2.6969E-03	2.9155E-03
113.900	3.4667E-03	7.4684E-04	5.2964E-04	3.4725E-04	4.3602E-04	1.1563E-03	2.5057E-03	2.6574E-03	2.8719E-03
114.000	3.5260E-03	7.5044E-04	5.3678E-04	3.5443E-04	4.9945E-04	1.1324E-03	2.4591E-03	2.6142E-03	2.8244E-03
120.000	2.3495E-03	5.2866E-04	3.8778E-04	3.9508E-04	3.6342E-04	5.0340E-04	1.0123E-03	1.1847E-03	1.0966E-03
130.000	2.3050E-03	5.2557E-04	3.7480E-04	4.7962E-04	1.6684E-04	3.8133E-04	1.3348E-03	2.0788E-03	1.8817E-03
140.000	7.5662E-03	7.2879E-04	4.7273E-04	7.1245E-04	4.8933E-04	2.5124E-04	1.6815E-03	3.6844E-03	3.2459E-03
150.000	7.5601E-03	1.5814E-03	8.9545E-04	1.5881E-03	2.3226E-03	2.0071E-03	2.1977E-03	9.6054E-03	8.2911E-03
154.000	1.1757E-02	2.5269E-03	1.2347E-03	2.3575E-03	4.7835E-03	4.1544E-03	3.3220E-03	1.5284E-02	1.3114E-02
154.700	1.1986E-02	2.5683E-03	1.2294E-03	2.3808E-03	5.0224E-03	4.4183E-03	3.2215E-03	1.5628E-02	1.3396E-02
154.900	1.1987E-02	2.5674E-03	1.2252E-03	2.3777E-03	5.0440E-03	4.4453E-03	3.1972E-03	1.5634E-02	1.3400E-02

HERTZ	1	2	3	4	5	6	7	8	9
155.000	1.1903E-04	4.5000E-03	1.2143E-03	2.3664E-03	5.0757E-03	4.4882E-03	3.1448E-03	1.5619E-02	1.3383E-02
160.000	7.0422E-04	1.4803E-03	5.8826E-04	1.2900E-03	3.5904E-03	3.4020E-03	1.1762E-03	9.3680E-03	7.9821E-03
180.000	1.9980E-03	3.7208E-04	3.9560E-05	1.9163E-04	1.4662E-03	1.5712E-03	5.5339E-04	2.4774E-03	2.1041E-03
200.000	1.4252E-03	2.3222E-04	1.3066E-04	3.9167E-05	1.0800E-03	1.1720E-03	7.1894E-04	1.3663E-03	1.1913E-03
220.000	1.4749E-03	2.1125E-04	2.4616E-04	1.6748E-04	8.4933E-04	7.6330E-04	6.1939E-04	7.0135E-04	6.6040E-04
240.000	2.5031E-03	3.3031E-04	5.6285E-04	4.6748E-04	7.2441E-04	2.4709E-03	2.3901E-03	2.7891E-03	2.4027E-03
260.000	2.5182E-03	3.3191E-04	5.7710E-04	5.0932E-04	9.4200E-04	3.1909E-03	3.1523E-03	3.5417E-03	3.0883E-03
280.000	2.5129E-03	3.3121E-04	5.7640E-04	5.0918E-04	8.4567E-04	3.2239E-03	3.1883E-03	3.5754E-03	3.1196E-03
300.000	2.5071E-03	3.3044E-04	5.7557E-04	5.0891E-04	8.5532E-04	3.2563E-03	3.2237E-03	3.6086E-03	3.1505E-03
320.000	2.4352E-03	3.2103E-04	5.6300E-04	5.0152E-04	9.0755E-04	3.4901E-03	3.4843E-03	3.8441E-03	3.3717E-03
340.000	1.0251E-03	1.3613E-04	2.5140E-04	2.3651E-04	9.9639E-04	3.4488E-03	3.7012E-03	3.6544E-03	3.3253E-03
360.000	4.6457E-04	4.0104E-05	1.4693E-04	1.2236E-04	8.0977E-04	2.5937E-03	3.0916E-03	2.6957E-03	2.5066E-03
380.000	9.3572E-04	7.1216E-05	3.2903E-04	2.9743E-04	5.2504E-04	2.1379E-03	3.1690E-03	2.2882E-03	2.3745E-03
400.000	1.5848E-03	1.1181E-04	6.0557E-04	5.7165E-04	2.463E-04	2.1410E-03	4.1478E-03	2.5831E-03	2.9341E-03
420.000	8.6191E-03	5.1442E-04	3.8169E-03	3.7270E-03	5.6982E-03	3.6098E-03	1.7896E-02	8.7974E-03	1.2221E-02
440.000	8.9272E-03	5.1673E-04	3.8504E-03	3.7607E-03	5.8167E-03	3.5423E-03	1.7972E-02	8.8071E-03	1.2274E-02
460.000	8.9339E-03	5.1688E-04	3.8542E-03	3.7646E-03	5.8339E-03	3.5292E-03	1.7977E-02	8.8044E-03	1.2277E-02
480.000	8.9458E-03	5.1681E-04	3.8619E-03	3.7726E-03	5.8794E-03	3.4890E-03	1.7976E-02	8.7891E-03	1.2276E-02
500.000	6.2381E-03	3.4347E-04	2.7464E-03	2.6919E-03	4.8824E-03	1.5337E-03	1.1986E-02	5.5929E-03	8.2057E-03
520.000	2.0583E-03	9.9906E-05	9.5513E-04	9.4194E-04	2.3929E-03	6.2630E-04	3.4469E-03	1.3870E-03	2.4221E-03
540.000	1.2600E-03	5.4380E-05	6.0750E-04	5.9977E-04	1.8762E-03	8.4948E-04	1.8566E-03	6.4699E-04	1.3665E-03
560.000	9.3919E-04	3.6542E-05	4.6594E-04	4.5974E-04	1.6619E-03	9.3815E-04	1.2237E-03	3.6175E-04	9.5644E-04
580.000	8.4228E-04	3.1259E-05	4.2274E-04	4.1571E-04	1.5958E-03	9.6323E-04	1.0334E-03	2.8717E-04	8.3575E-04
600.000	8.2698E-04	3.0432E-05	4.1590E-04	4.0984E-04	1.5854E-03	9.6716E-04	1.0034E-03	2.7467E-04	8.1642E-04
620.000	8.2644E-04	3.0403E-05	4.1566E-04	4.0965E-04	1.5850E-03	9.6721E-04	1.0024E-03	2.7422E-04	8.1617E-04
640.000	8.1544E-04	2.9812E-05	4.1076E-04	4.0476E-04	1.5775E-03	9.7000E-04	9.8093E-04	2.6528E-04	8.0263E-04
660.000	7.6723E-04	2.6154E-05	3.8009E-04	3.7408E-04	1.5302E-03	9.8672E-04	8.4717E-04	2.1558E-04	7.1901E-04
680.000	7.3935E-04	2.5734E-05	3.7653E-04	3.7052E-04	1.5247E-03	9.8838E-04	8.3178E-04	2.0431E-04	7.0941E-04
700.000	7.2768E-04	2.5125E-05	3.7136E-04	3.6533E-04	1.5167E-03	9.9131E-04	8.0934E-04	1.9529E-04	6.9549E-04
720.000	5.5634E-04	1.6235E-05	2.9310E-04	2.8640E-04	1.3945E-03	1.0286E-03	4.7414E-04	6.6816E-05	4.9194E-04
740.000	4.9203E-04	1.3078E-05	2.6348E-04	2.5622E-04	1.3475E-03	1.0403E-03	3.4968E-04	2.3699E-05	4.1917E-04
760.000	4.8104E-04	1.2516E-05	2.5804E-04	2.5068E-04	1.3388E-03	1.0423E-03	3.2716E-04	1.6575E-05	4.0616E-04
780.000	4.7834E-04	1.2384E-05	2.5671E-04	2.4937E-04	1.3368E-03	1.0428E-03	3.2172E-04	1.5001E-05	4.0309E-04
800.000	4.7745E-04	1.2341E-05	2.5636E-04	2.4894E-04	1.3361E-03	1.0429E-03	3.2001E-04	1.4502E-05	4.0209E-04
820.000	4.5075E-04	1.1062E-05	2.4377E-04	2.3603E-04	1.3159E-03	1.0472E-03	2.6780E-04	9.0514E-06	3.7238E-04

HERTZ	1	2	3	4	5	6	7	8	9
3.000	27.10	26.64	25.97	25.04	23.57	22.09	19.61	309.57	214.63
3.100	51.61	51.13	50.43	49.42	47.73	45.90	42.52	275.15	238.13
3.200	115.59	115.07	114.33	113.20	111.17	108.77	103.65	319.03	301.47
4.000	177.05	176.41	175.18	172.41	157.51	36.71	9.83	2.44	0.42
6.000	182.59	181.73	178.25	15.01	6.46	5.54	5.07	4.53	4.21
8.000	188.71	187.76	169.16	11.54	10.22	9.93	9.77	9.64	9.62
9.000	205.34	204.37	53.44	27.17	26.28	26.06	25.95	25.92	25.98
9.400	228.44	253.52	89.11	76.02	75.22	75.03	74.94	74.94	75.03
9.500	276.51	277.53	111.75	100.01	99.22	99.02	98.93	98.94	99.03
10.000	336.99	336.00	165.43	158.20	157.48	157.30	157.23	157.27	157.41
12.000	355.49	354.45	178.73	175.86	175.24	175.08	175.04	175.21	175.47
14.000	358.27	357.14	180.05	178.00	177.31	177.10	177.07	177.36	177.75
16.000	359.71	358.46	180.75	178.83	177.90	177.57	177.50	177.86	178.34
18.000	1.00	359.62	181.50	179.39	177.92	177.24	176.93	177.07	177.45
20.000	3.33	1.93	183.11	180.40	177.32	175.00	172.84	169.35	167.20
22.000	23.04	23.07	198.80	192.06	170.49	135.71	103.41	81.58	76.97
22.400	48.00	45.66	215.65	204.67	165.56	128.98	111.40	101.87	100.01
22.500	50.44	52.80	220.48	207.86	165.26	132.48	117.95	110.22	108.77
23.000	62.79	70.97	222.03	204.05	170.29	155.61	149.76	147.33	147.16
26.000	6.05	3.84	184.92	180.08	171.10	168.30	171.98	177.90	179.65
30.000	7.07	3.06	186.32	177.81	47.74	35.48	108.65	181.45	184.43
33.000	11.86	6.11	190.95	170.21	22.01	20.54	25.68	185.31	189.56
36.000	33.26	24.22	212.11	79.33	38.50	37.84	39.42	204.56	210.90
37.000	65.59	54.64	244.36	90.81	69.96	69.39	70.59	235.77	243.17
37.400	88.83	76.95	267.56	110.02	92.90	92.35	93.44	258.54	266.39
37.500	92.13	82.93	273.85	115.39	99.13	98.59	99.65	264.62	272.67
38.000	122.43	108.68	301.10	135.29	126.09	125.57	126.51	291.13	299.92
40.000	161.03	134.61	339.50	170.89	163.56	163.11	163.71	324.86	338.29
45.000	174.47	14.26	352.31	178.78	174.95	174.46	174.64	224.83	350.79
50.000	178.04	5.94	354.99	180.16	176.71	175.94	175.74	184.50	352.55
60.000	182.65	6.35	355.78	182.15	175.32	170.64	165.13	168.54	335.83
70.000	192.65	15.11	336.30	189.15	102.89	38.11	30.51	32.91	206.74
76.000	242.12	64.16	268.98	235.77	79.32	73.36	72.22	74.32	250.97
76.800	264.52	86.52	287.22	257.15	99.89	95.02	94.14	96.22	273.04
76.900	267.58	89.57	289.89	260.58	102.76	98.01	97.15	99.23	276.06
77.000	270.59	92.58	292.40	263.50	105.55	100.93	100.10	102.17	279.03
80.000	326.32	148.17	340.27	316.38	157.12	154.76	154.55	156.62	333.93
90.000	344.94	170.53	355.28	264.42	175.12	175.44	176.32	178.57	356.73
100.000	340.36	163.81	345.74	184.38	175.59	179.16	181.29	184.04	2.76
110.000	276.43	99.28	279.86	125.37	159.18	209.96	218.98	223.83	44.16
110.800	290.01	112.37	292.99	133.32	158.67	218.52	229.76	235.20	56.01
110.900	291.83	114.13	294.76	134.55	158.78	219.52	231.08	236.61	57.49
111.000	293.64	115.89	296.53	135.81	158.94	220.45	232.36	237.97	58.93
120.000	352.67	175.22	355.25	180.59	169.20	181.34	190.37	195.78	16.33
130.000	3.15	183.12	2.01	185.38	132.35	167.86	183.94	189.55	9.04
140.000	11.00	190.56	7.90	191.00	36.54	82.73	186.29	194.15	13.67
150.000	36.50	215.70	30.66	214.47	48.36	95.12	205.01	218.00	37.65
154.000	76.85	255.91	69.45	253.95	86.39	90.75	240.64	257.83	77.54
154.700	87.28	266.32	79.58	264.23	96.48	100.57	249.99	268.18	87.90
154.800	88.79	267.83	81.04	265.72	97.94	101.99	251.33	269.68	89.40

HEATZ	1	2	3	4	5	6	7	8	9
155.000	91.81	270.84	83.96	268.69	100.87	104.85	254.02	272.64	92.40
160.000	142.60	321.67	132.08	318.46	149.88	152.53	291.81	323.09	142.39
180.000	172.21	350.59	172.44	338.79	174.45	175.20	180.26	350.26	170.40
200.000	178.10	356.19	10.86	345.19	176.56	175.85	180.25	352.38	173.02
220.000	183.83	2.05	10.23	196.74	175.44	165.15	169.11	338.74	161.91
240.000	221.16	41.40	44.23	227.83	147.46	91.59	95.50	266.53	89.53
260.000	235.24	56.15	57.82	241.58	145.32	103.68	107.46	279.76	102.32
282.130	236.02	56.97	58.57	242.34	145.37	104.40	108.18	280.54	103.08
242.200	236.80	57.79	59.32	243.11	145.44	105.13	108.91	281.32	103.84
243.000	243.12	64.45	65.40	249.31	146.48	111.24	114.98	287.77	110.17
250.000	275.05	102.82	93.48	280.28	164.98	154.41	158.06	332.82	154.62
250.000	222.78	63.14	39.16	227.01	171.96	171.51	175.35	351.22	172.84
280.000	191.43	10.93	13.09	194.45	167.97	178.49	183.23	359.98	181.71
300.000	192.17	10.23	13.77	194.41	117.49	181.69	188.23	5.37	187.36
320.000	201.74	19.51	23.03	203.44	44.04	188.67	198.90	16.25	198.53
340.000	262.96	80.67	94.07	264.29	93.96	240.17	260.48	77.92	260.51
360.600	267.01	84.72	88.12	268.33	97.85	243.66	264.54	81.98	264.58
340.700	267.99	85.40	88.80	269.01	98.50	244.25	265.22	82.67	265.26
341.000	269.73	87.44	90.84	271.05	100.47	246.01	267.26	84.71	267.31
350.000	316.67	134.38	157.69	317.86	145.45	279.20	314.27	131.73	314.48
380.000	347.59	165.36	168.38	348.44	173.05	193.47	345.23	162.64	345.90
410.000	332.87	170.74	173.51	353.51	176.81	183.16	350.46	167.60	351.50
440.000	335.04	173.02	175.55	355.52	178.09	181.35	352.53	169.35	353.88
455.000	335.70	173.74	176.17	356.12	178.44	180.97	353.14	168.69	354.64
457.800	335.81	173.86	176.27	356.22	178.49	180.92	353.24	169.73	354.76
457.900	335.81	173.86	176.27	356.22	178.49	180.92	353.25	169.73	354.77
460.000	335.89	173.95	176.34	356.29	178.53	180.88	353.32	169.75	354.86
475.000	336.37	174.48	176.78	356.72	178.76	180.67	353.74	169.81	355.41
477.000	336.43	174.55	176.83	356.78	178.79	180.64	353.79	169.80	355.47
480.000	336.51	174.64	176.91	356.85	178.82	180.61	353.86	169.78	355.57
550.000	337.76	176.15	178.04	357.96	179.35	180.24	354.88	166.73	357.07
600.000	338.25	176.80	178.47	358.39	179.52	180.14	355.20	155.05	357.69
612.000	338.35	176.92	178.55	358.47	179.55	180.13	355.25	146.55	357.80
615.000	338.37	176.95	178.57	358.49	179.56	180.13	355.26	143.43	357.83
616.000	338.37	176.96	178.58	358.50	179.56	180.12	355.27	142.28	357.84
650.000	338.59	177.27	178.77	358.69	179.63	180.09	355.36	51.06	358.11